

# Mobile Cylinders

Product Information, Quick Reference Data & Application Guide

Catalog HY18-0001/US







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Cylinders were among the first hydraulic products of Commercial Hydraulics beginning in 1928. In 2000 Commercial Intertech Corp. merged with the Parker Hannifin Corp. Today, the Parker Mobile Cylinder Division is recognized as one of the largest independent manufacturers of hydraulic cylinders for mobile equipment and a leading supplier to refuse, mining, truck, material handling and positioning markets. Parker's Mobile Cylinder Division products include double- and single-acting telescopic cylinders, rod cylinders and "smart cylinders" with internal electronic controls. With manufacturing facilities in Youngstown, OH, Minneapolis, MN, Benton, AR and Geringswalde, Germany, the Parker Mobile Cylinder Division has the resources and expertise to support multinational markets and customers.

Additionally, the company enjoys a reputation for quality custom built cylinders that demonstrate engineering know how and adept manufacturing. Cylinders represent a significant portion of the company's Hydraulic Systems sales - their contribution has resulted in significant capital expenditures geared to establish Parker's Mobile Cylinder Division as the world's leading manufacturer of hydraulic cylinders.

# About our Quick Reference and Application Guide . . .

It's our goal to provide our customers with the highest quality cylinders. If the need arises for a cylinder not listed in this guide, please furnish us the following information and we'll do our best to provide a new or replacement.

- \* OEM identification number
- \* Single- or double-acting type
- \* Outside diameter of the largest moving stage or rod and bore diameters
- \* Number of moving stages
- \* Application

- \* Retract or closed length
- \* Stroke length
- \* Extend or open length
- \* Pineye sizes
- \* Plunger mount type
- \* Base mount type

Please refer to the cylinder specification data forms for more information.

Parker's Mobile Cylinder Division has complete manufacturing facilities to fill all your cylinder needs from 1 piece to 1000 pieces. If you're looking for a prototype, a special cylinder, or a production run, Parker's Mobile Cylinder Division is the place to call.

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Mobile Cylinder Div.

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# MOBILE CYLINDER DIVISION PRODUCTS & CAPABILITIES

- TELESCOPIC CYLINDERS Single Acting
   Double Acting
- SINGLE STAGE "Rod Type" CYLINDERS
  Single Acting
  Double Acting
- BUILD TO CUSTOMER PRINTS OR PER APPLICATION SPECIFICATIONS
- BORE SIZES UP TO 20" DIAMETER
- STROKE LENGTHS UP TO 500"
- OPERATING PRESSURES UP TO 10,000 PSI
- VARIOUS OPERATING FLUIDS
- BATCH SIZES 1PC TO 100's
- VARIOUS MATERIALS & COATINGS Stainless Steel Electroless Nickel Nitriding Chrome Double Chrome
- TYPICAL OPTIONS

   Load Holding Valves
   Electro-Hydraulic Transducers
   End of Stroke Hydraulic Cushions
   Protective Rod Boots
   Proximity Switches
   Flow Controls
   Flow Fuses

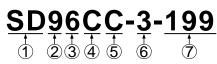
The code and model numbers of a Commercial Hydraulics Cylinder are references to its size and type. Using these numbers when ordering or inquiring greatly facilitates accurate understanding.

The following are examples of Commercial Hydraulics cylinder code and model numbers.

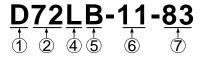
Single-acting Telescopic

**S63MB-9-120** 

**Double-acting Telescopic** 



**Double-acting Piston Rod** 



1. S = Single-acting Telescopic or Displacement Cylinder

(Commercial has also used SA, SF, and H as a prefix)

**SD = Double-acting Telescopic Cylinder** 

D = Double-acting Piston Rod Cylinder

2. = Nominal O.D. of the largest moving stage on Single-acting and Double-acting Telescopic cylinders or the

Nominal Bore of Double-acting Piston Rod Cylinders

- 3. = Number of moving stages or sleeves in a Telescopic Cylinder
- **4. = Mounting option on the body or base end of cylinder** (See mounting Option and Code Chart for mount descriptions)
- 5. = Mounting option on the rod or plunger end of cylinder (See mounting Option and Code Chart for mount descriptions)
- 6. = Modification or design variation of the cylinder
- 7. = Length of cylinder stroke in inches

#### OUR DESIGN ADVANTAGES INCLUDE:

- \* Longer sleeve overlap for improved stability and higher column loading.
- \* Nylon tipped set screws that conform to the shape of the packing nut threads. It is nearly impossible for the packing nut to back off accidentally.
- \* Snap-on, glass-filled bearings that absorb contaminants without damaging cylinder walls.
- \* Threaded steel stop rings for easier servicing and more reliable stopping action.
- \* External packing nuts give added support to the tube exterior while making service procedures easier.
- \* Wave springs and chevron packing for self-compensating seals.
- \* Hytrel rod wipers that resist higher temperatures without extrusion.
- Positive manual air bleeder prevents cavitation and "mushy" cvlinder action.
- \* Cast steel mountings offer dependable strength. Pin-eye and rod-end are welded into a single unit.



#### "S" SERIES SINGLE-ACTING, SINGLE & MULTIPLE STAGE CYLINDERS

Sleeve or Plunger O.D. (in inches)	Effective Area in square inches	Load Capactity lbs @ 2000 p.s.i.	Displacement per inch of stroke in gallons *
1.75"	2.41"	4,811	0.010
2.75"	5.94"	11,880	0.026
3.75"	11.04"	22,089	0.048
4.75"	17.72"	35,441	0.077
5.75"	25.97"	51,935	0.112
6.75"	35.78"	71,570	0.155
7.90"	49.02"	98,034	0.212
9.38"	69.03"	138,059	0.299
10.75"	90.76"	181,526	0.393
12.50"	122.72"	245,438	0.531
14.00"	153.94"	307,877	0.666

#### "SD" SERIES DOUBLE-ACTING, MULTIPLE STAGE CYLINDER

Sleeve or Plunger O.D. (in inches)	Bore of Main or Sleeve (in inches)	Effective area (sq. inches) to extend	Effective area (sq. inches) to retract	Load capacity lbs @ 2000 p.s.i. extending	Load capacity lbs @ 2000 p.s.i. retracting	Displacement per inch of stroke (in gallons)* to extend	Displacement per inch of stroke (in gallons)* to retract
1.75"	2.25"	3.98"	1.57"	7,952	3,142	0.017	0.007
2.75"	3.25"	8.29"	2.35"	16,592	4,712	0.036	0.010
3.75"	4.25"	14.18"	3.14"	28,372	6,283	0.061	0.014
4.75"	5.25"	21.64"	3.92"	43,296	7,854	0.094	0.017
5.75"	6.25"	30.68"	4.71"	61,360	9,426	0.133	0.020
6.75"	7.25"	41.28"	5.49"	82,564	10,994	0.179	0.024
7.90"	8.44"	55.68"	6.97"	111,360	13,946	0.242	0.030
9.38"	9.88"	76.59"	7.56"	153,180	15,120	0.332	0.033
10.75"	11.50"	103.87"	13.11"	207,738	26,213	0.450	0.057
12.50"	13.00"	132.73"	10.01"	265,465	20,028	0.575	0.043
14.00"	14.50"	165.13"	11.19"	330,261	22,384	0.715	0.048

Note: The Effective area to RETRACT a Standard "SD" series double acting multiple stage cylinder is the effective area of the PLUNGER (plunger bore area minus the plunger O.D. area).

Example: Retract force for a SD94CC-8-190 (which has 5.75" O.D. plunger and fits in 6.25" bore) would be 9,426 lbs @ 2,000 psi, based on a 4.71 sq. in. effective area.

To calculate effective area in square inches: Multiply diameter times diameter times .78

Example: 5 dia. x 5 dia. = 25 x .78 = 19.63 Square inches of area

To calculate load capacity / cylinder force: Multiply effective area times operating pressure (psi)

Example: 19.63 Square inches x 1750 P.S.I = 34,361 lbs of force

#### To calculate the required gallons of fluid to extend a cylinder:

Add each "Displacement per inch of stroke" (from chart) for the required sleeve sizes.

Divide this total by the number of moving sleeves, then multiply that total by the desired cylinder stroke.

Note: The "Gallons required to extend" does not include the necessary fluid to fill an empty cylinder.

Example: Required fluid to extend a S83DC-40-134 single-acting telescopic cylinder with following stage sizes:

5.75" O.D.= .112

6.75" O.D.= .155

7.90" O.D.= <u>.210</u>

.477

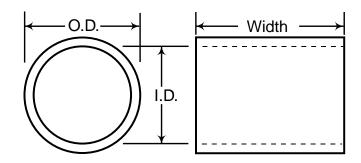
 $.447 \div 3 = .159$  gallons per inch of stroke

.159 gallons per inch x 134" of stroke = 21.31 gallons to extend cylinder



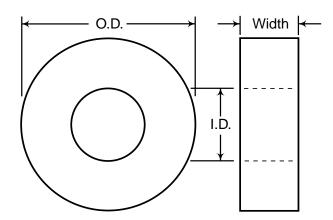
Code Letter	Mount Description	Mount Sketch	Mount Location
A	Plain No Mount		Body or Rod
В	Pin-Eye Drilled Thru Rod		Rod
С	Pin-Eye Drilled Thru Lug		Body or Rod
D	Cross Tube		Body or Rod
E	Threaded		Body or Rod
F	Drilled and Tapped		Body or Rod
G	Flange Mount at Base		Body
Н	Flange Mount Mid-Body		Body
J	Foot / Pad Mount		Body
K	Centerline Mount		Body
L	Double Lug Clevis Mount		Body or Rod
M	Trunnion Mount		Body
N	Rod End Drilled and Tapped		Rod
0	Ball Mount		Body or Rod
P	Socket Mount		Body or Rod

# Accessories to Modify Cylinder Mounting Pin Hole Diameters and Mount Widths



# **Pin - Eye Mount Bushings**

Engineering #	Part Number	<b>Nominal Dimensions</b>
CC1519-1-1.5	375-1519-556	1.50 I.D. X 2.00 O.D. X 1.50 Wide
CC1519-1-2.0	375-1519-557	1.50 I.D. X 2.00 O.D. X 2.00 Wide
CC1519-2-1.5	375-1519-558	1.75 I.D. X 2.00 O.D. X 1.50 Wide
CC1519-2-2.0	375-1519-559	1.75 I.D. X 2.00 O.D. X 2.00 Wide
CC1519-3-2.0	375-1519-560	1.31 I.D. X 2.00 O.D. X 2.00 Wide
CC1519-4-2.0	375-1519-561	1.68 I.D. X 2.00 O.D. X 2.00 Wide



**Pin - Eye Mount Spacers** 

	_	-
Engineering #	Part Number	<b>Nominal Dimensions</b>
CC1519-5-0.25	375-1519-562	1.81 I.D. X 3.00 O.D. X 0.25 Wide
CC1519-5-0.50	375-1519-563	1.81 I.D. X 3.00 O.D. X 0.50 Wide
CC1519-5-1.0	375-1519-564	1.81 I.D. X 3.00 O.D. X 1.00 Wide
CC1519-5-0.75	375-1519-565	1.81 I.D. X 3.00 O.D. X 0.75 Wide
CC1519-6-0.25	375-1519-566	2.12 I.D. X 3.00 O.D. X 0.25 Wide
CC1519-6-0.50	375-1519-567	2.12 I.D. X 3.00 O.D. X 0.50 Wide
CC1519-6-1.0	375-1519-568	2.12 I.D. X 3.00 O.D. X 1.00 Wide
CC1519-6-0.75	375-1519-569	2.12 I.D. X 3.00 O.D. X 0.75 Wide

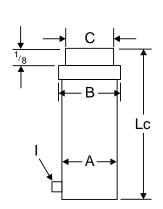
## Closed Length Calculations for Single-Acting Single & Multiple Stage Cylinders

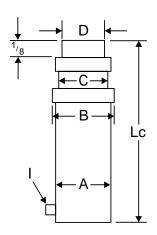
\* Closed length (Lc) for S Models is computed by one of the three equations below. Model number and stroke required determines which equation to use. Example: To find Lc for S41 cylinder with 68" stroke. Under S41 column, use equation III, because the stroke is over 50".

Lc = Stroke + 
$$X_1$$
 +  $X_2$  = 68" + 7.50" +  $\frac{(68 - 50)}{10}$  = 68" + 7.50" + (1.8)

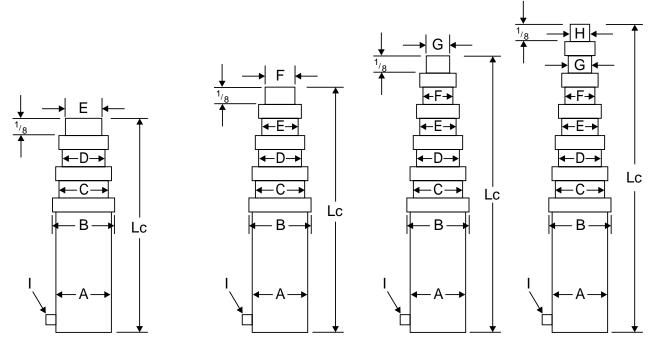
Use next largest whole number. = 68" + 7.50" + 2" = 77.50".

The closed length (Lc) is 77.50". Add Lc 77.50" to the stroke 68" for extended length of 145.50"





			SINGLE STAGE									2 STAGE			
Cylinder Dimensions (inches)		S31	S41	S51	S61	S71	S81	S91		S42	S52	S62	S72	S82	S92
Main Cylinder O.D.	Α	3 <sup>3</sup> / <sub>4</sub>	43/4	5 <sup>3</sup> / <sub>4</sub>	6 <sup>3</sup> / <sub>4</sub>	8	9 <sup>1</sup> / <sub>8</sub>	10 <sup>13</sup> / <sub>16</sub>	Α	43/4	5 <sup>3</sup> / <sub>4</sub>	6 <sup>3</sup> / <sub>4</sub>	8	9 <sup>1</sup> / <sub>8</sub>	10 <sup>13</sup> / <sub>16</sub>
Largest Packing Nut O.D.	В	4 <sup>3</sup> / <sub>8</sub>	5 <sup>3</sup> / <sub>8</sub>	6 <sup>3</sup> / <sub>8</sub>	73/8	8 <sup>5</sup> / <sub>8</sub>	97/8	11 <sup>3</sup> / <sub>4</sub>	В	5 <sup>3</sup> / <sub>8</sub>	6 <sup>3</sup> / <sub>8</sub>	7 <sup>3</sup> / <sub>8</sub>	8 <sup>5</sup> / <sub>8</sub>	9 <sup>7</sup> / <sub>8</sub>	11 <sup>3</sup> / <sub>4</sub>
1st Sleeve O.D.	С	23/4	33/4	43/4	53/4	63/4	7 <sup>7</sup> / <sub>8</sub>	9 <sup>3</sup> / <sub>8</sub>	С	33/4	43/4	5 <sup>3</sup> / <sub>4</sub>	63/4	7 <sup>7</sup> / <sub>8</sub>	93/8
2nd Sleeve O.D.	D								D	23/4	3 <sup>3</sup> / <sub>4</sub>	43/4	53/4	63/4	7 <sup>7</sup> / <sub>8</sub>
3rd Sleeve O.D.	E					Е									
4th Sleeve O.D.	F								F						
5th Sleeve O.D.	G								G						
6th Sleeve O.D.	Н								Н						
NPT Port	ı	3/4	3/4	3/4	1	1	1 <sup>1</sup> / <sub>4</sub>	1 <sup>1</sup> / <sub>4</sub>	I	3/4	3/4	1	1	1 <sup>1</sup> / <sub>4</sub>	1 <sup>1</sup> / <sub>4</sub>
Max. Stroke at 2000 PSI		71	84	88	95	118	128	190		126	137	138	164	186	265
*To Find Closed Length - Lc	х	5.75	5.75	5.75	6.00	6.00	6.50	6.62	Х	6.69	6.69	6.94	6.94	7.44	7.56
Equation I	L <sub>C</sub>				Stroke + 2 to 35" str			O.L. = 1 <sup>1</sup> / <sub>4</sub> "	L <sub>C</sub>	Stroke 2 + X O.L. up to 35" stroke				O.L. = 1 <sup>1</sup> / <sub>4</sub>	
	X <sub>1</sub>	7.50	7.50	7.50	7.75	7.75	8.25	8.38	X <sub>1</sub>	8.44	8.44	8.69	8.69	9.19	9.31
Equation II	L <sub>C</sub>		Stroke + X <sub>1</sub> O.L. = 3" 36" to 50" stroke					Lc				oke 2 + X 0" stroke	1	O.L. = 3	
Equation III	X <sub>2</sub>		(	-	Stroke - 5 10 argest who	0 ole numbe	er)		X <sub>2</sub>		(To ne	Stroke 2 ext larges	20	umber)	
Equalion III	L <sub>C</sub>				oke + X <sub>1</sub> · er 50" str		О.	L. = 3" + X <sub>2</sub>	L <sub>C</sub>			-	oke 2 + X <sub>1</sub> )" stroke	+ X 2 O	.L. = 3" + X <sub>2</sub>



		3 ST	AGE					4 STAGE				5 S1	AGE			6 ST	AGE
	S53	S63	S73	S83	S93		S64	S74	S84	S94		S75	S85	S95		S86	S96
А	53/4	63/4	8	9 <sup>1</sup> / <sub>8</sub>	10 <sup>13</sup> / <sub>16</sub>	Α	63/4	8	9 <sup>1</sup> / <sub>8</sub>	10 <sup>13</sup> / <sub>16</sub>	Α	8	9 <sup>1</sup> / <sub>8</sub>	10 <sup>13</sup> / <sub>16</sub>	Α	91/8	10 <sup>13</sup> / <sub>16</sub>
В	6 <sup>3</sup> / <sub>8</sub>	7 <sup>3</sup> / <sub>8</sub>	8 <sup>5</sup> / <sub>8</sub>	9 <sup>7</sup> / <sub>8</sub>	11 <sup>3</sup> / <sub>4</sub>	В	7 <sup>3</sup> / <sub>8</sub>	8 <sup>5</sup> / <sub>8</sub>	9 <sup>7</sup> / <sub>8</sub>	11 <sup>3</sup> / <sub>4</sub>	В	8 <sup>5</sup> / <sub>8</sub>	9 <sup>7</sup> / <sub>8</sub>	11 <sup>3</sup> / <sub>4</sub>	В	9 <sup>7</sup> / <sub>8</sub>	11 <sup>3</sup> / <sub>4</sub>
С	43/4	5 <sup>3</sup> / <sub>4</sub>	6 <sup>3</sup> / <sub>4</sub>	7 <sup>7</sup> / <sub>8</sub>	9 <sup>3</sup> / <sub>8</sub>	С	5 <sup>3</sup> / <sub>4</sub>	6 <sup>3</sup> / <sub>4</sub>	7 <sup>7</sup> / <sub>8</sub>	9 <sup>3</sup> / <sub>8</sub>	С	6 <sup>3</sup> / <sub>4</sub>	7 <sup>7</sup> / <sub>8</sub>	9 <sup>3</sup> / <sub>8</sub>	С	7 <sup>7</sup> / <sub>8</sub>	9 <sup>3</sup> / <sub>8</sub>
D	33/4	4 <sup>3</sup> / <sub>4</sub>	53/4	63/4	7 <sup>7</sup> / <sub>8</sub>	D	43/4	5 <sup>3</sup> / <sub>4</sub>	63/4	7 <sup>7</sup> / <sub>8</sub>	D	5 <sup>3</sup> / <sub>4</sub>	63/4	7 <sup>7</sup> / <sub>8</sub>	D	6 <sup>3</sup> / <sub>4</sub>	7 <sup>7</sup> / <sub>8</sub>
Е	2 <sup>3</sup> / <sub>4</sub>	3 <sup>3</sup> / <sub>4</sub>	43/4	5 <sup>3</sup> / <sub>4</sub>	6 <sup>3</sup> / <sub>4</sub>	Е	3 <sup>3</sup> / <sub>4</sub>	43/4	5 <sup>3</sup> / <sub>4</sub>	6 <sup>3</sup> / <sub>4</sub>	Е	43/4	5 <sup>3</sup> / <sub>4</sub>	6 <sup>3</sup> / <sub>4</sub>	Е	5 <sup>3</sup> / <sub>4</sub>	6 <sup>3</sup> / <sub>4</sub>
F						F	2 <sup>3</sup> / <sub>4</sub>	33/4	43/4	5 <sup>3</sup> / <sub>4</sub>	F	33/4	$4^{3}/_{4}$	5 <sup>3</sup> / <sub>4</sub>	F	4 <sup>3</sup> / <sub>4</sub>	5 <sup>3</sup> / <sub>4</sub>
G						G					G	23/4	3 <sup>3</sup> / <sub>4</sub>	43/4	G	33/4	43/4
Н						Н					Н				Н	23/4	3 <sup>3</sup> / <sub>4</sub>
1	3/4	1	1	1 <sup>1</sup> / <sub>4</sub>	1 <sup>1</sup> / <sub>4</sub>	1	1	1	1 <sup>1</sup> / <sub>4</sub>	1 <sup>1</sup> / <sub>4</sub>	I	1	1 <sup>1</sup> / <sub>4</sub>	1 <sup>1</sup> / <sub>4</sub>	I	1 <sup>1</sup> / <sub>4</sub>	1 <sup>1</sup> / <sub>4</sub>
	181	186	204	224	312		238	262	265	352		335	336	410		T.B.A.	T.B.A.
Х	7.62	7.88	7.88	8.38	8.50	Х	8.81	8.81	9.31	9.44	Х	9.75	10.25	10.38	Х	11.19	11.31
L <sub>c</sub>			Stroke 3 + to 50" str		O.L. = 1 <sup>1</sup> / <sub>4</sub> "	L <sub>C</sub>			oke + X )" stroke	O.L. = 1 <sup>1</sup> / <sub>4</sub> "	L <sub>C</sub>	"	+ X to 85" str	O.L. = 1 <sup>1</sup> / <sub>4</sub> "	L <sub>C</sub>	Stroke 6 + 2 up to 10	O.L. = 1 <sup>1</sup> / <sub>4</sub> "
X <sub>1</sub>	9.38	9.62	9.62	10.12	10.25	X <sub>1</sub>	10.56	10.56	11.06	11.19	X <sub>1</sub>	11.50	12.00	12.12	X <sub>1</sub>	12.94	13.06
L <sub>C</sub>		51"	Stroke 3 to 75" str		O.L. = 3"	L <sub>C</sub>		Stro 4 71" to	ke + X <sub>1</sub>	O.L. = 3"	L <sub>C</sub>		+ X <sub>1</sub> to 125" s	O.L. = 3"	L <sub>C</sub>	Stroke 6 + X 101" to 1	O.L. = 3" 50" stroke
X <sub>2</sub>	(	To next la	Stroke - 7 30 argest who	ole numb		X <sub>2</sub>	Stroke - 100 40 (To next largest whole number)			X <sub>2</sub>			le number)	X <sub>2</sub>	(To next largest	e - 150 60 whole number)	
L <sub>C</sub>			oke 3 er 75" str	+ X <sub>2</sub>	L. = 3" + X <sub>2</sub>	L <sub>C</sub>	<u>S</u>	troke 4 + 2 over 100		L. = 3" + X <sub>2</sub>	L <sub>C</sub>		o + X <sub>1</sub> + X <sub>2</sub> er 125" st		L <sub>C</sub>	Stroke 6 + X <sub>1</sub> over 15	O.L. = 3" + X <sub>2</sub> + X <sub>2</sub> O" stroke

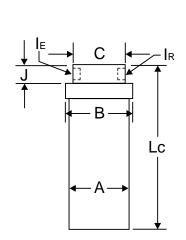
## Closed Length Calculations for Double-Acting Single & Multiple Stage Cylinders

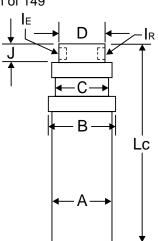
\* Closed length (Lc) for SD Models is computed by one of the three equations below. Model number and stroke required determines which equation to use. Example: To find Lc for SD41 cylinder with 68" stroke. Under SD41 column, use equation III, because the stroke is over 66".

Lc = Stroke + 
$$X_1$$
 +  $X_2$  = 68" + 12" +  $\frac{68 - 50}{4.5}$  = 68" + 12" + (.666).

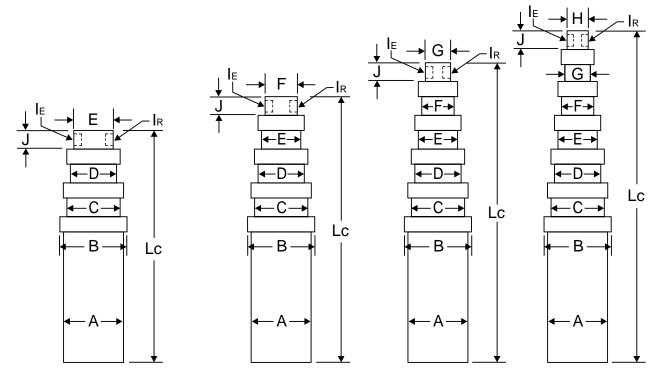
Use next largest whole number. = 68" + 12" + 1" = 81".

The closed length (Lc) is 81". Add Lc 81" to the stroke 68" for extended length of 149"

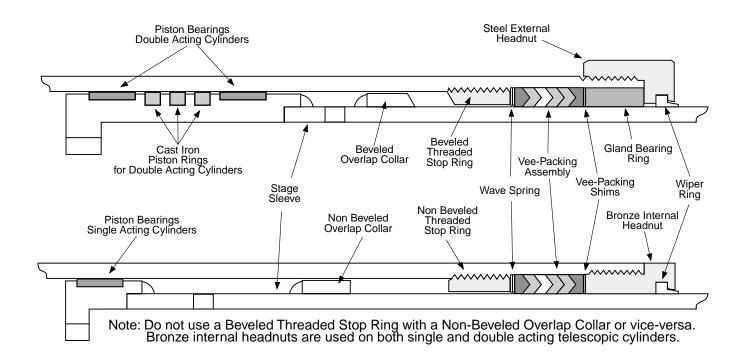




		SINGLE STAGE										2 STAGE			
Cylinder Dimensions (inches)		SD31	SD41	SD51	SD61	SD71	SD81	SD91		SD42	SD52	SD62	SD72	SD82	SD92
Main Cylinder O.D.	А	33/4	4 <sup>3</sup> / <sub>4</sub>	5 <sup>3</sup> / <sub>4</sub>	6 <sup>3</sup> / <sub>4</sub>	8	9 <sup>1</sup> / <sub>8</sub>	10 <sup>13</sup> / <sub>16</sub>	Α	43/4	5 <sup>3</sup> / <sub>4</sub>	63/4	8	9 <sup>1</sup> / <sub>8</sub>	10 <sup>13</sup> / <sub>16</sub>
Largest Packing Nut O.D.	В	43/8	5 <sup>3</sup> / <sub>8</sub>	6 <sup>3</sup> / <sub>8</sub>	73/8	8 <sup>5</sup> / <sub>8</sub>	97/8	11 <sup>3</sup> / <sub>4</sub>	В	5 <sup>3</sup> / <sub>8</sub>	6 <sup>3</sup> / <sub>8</sub>	73/8	8 <sup>5</sup> / <sub>8</sub>	97/8	11 <sup>3</sup> / <sub>4</sub>
1st Sleeve O.D.	С	2 <sup>3</sup> / <sub>4</sub>	33/4	43/4	5 <sup>3</sup> / <sub>4</sub>	6 <sup>3</sup> / <sub>4</sub>	7 <sup>7</sup> / <sub>8</sub>	9 <sup>3</sup> / <sub>8</sub>	С	33/4	43/4	53/4	63/4	7 <sup>7</sup> / <sub>8</sub>	9 <sup>3</sup> / <sub>8</sub>
2nd Sleeve O.D.	D								D	2 <sup>3</sup> / <sub>4</sub>	33/4	43/4	53/4	6 <sup>3</sup> / <sub>4</sub>	7 <sup>7</sup> / <sub>8</sub>
3rd Sleeve O.D.	Е								Е						
4th Sleeve O.D.	F								F						
5th Sleeve O.D.	G								G						
6th Sleeve O.D.	Н								Н						
NPT Port - Extend	I <sub>E</sub>	3/4	3/4	3/4	1	1	11/4	1 <sup>1</sup> / <sub>4</sub>	Ι <sub>Ε</sub>	3/4	1	1	1 <sup>1</sup> / <sub>4</sub>	11/4	1 <sup>1</sup> / <sub>4</sub>
NPT Port - Retract	I <sub>R</sub>	1/2	1/2	1/2	3/4	3/4	1	1	I <sub>R</sub>	1/2	3/4	3/4	1	1	1
Plunger Extension	J	1 <sup>5</sup> / <sub>8</sub>	1 <sup>5</sup> / <sub>8</sub>	1 <sup>5</sup> / <sub>8</sub>	21/8	2 <sup>1</sup> / <sub>8</sub>	2 <sup>5</sup> / <sub>8</sub>	2 <sup>5</sup> / <sub>8</sub>	J	1 <sup>5</sup> / <sub>8</sub>	2 <sup>1</sup> / <sub>8</sub>	21/8	2 <sup>5</sup> / <sub>8</sub>	2 <sup>5</sup> / <sub>8</sub>	2 <sup>5</sup> / <sub>8</sub>
Max. Recommended Ext. Lgth. at 2000 PSI		131	155	170	186	235	272	386		171	184	199	241	275	390
Max. Stroke at 2000 PSI		59	70	77	84	106	122	174		100	108	117	142	162	234
*To Find Closed Length - Lc	Х	9.38	9.38	9.38	10.12	10.12	11.12	11.25	Х	13.00	13.50	13.75	14.50	14.75	14.88
Equation I	L <sub>C</sub>				Stroke + 2 to 45" str		0.1	L. = 3 <sup>3</sup> / <sub>8</sub> "	L <sub>C</sub>			_	bke + X 5" stroke		O.L. = 6"
E-mation II	X <sub>1</sub>	12.00	12.00	12.00	12.75	12.75	13.75	13.88	X <sub>1</sub>		(To ne		<u>e - 95</u> 6 t whole n	umber)	
Equation II	L <sub>c</sub>		•		Stroke + > to 65" str			O.L. = 6"	L <sub>C</sub>			_	oke + X +	X <sub>1</sub>	L. = 6" + X <sub>1</sub>
Equation III	X <sub>2</sub>		(		Stroke - 6 4.5 rgest who	5 ole numbe	er)		X <sub>2</sub>			Not Re	equired		
Equation III	L <sub>C</sub>				oke + X <sub>1</sub> - stroke to				L <sub>C</sub>	Not Required					



		3 ST	AGE					4 STAGE				5 ST	AGE			6 ST	AGE
	SD53	SD63	SD73	SD83	SD93		SD64	SD74	SD84	SD94		SD75	SD85	SD95		SD86	SD96
Α	5 <sup>3</sup> / <sub>4</sub>	6 <sup>3</sup> / <sub>4</sub>	8	9 <sup>1</sup> / <sub>8</sub>	10 <sup>13</sup> / <sub>16</sub>	Α	63/4	8	9 <sup>1</sup> / <sub>8</sub>	10 <sup>13</sup> / <sub>16</sub>	Α	8	9 <sup>1</sup> / <sub>8</sub>	10 <sup>13</sup> / <sub>16</sub>	Α	91/8	10 <sup>13</sup> / <sub>16</sub>
В	6 <sup>3</sup> / <sub>8</sub>	7 <sup>3</sup> / <sub>8</sub>	8 <sup>5</sup> / <sub>8</sub>	9 <sup>7</sup> / <sub>8</sub>	11 <sup>3</sup> / <sub>4</sub>	В	73/8	8 <sup>5</sup> / <sub>8</sub>	97/8	11 <sup>3</sup> / <sub>4</sub>	В	8 <sup>5</sup> / <sub>8</sub>	97/8	11 <sup>3</sup> / <sub>4</sub>	В	97/8	11 <sup>3</sup> / <sub>4</sub>
С	43/4	5 <sup>3</sup> / <sub>4</sub>	63/4	7 <sup>7</sup> / <sub>8</sub>	9 <sup>3</sup> / <sub>8</sub>	С	53/4	6 <sup>3</sup> / <sub>4</sub>	7 <sup>7</sup> / <sub>8</sub>	9 <sup>3</sup> / <sub>8</sub>	С	6 <sup>3</sup> / <sub>4</sub>	7 <sup>7</sup> / <sub>8</sub>	93/8	С	7 <sup>7</sup> / <sub>8</sub>	9 <sup>3</sup> / <sub>8</sub>
D	33/4	43/4	53/4	63/4	7 <sup>7</sup> / <sub>8</sub>	D	43/4	5 <sup>3</sup> / <sub>4</sub>	63/4	7 <sup>7</sup> / <sub>8</sub>	D	5 <sup>3</sup> / <sub>4</sub>	63/4	7 <sup>7</sup> / <sub>8</sub>	D	6 <sup>3</sup> / <sub>4</sub>	7 <sup>7</sup> / <sub>8</sub>
Е	2 <sup>3</sup> / <sub>4</sub>	33/4	43/4	5 <sup>3</sup> / <sub>4</sub>	6 <sup>3</sup> / <sub>4</sub>	E	33/4	43/4	5 <sup>3</sup> / <sub>4</sub>	6 <sup>3</sup> / <sub>4</sub>	Е	43/4	5 <sup>3</sup> / <sub>4</sub>	6 <sup>3</sup> / <sub>4</sub>	E	5 <sup>3</sup> / <sub>4</sub>	6 <sup>3</sup> / <sub>4</sub>
F						F	23/4	33/4	43/4	5 <sup>3</sup> / <sub>4</sub>	F	33/4	43/4	5 <sup>3</sup> / <sub>4</sub>	F	43/4	5 <sup>3</sup> / <sub>4</sub>
G						G					G	2 <sup>3</sup> / <sub>4</sub>	33/4	43/4	G	33/4	4 <sup>3</sup> / <sub>4</sub>
Н						Н					Ι				Н	23/4	3 <sup>3</sup> / <sub>4</sub>
ΙE	3/4	1	1	1 <sup>1</sup> / <sub>4</sub>	1 <sup>1</sup> / <sub>4</sub>	ΙE	3/4	1	1	1 <sup>1</sup> / <sub>4</sub>	ΙE	3/4	1	1	ΙE	3/4	1
I <sub>R</sub>	1/2	3/4	3/4	1	1	I <sub>R</sub>	1/2	3/4	3/4	1	I <sub>R</sub>	1/2	3/4	3/4	I <sub>R</sub>	1/2	3/4
J	1 <sup>5</sup> / <sub>8</sub>	21/8	21/8	2 <sup>5</sup> / <sub>8</sub>	2 <sup>5</sup> / <sub>8</sub>	J	1 <sup>5</sup> / <sub>8</sub>	2 <sup>1</sup> / <sub>8</sub>	21/8	2 <sup>5</sup> / <sub>8</sub>	٦	1 <sup>5</sup> / <sub>8</sub>	21/8	2 <sup>1</sup> / <sub>8</sub>	J	1 <sup>5</sup> / <sub>8</sub>	2 <sup>1</sup> / <sub>8</sub>
	215	220	259	289	403		263	289	314	425		350	370	465		T.B.D.	T.B.D.
	146	150	175	194	268		191	209	226	304		259	272	335		T.B.D.	T.B.D.
Х	14.00	14.75	14.75	15.75	15.88	Х	15.25	15.75	16.25	16.88	Х	16.25	17.25	17.88	Х	17.75	18.38
L <sub>C</sub>		up t	Stroke 3 to 120" st		O.L. = 6"	L <sub>C</sub>		Stro 4 up to 14	oke + X 0" stroke	O.L. = 6"	L <sub>C</sub>		<u><e< u=""> + X 40" strok€</e<></u>	O.L. = 6"	L <sub>C</sub>	Stroke + 6 up to 150" s	
X <sub>1</sub>	(		stroke - 12 5 argest who		er)	X <sub>1</sub>	Stroke - 140 6 (To next largest whole number)				X <sub>1</sub>	(To next la	troke - 14 8 rgest whol		X <sub>1</sub>	Stroke 1 (To next largest	0
L <sub>C</sub>			roke 3 + X + stroke to	- X <sub>1</sub>	= 6" + X <sub>1</sub>	L <sub>C</sub>	Stroke 4 X + X 1  Stroke to max.				L <sub>C</sub>		+ X + X to 210" s		L <sub>C</sub>	Stroke + X 6 150" to 25	
X <sub>2</sub>		N	ot Requir	ed		X <sub>2</sub>					X <sub>2</sub>	(To next la	troke - 21 3.5 trgest whol	e number)	X <sub>2</sub>	Check Engine	
L <sub>C</sub>		N	ot Requir	ed		L <sub>C</sub>					$L_{C} = \frac{\frac{\text{Stroke}}{5} + X + X_{2} + 9}{211" \text{ stroke to max.}}$				L <sub>C</sub>	Check with Engineering	



# Following are service parts used on STANDARD DESIGN COMMERCIAL cylinders Note: Commercial Hydraulics manufactures many different types and designs of cylinders. If you are not sure of service parts required please refer to the Cylinder Assembly print.

Description	2.75" Stage	3.75" Stage	4.75" Stage	5.75" Stage	6.75" Stage	7.905" Stage	9.375" Stage
Vee-Packing Assembly with Shims	375-9009-017	375-9009-034	375-9009-018	375-9009-019	375-9009-020	375-9009-021	375-9009-022
	AJ3009-27	AJ3009-37	AJ3009-47	AJ3009-57	AJ3009-67	AJ3009-79	AJ3009-93
Wiper Ring	391-3882-061	391-3882-053	391-3882-054	391-3882-055	391-3882-056	391-3882-057	391-3882-058
	Y3026-27	Y3026-37	Y3026-47	Y3026-57	Y3026-67	Y3026-79	Y3026-93
Gland Bearing	391-2682-001	391-2682-002	391-2682-003	391-2682-004	391-2682-005	391-2682-006	391-2682-007
Glass Filled Nylon	J1003-1-1	J1003-1-2	J1003-1-3	J1003-1-4	J1003-1-5	J1003-1-6	J1003-1-7
Gland Bearing	375-9003-064	375-9003-044	375-9003-045	375-9003-046	375-9003-063	375-9003-041	375-9003-066
Bronze	AH1003-27	AH1003-37	AH1003-47	AH1003-57	AH1003-67	AH1003-79	AH1003-93
Wave Spring	391-3581-320	391-3581-300	391-3581-301	391-3581-302	391-3581-303	391-3581-304	391-3581-310
	P1327-27	P1327-37	P1327-47	P1327-57	P1327-67	P1327-79	P1327-93
Threaded Stop	375-4020-001	375-3020-002	375-6020-002	375-5020-001	375-7020-001	375-8020-001	375-9020-001
Non-Beveled	F4020	F3020	F6020	F5020	F7020	F8020	F9020
Threaded Stop	375-3020-110	375-3020-111	375-3020-112	375-3020-113	375-3020-114	375-3020-120	375-3020-119
Beveled	AG3020-1-27	AG3020-1-37	AG3020-1-47	AG3020-1-57	AG3020-1-67	AG3020-2-79	AG3020-2-93
Piston Bearing	391-2684-123	391-2684-069	391-2684-070	391-2684-071	391-2684-072	391-2684-073	391-2684-098
Glass Filled Nylon	U3023-32	U3023-42	U3023-52	U3023-62	U3023-72	U3023-84	U3023-98
Piston Bearing	N/A	375-9023-079	375-9023-038	375-9023-039	375-9023-040	375-9023-041	375-9023-042
Bronze		AM3023-42	AM3023-52	AM3023-62	AM3023-72	AM3023-84	AM3023-98
Cast Iron	391-2683-010	391-2683-014	391-2683-019	391-2683-024	391-2683-029	391-2683-034	391-2683-042
Piston Ring	X78-3.25-1	X78-4.25	X78-5.25	X78-6.25	X78-7.25	X78-8.437	X78-9.88
Steel External	391-1470-162	391-1470-163	391-1470-164	391-1470-165	391-1470-166	391-1470-167	391-1470-177
Headnut	YA3011-27	YA3011-37	YA3011-47	YA3011-57	YA3011-67	YA3011-79	YA3011-93
Bronze Internal	391-1470-079	391-1470-070	391-1470-071	391-1470-072	391-1470-073	391-1470-074	391-1470-075
Headnut	MA3011-27	MA3011-37	MA3011-47	MA3011-57	MA3011-67	MA3011-79	MA3011-93



# Repair Kits

# For Standard Commercial Hydraulics Single-acting, Dump Body Cylinders.

Commercial's genuine replacement parts are available in kits to rebuild or repack our dump body cylinders. These parts are the same as originally installed.

As with all hydraulic repairs, be sure your shop is properly equipped and that the work area is clean.

#### Rebuilding kits consist of:

1. Wiper Rings

5. Piston Bearings

2. Packing Assemblies

6. Set Screws

3. Gland Bearing Rings

7. Nylon Balls

4. Wave Springs

#### Repacking kits consist of:

1. Wiper Rings

2. Packing Assemblies

# Ordering Information

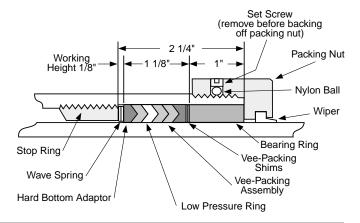
Please order by part number

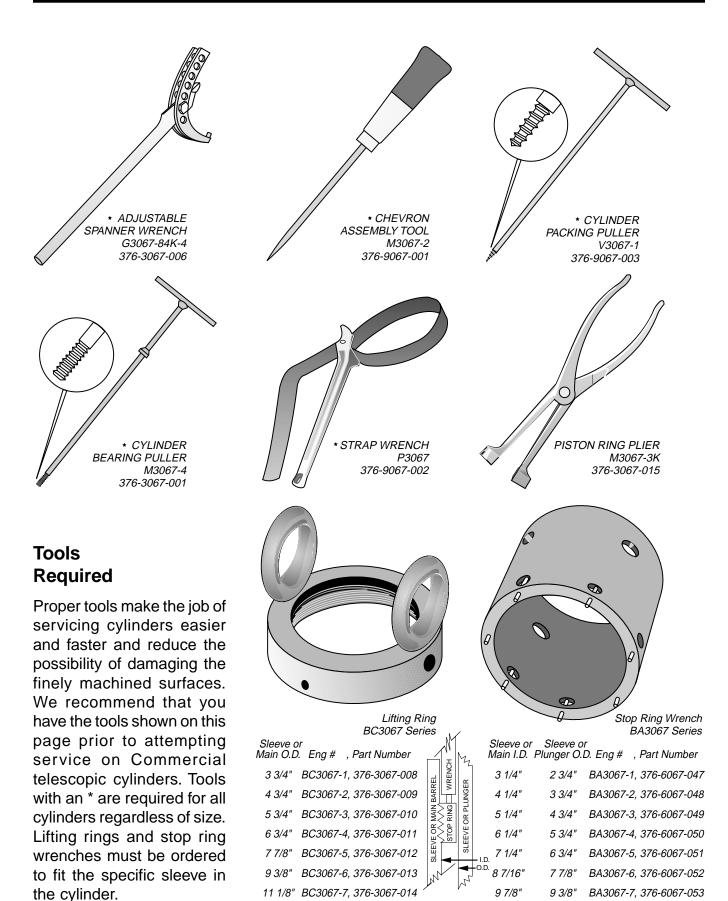
Cylinder Medel	Dobuild k	Cit Number	Danadak	Cit Niumbor
Cylinder Model	Rebuild r	Kit Number	кераск г	Kit Number
S53 Series	A300-253	375-1804-017	A300-11	391-1804-020
S62 Series	A300-256	375-1804-020	A300-262	375-1804-026
S63 Series	A300-254	375-1804-018	A300-104	391-1804-109
S64 Series	A300-257	375-1804-021	A300-230	391-1804-237
S73 Series	A300-255	375-1804-019	A300-238	375-1804-002
S74 Series	A300-258	375-1804-022	A300-239	375-1804-003
S84 Series	A300-259	375-1804-023	A300-49	391-1804-054
S85 Series	A300-260	375-1804-024	A300-22	391-1804-060
S95 Series	A300-261	375-1804-025	A300-240	375-1804-004

#### Cylinder Packing Installation

Remove shims from packing set and measure packing height under finger pressure. Add one shim for each 1/32" that the packing measures under 1-1/8". Measure depth to top of stop ring from top of tube. This dimension should be 2-1/4", however, it may vary slightly due to seating of the stop ring. Add one shim to the packing set for every 1/32" that this dimension measures over 2-1/4" or remove one shim from the packing set for every 1/32" that this dimension measures under 2-1/4". Install wave spring with gap edge against top of the cylinder stop ring. Soak packing in hydraulic oil for a few minutes. (Check bottom adapter. Only hard type can be used with the wave spring). Install packing, one ring at a time, in the proper sequence as shown in the sketch. Note: The soft, low pressure ring must be in the second position from the pressure side. Installed packing height need not be checked because wave spring will vary this

dimension. Installation of bearing and packing nut will compress wave spring for proper packing pre-load. Pull down tight against tube.







Packing, wipers and bushings are considered normal maintenance or service items. These items are subject to contamination from external and internal foreign materials, many of which are abrasive in nature, causing abnormal wear or damage to the parts, to the extent that replacements are required.

#### **WARNING!!**

Before working on a telescopic cylinder mounted on a truck or trailer unit, use supports or holding devices that will absolutely prevent the body from accidentally lowering. Place control valve in the "Lower" position to assure that all pressure has been relieved from the cylinder.

Because of our self-compensating Packing design, Standard Commercial Hydraulics Cylinders require no packing adjustment. For Commercial Packing Assembly Installation Procedure see Service Repair Kit Page.

Procedure for Adjusting Telescopic Cylinder Head Nuts. (For Cylinder designs using no Wave Springs and/or no U-Seals)

- 1.Loosen set screw (or set screws) in head nut that holds in packing of leaking stage.
- 2. Lightly tap head nut around circumference with a hammer.
- 3.Back head nut off 1/2 to 1 full turn using a spanner or chain wrench.

  (Note: If stage rotates when head nut is turned, hold stage with a strap wrench.)
- 4. Cycle cylinder 2 to 3 times to reset chevron vee packing.
- 5. Retighten head nut approximately 1/2 turn further than it was when it was loosened.
- 6. Tighten set screws.

## Procedure for Mis-Staging or Mis-Sequencing Cylinder.

- 1. Loosen set screws in head nut that holds in packing which fits over stage that is sticking.
- 2. Lightly tap head nut around circumference with a hammer.
- 3. Back head nut off 1/2 turn using a spanner or chain wrench.
- 4. Cycle cylinder, if cylinder still mis-stages, back head nut off another 1/2 turn.
- 5. Cycle cylinder, if cylinder still mis-stages, tighten the head nut of the next stage that is extending.
- 6. Tighten set screws.

### **Bleeding Air from Single-Acting Telescopic Cylinders.**

For smooth operation of these cylinders, it is advisable to bleed the air from the cylinder weekly.

Manual bleeding is accomplished by:

- 1. Empty the dump body of any material.
- 2. Remove the cover plate from the dog house of the dump body to access the bleeder valve.
- 3. Fully extend the cylinder, raising the EMPTY dump bed.
- 4. Lower the dump to within 1 foot from resting on the frame.
- 5. With the fingers, turn the bleeder valve in a counterclockwise direction. This opens the valve and allows the air to escape from the cylinder.
- 6. When a steady stream of oil comes from the bleeder, turn the valve in a clockwise direction until it is closed.

If these procedures fail to correct the problem, please contact an Authorized Service Center for Instructions.



#### 1) Function Test

Once the Cylinder is placed on the test stand and hydraulic lines attached, the cylinder will be cycled its full stroke a minimum of three (3) full cycles.

The cylinder will be rejected if it functions erratically. Erratic function is excessive chatter, slapstrick, stalling and uncorrectable misstaging.

#### 2) Proof Pressure Test

After the function test is performed the cylinder will be extended fully and pressure held for a minimum of thirty (30) seconds. This pressure will be 2500 psi or a pressure indicated on the assembly print. The cylinder will be rejected for external leakage or structural deformation. If the cylinder is double acting, it will be fully retracted and pressure held for a minimum of thirty (30) seconds. The pressure will be 2500 psi or a pressure indicated on the assembly print. The cylinder will be rejected for external leakage or structural deformation.

#### 3) Internal Bypass Test

This test will be performed on all double acting cylinders and can be done in conjunction with the Proof Pressure Test. The cylinder will be fully extended and pressure held at 2500 psi. The retract line will be removed and piston seal bypass will be determined by the flow out of this cylinder port. Excessive bypass will be a cause for cylinder rejection. The cylinder will be fully retracted and pressure held at 2500 psi. The extend line will be removed and the piston bypass will be determined by the flow out of this cylinder port. The cylinder will be rejected for excessive flow. When making this test the hydraulic line should be completely removed from the cylinder port, and the open line from the valve should be plugged or capped since a slight back pressure in the tank return line would spill oil from the line if not plugged.

Pass/Fail criteria if not noted on Assembly drawing is as follows:

Cast Iron Rings Normal Maximum leakage 1 GPM.

Bypass = 1/2 cubic inch per inch of bore diameter per minute.

Extend bypass would be 1/2 cubic inch per inch of plunger piston ring OD

per minute.

Retract bypass would be 1/2 cubic inch per inch of piston ring OD per minute

of each stage added together.

Example SD73 Series;

Extend bypass;  $1/2 \times 5.25 = 2.62$  cubic in / min.

Retract bypass;  $(1/2 \times 7.25) + (1/2 \times 6.25) + (1/2 \times 5.25) = 9.38$  cubic in / min.

Soft Seals Maximum leakage 5 drops per minute.

#### Please Note

#### Before Installing a New Cylinder in an old application

Has the problem been corrected that caused the original cylinder to fail?
Is the hydraulic fluid clean of all contamination, water, and entrapped air?
Are the hydraulic system relief valve pressures set and operating properly?
Is the mechanism or unit the cylinder is operating in good mechanical condition?



All cylinder parts, with the exception of a few items, are lubricated by the hydraulic oil in the circuit. Particular attention must be paid to keep the oil in the circuit clean. Whenever there is a hydraulic component failure (cylinder, pump, valve), and there is a reason to feel that metal particles may be in the system, the oil must be drained, the entire system flushed clean, and any filter screens thoroughly cleaned or replaced. New oil should be supplied for the entire system. Oil suitable and recommended for use in circuits involving Commercial cylinders should meet the following specifications:

# These suggestions are intended as a guide only. Obtain your final oil recommendations from your oil supplier.

#### **Viscosity Recommendations:**

Optimum operating viscosity is considered to be about 100 SSU.

- \* 50 SSU minimum @ operating temperature 7500 SSU maximum @ starting temperature
- \* 150 to 225 SSU @ 100<sup>o</sup> F. (37.8<sup>o</sup> C.) (generally) 44 to 48 SSU @ 210<sup>o</sup> F. (98.9<sup>o</sup> C.) (generally)

#### Other Desirable Properties:

Viscosity Index: 90 minimum Aniline point: 175 minimum

#### **Additives Usually Recommended:**

Rust and Oxidation (R & O) Inhibitors Foam Depressant

#### Other Desirable Characteristics:

Stability of physical and chemical characteristics.

High demulsibility (low emulsibility) for separation of water, air and contaminants.

Resistant to the formation of gums, sludges, acids, tars and varnishes.

High lubricity and film strength.

#### **General Recommendations:**

A good quality hydraulic oil conforming to the characteristics listed above is essential to the satisfactory performance and long life of any hydraulic system.

Oil should be changed on regular schedules in accordance with the manufactures recommendations and the system periodically flushed.

Oil operating temperature should not exceed 200° F. (93° C.) with a maximum of 180° F. (82° C.) generally recommended. 120° F. to 140° F. (50° C. to 60° C.) is generally considered optimum. High temperatures result in rapid oil deterioration and may point out a need for an oil cooler or a larger reservoir. The nearer to optimum temperature, the longer the service life of the oil and the hydraulic components.

Reservoir size should be large enough to hold and cool all the fluid a system will need, yet it should not be wastefully large. Minimum required capacity can vary anywhere between 1 and 3 times pump output. The reservoir must be able to hold all of the fluid displaced by retracted cylinders when the system is not operating, yet provide space for expansion and foaming.

Oil poured into the reservoir should pass through a 100 mesh screen. Pour only clean oil from clean containers into the reservoir.

Never use Crank Case Drainings, Kerosene, Fuel Oil, or any Non-Lubricating Fluid, such as Water.

#### Approximate SSU at ...

Oil Grade	100°F. (37.8°C.)	210° F. (98.9°C.)		
SAE 10	150	43		
SAE 20	330	51		

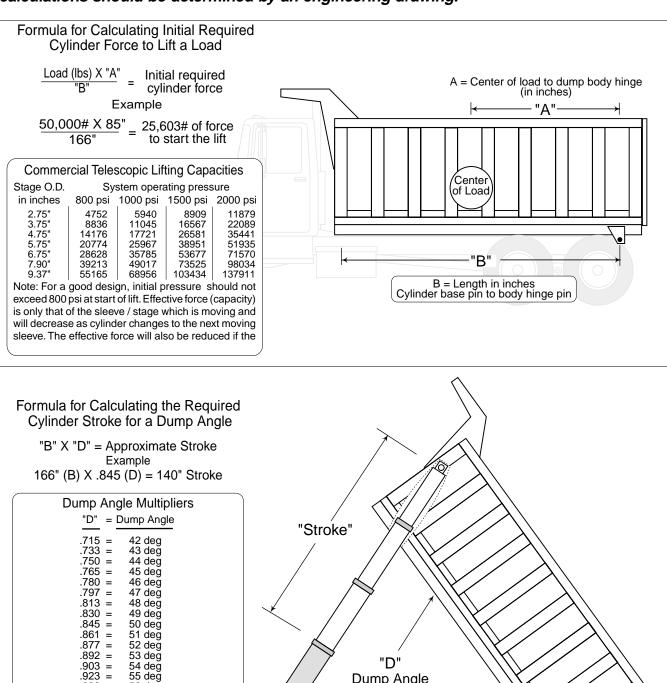
#### **Normal Temperatures:**

 $0^{\rm O}$  F. (-18° C.) to  $100^{\rm O}$  F. (37.8° C.) ambient  $100^{\rm O}$  F. (37.8° C.) to  $180^{\rm O}$  F. (82.2° C.) system

Be sure the oil you use is recommended for the temperature you expect to encounter.



Note: This guide is for use to determine approximate stroke and lifting requirements for a Front Mount Dump Body, they should not be used for Frameless Type Trailers, Underbody, Telescopic Farm (slant forward or rearward), or Scissor Type Hoists. Final dimensions and calculations should be determined by an engineering drawing.



.923 =

.954 =

56 deg

57 deg Normal minimum dump angle is 45 degree Normal maximum dump angle id 57 degree

.939

**Dump Angle** 

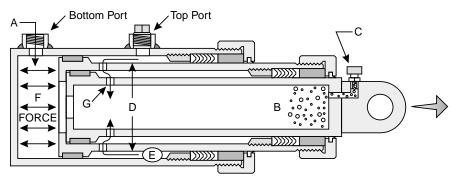
"B"

B = Length in inches Cylinder base pin to body hinge pin

Material	lbs. / cu. yd.	tons / cu. yd.	ights of Materials  Material	lbs. / cu. yd.	tons / cu. yd	
Andesite stone	4887	2.44	Earth & sand, wet	3240	1.62	
Ashes	1080	0.52	Fire Brick	3915	1.95	
Asphalt	2700	1.35	Fire Clay	3510	1.75	
Asphaltum			-	1150	0.57	
Basalt rock	2349 1.17 Garbage 4887 2.44 Gravel, dry		Gravel, dry	2970	1.48	
Brick, soft clay	2718	1.35	Gravel, out of water	1620	0.81	
Brick, hard clay	3397	1.69	Granite	4536	2.26	
Brick, pressed	3806	1.9	Lime, quick, loose	1431	0.71	
• •	4246	2.12		+	0.71	
Brick, paving			Lime, quick, shaken	1485		
Block, paving	3694	1.84	Limestone, solid	4536	2.26	
Bluestone	2970	1.48	Limestone, loose	2592	1.29	
Cement, natural	1512	0.75	Marble, solid	4455	2.22	
Cement, Portland	2430	1.21	Marble, loose	2592	1.29	
Coment, Portland, set	4941	2.47	Mortar, set	2781	1.39	
Cindors	1863	0.93	Mud, dry	2430	1.21	
Clovedra	1080	0.54	Mud, packed	3105	1.55	
Clay, dry	1701	0.85	Mud, wet	2916	1.45	
Clay, wet	2970	1.48	Pitch	1863	0.93	
Clay & gravel, dry	2700	1.35	Plaster of Paris	2646	1.32	
Coal, anthracite	1536	0.76	Powder, blasting	1682	0.84	
Coal, bituminous	1275	0.64	Quartz	4374	2.18	
Coke	837	0.42	Rubbish	199.8	0.09	
Concrete, cinders	2970	1.48	Sand, dry, loose	2619	1.30	
Concrete, gravel	4104	2.05	Sand, wet	3186	1.59	
Concrete, limestone	4050	2.02	Sandstone	4023	2.01	
Concrete, sandstone	3915	1.95	Slag, bank	1890	0.94	
Concrete, trap rock	4185	2.09	Slag, screenings	2700	1.35	
Crushed stone	2700	1.35	Slag, machine	2592	1.29	
Earth, dry, loose	1890	0.94	Slag, sand	1485	0.74	
Earth, damp, loose	2106	1.05	Shale	4374	2.18	
Earth, damp packed	2592	1.29	Slate	4725	2.31	
Earth & gravel, dry	2700	1.35	Tar	1674	0.83	
Earth & gravel, wet	3240	1.62	Tile	2970	1.43	
Earth & sand, dry	2709	1.35	Trap stone	5849	2.52	
	Steel & Al	uminum Gaug	je, Thickness and	l Weight		
Steel	Т			1 .=		
3/8"		0.375		15.320 lbs. per sq. ft.		
1/4" (approx. 3 Ga.)		0 250		10.200 lbs. per sq. ft.		
3/16" (approx. 7 Ga.)		0.188		7.650 lbs. per sq. ft.		
8 Ga.		0.164		6.875 lbs. per sq. ft.		
9 Ga.		0.149		6.250 lbs. per sq. ft.		
10 Ga.		0.134		5.625 lbs. per sq. ft.		
11 Ga.		0.120		5.000 lbs. per sq. ft.		
12 Ga.		0.105		4.375 lbs. per sq. ft.		
13 Ga.		0.090		3.750 lbs. per sq. ft.		
14 Ga. Aluminum			0.075	3.125 lb	os. per sq. ft.	
3/8"			0.375	5 18 lbs	s. per sq. ft.	
		0.375		3.53 lbs. per sq. ft.		
1/4"			·	2.65 lbs. per sq. ft.		
1/4" 3/16"			0.188	2.65 lbs	s perso ft	



# Extending

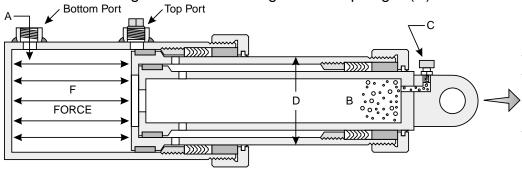


#### To Extend:

High pressure oil from the pump is directed by the control valve through the port (A) to fill the cylinder. Any air in the system is trapped in the end of the cylinder (B) and may be bled off through the bleeder valve (C).

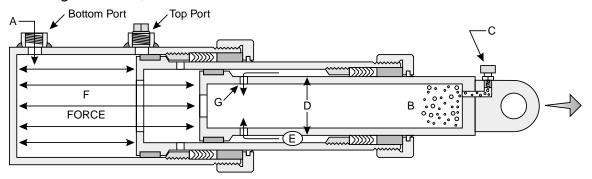
Generally, bleeding is only necessary on initial start up or if air has been allowed to enter the system (Note: the bleeder valve must be located at the highest part of the cylinder).

Oil pushes on the bottom of the sleeve or plunger forcing (F) it to move out. The outside diameter or sealing area of the moving sleeve or plunger (D) determines the effective area.

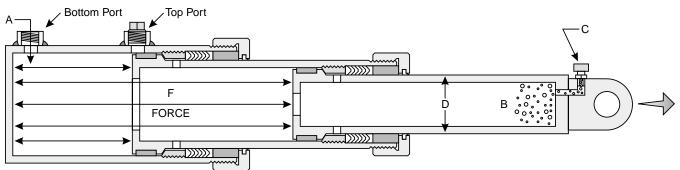


As the sleeve or plunger moves out, the oil trapped between (E) the sleeve or plunger wall is released through transfer holes (G) which are drilled in the sleeve or plunger.

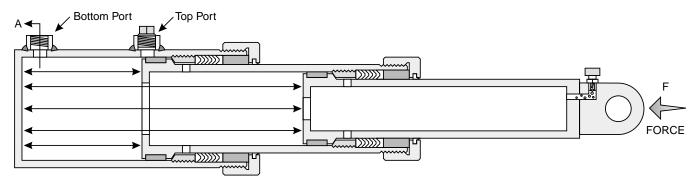
Under normal operating conditions the largest diameter moving sleeve extends first, then the next largest sleeve, etc.



So at a given PSI (pressure) and GPM (gallons per min.) the cylinder will develop less force and increase in speed as it changes to the next moving stage.

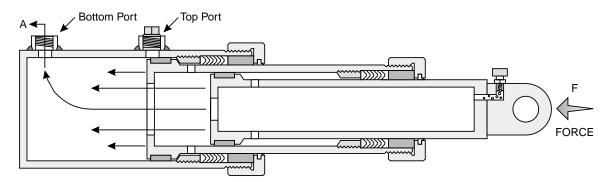


# Retracting

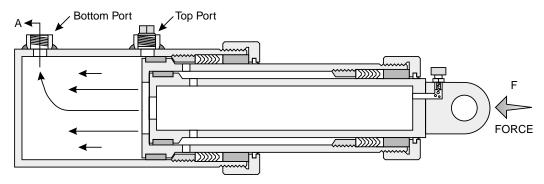


#### To Retract:

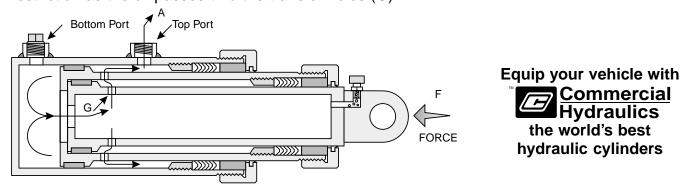
A single acting cylinder must be retracted by gravity or mechanical means (F).



Under normal operating conditions the plunger or smallest diameter sleeve is pushed closed first, forcing oil out through port (A), then the next smallest, etc.



**Note:** If the top port is used, the speed of the cylinder may be reduced because of the oil flow restriction as the oil passes thru the transfer holes (G).



# **Single-Acting Telescopic Dump Cylinders**

WARNING!! Telescopic cylinders commonly installed on dumping vehicles are devices intended to provide only a lifting force. The cylinder is not a structural member, and is not designed for, nor intended to provide stability to the dumping vehicle. The cylinder can bend, buldge or separate causing the dump body to drop suddenly, resulting in serious personal injury or property damage, if the following warnings are not observed.

Cylinders are not to be used as a stabilizer on a dump body or dump trailer. The cylinder is strictly a lifting device and is not a structural member of the dumping unit.

The cylinder should float in the pin mountings. The cylinder should be installed with 1/8" to 3/16" of clearance between the pin and the pin hole if the mounting eye is wider than 5", or with 1/16" to 1/8" clearance if the mounting eye is less than 5" wide. There should be a clearance of 1/8" to 1/4" per side on eyes less than 5" wide and 1/4" to 1/2" clearance per side on eyes in excess of 5" wide. This is to allow the body to sway slightly while dumping, without putting a side load on the cylinder. The cylinder plunger or one of the sleeves should be extended a minimum of 1/4" when the dump body is in the down position. The cylinder end mounts should be lubricated regularly.

Cylinders cannot withstand side pressures from a dump unit leaning. A tractor trailer unit must be in a straight line when dumping, not jackknifed. All dump units must be on firm, level ground and not operated during heavy crosswinds. Failure to do so may cause the unit to upset.

Do not overload the dump unit. The load must be evenly distributed during loading and unloading. Do not jerk or slingshot dump unit in an attempt to free a sticking or frozen load. Pulling forward (or backing up) and hitting the brakes, or lowering the body part way and then quickly engaging the valve in the "HOLD" or "RAISE" position will cause a tremendous pressure spike. This pressure spike may bulge or split one of the larger stages of the cylinder.

When lowering a load that is sticking, the dump unit must be feathered down slowly to avoid a high pressure build up in the cylinder.

Do not operate cylinder at pressures above factory recommended operating pressures (Normally 2,000 P.S.I unless otherwise approved).

The driver should stay at the controls during the entire dumping operation. If the body starts to lean to one side, the driver should immediately lower the body. It is important to feather the control valve into the hold position to avoid a pressure spike in the cylinder.

Do not operate cylinder with personnel or equipment alongside.

The dump unit must be lowered completely before moving unit.

Do not drive with P.T.O. or Hydraulic Pump engaged.

Hydraulic hoses should be inspected regularly and replaced if worn out or damaged.

Hydraulic oil should be inspected and or changed regularly and whenever a new cylinder is installed.

A damp to light film of oil on each plunger or stage of a telescopic cylinder indicates good cylinder operation. After many cycles of the cylinder, a small accumulation of oil may be noticed on the plungers or sleeves at the head nuts. This should not be mistaken for packing leakage.

Cylinder should be free of entrapped air. It is advisable to bleed air from cylinder weekly for a smooth operation.

Before working on a telescopic cylinder mounted on a truck or trailer unit, use supports or holding devices that will absolutely prevent the body from accidentally lowering. Place control valve in the "LOWER" position to assure that all pressure has been relieved from the cylinder.



#### **1. Wind:**

Do not attempt dumping operations in high gusty wind conditions. If possible, raise the dump body directly into the wind.

#### 2. Terrain:

Uneven terrain, causing the trailer wheels to be 3 1/2 inches to 4 inches higher than the other side, puts the top of the body 12 inches to 14 inches off center when the cylinder is fully extended. On fresh fill, loaded trailer wheels may sink on one side, again setting up potential tipover. On road construction, the crown is also critical on spread application, as in dumping on a slope. A 4 inch plus, height differential of wheels on an axle 8 feet wide, is another rollover potential.

## 3. Hung Loads:

A hung load is commodity that does not discharge when a dump body is raised to an elevated position. This condition exists due to surface adhesion between the commodity and the interior of the dump body. To avoid a tipover due to a hung load, the driver should be warned by an observer or be aware of the material's moisture content, if this condition exists, immediately lower the dump body.

# 4. Humping:

Humping is a rapid acceleration / deceleration method used to loosen a hung load from a trailer. If the load is off center and the trailer is moved, a tipover may occur. Also, serious damage to the hoist may occur if an extreme humping motion is used to get a sticky load out of the body.

#### 5. Jackknife:

A jackknife position of the tractor with the trailer is not recommended when dumping. In a jackknifed position, the upper coupler pivots on bearings, contributing nothing to dump stability. When the tractor and trailer are straight, the coupler bearings are normally 34 inches apart, assisting in stabilizing the dump.

#### 6. Tires:

A blown tire or a severely underinflated tire can cause dump instability, when dumping. Always check tires visually for cuts or punctures by nails and metal. Make sure all tires are inflated properly. Proper tire inflation also improves wear and fuel economy.

# 7. Stay at the Controls:

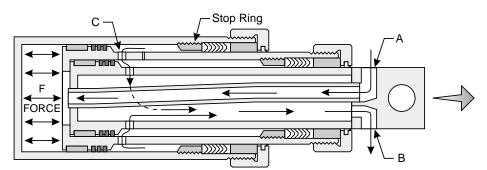
An operator who fails to stay at the controls will never control the body when it starts to lean over for a tipover. If a problem exists, the body can be lowered and the operator can check and remedy any potential problems, then resume dumping the load.

# 8. Overloading:

Overloading is a very common occurrence that aggravates all the above conditions that cause tipovers.



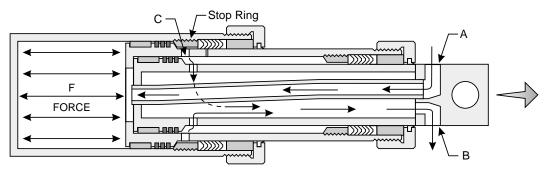
# Extending



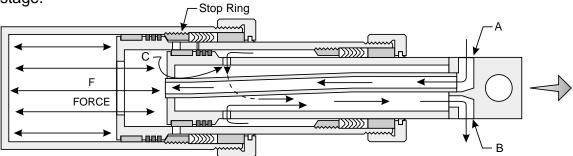
#### To Extend:

High pressure oil is directed by the control valve into port A. The oil passes through the transfer tube in the rod to the base of the cylinder.

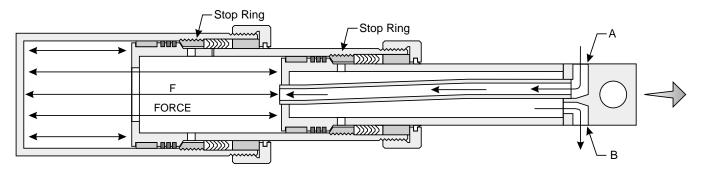
The pressure acts on the effective area (area of the largest piston) and extends all stages to the first stop ring. The next stage then begins to extend.



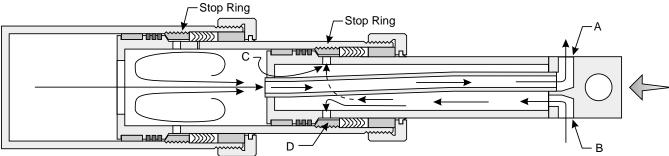
The effective area of each stage is figured from the inside diameter of the next largest stage. Each stage extends in its turn to the stop ring. So at a given PSI (pressure) and GPM (gallons per min.) the cylinder will develop less force and increase in speed as it changes to the next moving stage.



Oil trapped between the sleeves escapes through holes (C) in each sleeve and returns to the tank through port B.

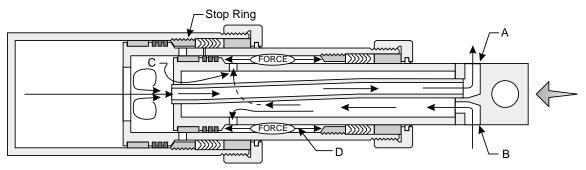


# Retracting

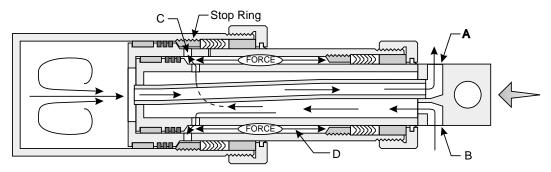


#### To Retract:

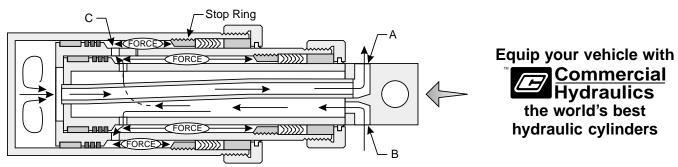
High pressure oil is directed by the control valve into port B. The pressure is applied to the effective area (D) of the plunger which retracts first. Each stage from the smallest to the largest



retracts in its turn, however, THE EFFECTIVE AREA FOR RETRACTING EACH STAGE IS THE AREA (D) OF THE PLUNGER.



Oil inside the cylinder is forced out of port A. Because of the area differential, the flow into port B must be multiplied by this differential to determine the flow out of port A. It may be necessary to install a dump-to-tank valve to speed up the retracting cycle.



# **Double-Acting Telescopic Cylinders**

A double-acting telescopic cylinder should be fully retracted when not in use.

A double-acting telescopic cylinder should not be extended until it has been fully retracted. A partially extended cylinder with pressure relieved may drift out of position. This can happen if a cylinder experiences vibration, such as an ejector or push out cylinder does in a refuse body. If this happens and the cylinder is then extended, the out of position plunger or sleeves will rapidly reposition themselves and possibly cause high pressure oil to be trapped on the retract side of the cylinder. This could cause a stage / sleeve to bulge and or the packing and bearings to be blown out from under a head nut.

Do not operate a packer / ejector cylinder with a misaligned blade. If the packer / ejector blade is bent, damaged, or if the blade slide shoes or guide track assembly are worn out, this can cause excessive side loading to the cylinder damaging it internally or bending a plunger or sleeve.

If the hydraulic system uses quick disconnects (such as on a transfer trailer) or holding / lock type valves, make sure they are properly connected. If not, oil may become trapped in the cylinder causing an excessive pressure build up and damage to the cylinder. This is particularly true if there is a blockage on the retract side of the cylinder and the cylinder is then extended. This could internally intensify the pressure 10 times or more inside the cylinder. (Example; The pump develops 2,000 P.S.I. trying to extend the cylinder, oil pressure trapped on the retract side of the cylinder could see 20,000 P.S.I.)

Do not operate a cylinder at pressures above factory recommended operating pressures (Normally 2,000 P.S.I. unless otherwise approved). Make sure hydraulic pump is developing required G.P.M., Double-acting telescopic cylinders normally require 15 G.P.M. to retract properly.

Most double-acting telescopic cylinders will self bleed themselves of air. Upon installation of a new cylinder this will require cycling the cylinder approximately 10 times to the complete extend and retract positions with no load against it. Check to make sure stages sequence properly. When extending, the largest stage should move first then the next largest, etc. and when retracting, the smallest should move first then the next smallest, etc.

On Roll Off units, if the Lift Cylinders are not completely extended when a container is being pulled onto the hoist, the lift cylinders may be pulled open by the weight of the load. Then as the container is pulled over center, the cylinders will be forced closed until they hit the column of oil inside the cylinders causing a sudden pressure surge. If the lift cylinders are pulled open by the load, they should be extended with the control valve to fill them with oil before pulling the container on the rest of the way. Care should be taken if moving Roll Off unit with tilt cylinders extended, avoiding sudden stops or jolts.

On Roll Off units dropping off a loaded container, feather control valve to avoid any pressure surges in the reeving / cable cylinders as gravity pulls the container to the ground.

On Roll Off units, if the container is not evenly loaded and is heavy on one side, the lift cylinders may mis-stage. When the plunger / sleeves attempt to correct themselves, there may be a sudden pressure surge, possibly damaging the cylinder.



#### **STORAGE**

It pays to keep spare hydraulic cylinders on hand for use when you need them. But, you must know and follow these recommended storage practices or the cylinders can be ruined. Hydraulic cylinders, though often large and unwieldy, are precision machines with finely finished parts and close tolerances. And they're expensive. So handle them with care.

For optimum storage life, hydraulic cylinders should be kept in an environment that is protected from excessive moisture and temperature extremes. A hot, dry dessert climate with cold nights, for example, must be accommodated when choosing the storage area. Daytime heat quickly bakes oil out of sealing materials, which causes leaks and rapid wear when the cylinder is placed in service. Cooling at night causes water condensation and corrosion damage to wear surfaces. Storage areas that allow exposure to rain, snow and extreme cold must like wise be avoided.

It's best to store cylinders indoors if possible. But indoors or out, be sure that plugs or closures are properly installed in all ports to keep out moisture and dirt. However, overtightening of port plugs should be avoided. Widely varying temperatures and tightly closed ports may cause pressure inside the cylinder to build up to the point where the piston moves far enough to expose the rod to corrosion or contamination. Try to choose a storage location where the cylinders are protected from physical damage. Even a little ding from a falling bar or forklift tine can cause trouble later.

Cylinders, Particularly large ones, should be stored closed in a vertical position with the rod end down. Be sure they're blocked securely to keep them from toppling. Storing with the rod ends down keeps oil on the seals, which protects them from drying out. This is more critical with fabric and butyl seals than with urethane sealing materials. Storing single-acting cylinders with the rod end up can cause port closures to pop open and leak, exposing the sleeves to corrosion damage and contamination. Storing with the rod end down also discourages the temptation to lift a cylinder by the rod eye – a dangerous practice. If horizontal storage cannot be avoided, the rod or cylinder should be rolled into a new position every two months or so to prevent drying, distortion and deterioration of the seals. Don't forget that a cylinder can be a major source of contamination. A small scratch or nick on the sleeve will quickly shred packing and contaminate the system. Store cylinders carefully and keep them clean.

The following procedures should be followed in order to prevent oxidation and maintain the surfaces of a mounted hydraulic cylinder during idle periods. These idle periods may include; inventory units, demo units, out of service units, etc.

- · All machined surfaces left expose should be coated with a light film of grease, if not oxidation will occur.
- · If oxidation is present, apply a light coat of oil to the surfaces.
- · Buff surfaces with 320 or 400 grit sandpaper. Do not buff surfaces up and down the length, buff only around the circumference.
- · If after buffing, the surfaces show evidence of oxidation damage i.e., pitting, the cylinder should be inspected by an authorized service center for evaluation.
- · Operation of a hydraulic cylinder with surface damage will shorten the longevity and preclude any warranty express or implied.

#### **INSTALLATION**

- ·Cleanliness is an important consideration, and Parker cylinders are shipped with the ports plugged to protect them from contaminants entering the ports. These plugs should not be removed until the piping is to be installed. Before making the connection to the cylinder ports, the piping should be thoroughly cleaned to remove all chips or burrs which might have resulted from threading or flaring operations. One small foreign particle can cause premature failure of the cylinder or other hydraulic system components. If oxidation is present, apply a light coat of oil to the surfaces.
- Proper alignment of the cylinder piston rod and its mating component on the machine should be checked in both the extended and retracted positions. Improper alignment will result in excessive rod gland and/or cylinder bore wear.
- · Cylinders operating in an environment where air drying material are present such as fast- drying chemicals, paint, or welding splatter, or other hazardous conditions such as excessive heat, should have shields installed to prevent damage to the piston rod and piston rod seals.



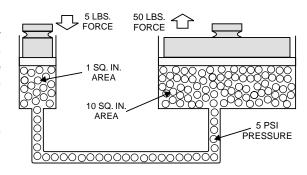
The basis for all hydraulic systems is expressed by Pascal's law which states that the pressure exerted anywhere upon an enclosed liquid is transmitted undiminished, in all directions, to the interior of the container. This principle allows large forces to be generated with relatively little effort. As illustrated, a 5 pound force exerted against a 1 inch square area creates an internal pressure of 5 psi. This pressure, acting against the 10 square inch area develops 50 pounds of force.

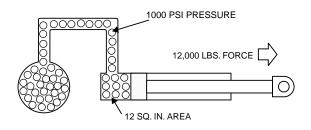
In a basic hydraulic circuit, the force exerted by a cylinder is dependent upon the cylinder bore size and the pump pressure. (There is no force generated unless there is resistance to the movement of the piston). With 1000 psi pump pressure exerted against a 12 square inch piston area (approximately 4" dia.), a force of 12,000 pounds is developed by the cylinder. The speed at which the piston will move is dependent upon the flow rate (gpm) from the pump and the cylinder area. Hence, if pump delivery is 1 gallon per minute (231 cu. in./min.) the cylinder piston will move at a rate of 19.25 in./min. (231 cu. in. ÷ 12 sq. in./min.).

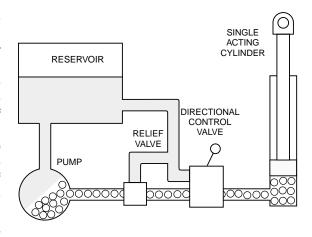
The simplest hydraulic circuit consists of a reservoir, pump, relief valve, 3-way directional control valve, single acting cylinder, connectors and lines. This system is used where the cylinder piston is returned by mechanical force. With the control valve in neutral, pump flow passes through the valve and back to the reservoir. With the valve shifted, oil is directed to the piston side of the cylinder, causing the piston to move, extending the rod. If the valve is returned to neutral, the oil is trapped in the cylinder, holding it in a fixed position, while pump flow is returned to the reservoir. Shifting the valve in the opposite direction permits the oil to pass through the valve back to the reservoir. The relief valve limits the system pressure to a pre-set amount. Relief valves are commonly incorporated into the directional control valve.

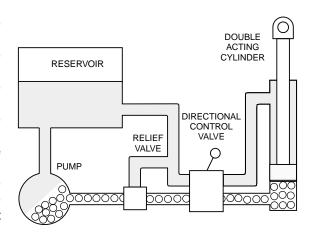
A hydraulic system using a double acting cylinder and a 4-way valve differs from a single acting cylinder system in that the cylinder can exert force in both directions. With the control valve in neutral, flow is returned to the reservoir. When shifted in one direction, oil is directed to the piston side of the cylinder, causing the cylinder to extend. Oil from the rod side passes through the valve back to the reservoir. If the valve is shifted to neutral, oil in the cylinder is trapped, holding it in a fixed position. When the valve is shifted in the opposite position, oil is directed to the rod side of the cylinder, causing the cylinder to retract. Oil from the piston side passes through the valve back to the reservoir.

Cylinder extend force is the result of pressure (psi) times the piston area (minus any force resulting from the pressure acting against the rod side of the piston). Retract force is a result of the pressure (psi) times the area difference between the rod and the piston (minus any force resulting from pressure acting against the piston side of the cylinder).







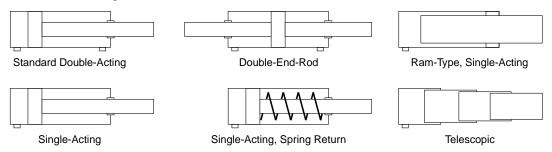


All of the systems described above are open center systems due to the oil flowing through the control valve back to the tank. Most systems are this type. Closed center systems use control valves with the inlet port blocked and variable displacement pumps. With the control valve in neutral, the pump is "de-stroked" to zero flow.



The function of a cylinder in a fluid power system is to convert energy in the fluid stream into an equivalent amount of mechanical energy. Its power is delivered in a straight-line, push-pull motion.

**Graphic Symbols:** Following diagram illustrates standard ANSI (American National Standards Institute) graphic symbols for use in circuit diagrams. Six of the more often used are shown:

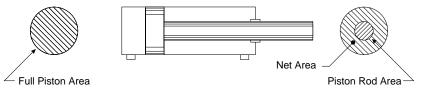


Standard ANSI (American National Standards Institute) Graphic Symbols for Use in Circuit Diagrams.

The standard double-acting cylinder with piston rod out one end, is used in the majority of applications. It develops force in both directions of piston travel. The double-end-rod type is a variation of the standard cylinder but having a piston rod extending out both end caps. It is occasionally used where it is necessary to have equal area on both sides of the piston, such as a steering application, or where one of rod extensions is to be used for mounting a cam for actuation of a limit switch, or for mounting a stroke limiting stop. The single-acting cylinder develops force in one direction, and is retracted by the reactive force from the load or an internal or external spring. The single-acting ram is a construction often used on fork lift mast raise, or a refuse body tailgate raise, or a high tonnage press cylinders. The telescoping cylinder is built in both single-acting and double-acting types. Its purpose is to provide a long stroke with a relatively short collapsed length. The single-acting telescopic is a construction often used to raise dump trucks and dump trailers. The double-acting telescopic is a construction often used to pack and eject the load.

#### Force Produced by a Cylinder:

A standard double-acting cylinder has three significant internal areas. The full piston area when exposed to fluid pressure, produces force to extend the piston rod. The amount of this force, in pounds, is calculated by multiplying piston square inch area times gauge pressure, in PSI.

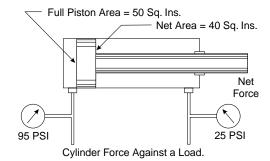


Significant Areas in a Double-Acting Cylinder, Single-End-Rod Type.

The "net" area on the front side of the piston is less than full piston area because part of the piston surface is covered by the rod. Net area is calculated by subtracting rod area from full piston area. Because net area is always less than piston area, cylinder force for rod retraction is always less than can be developed for extension when working at the same pressure.

**Cylinder Force Against a Load:** The force which a cylinder can exert against a load is determined by making two calculations. First, extension force is calculated according to piston area and PSI pressure against it. Then, the opposing force on the opposite side of the piston is calculated the same way. Net force against a load is the difference between the two.

Caution! It is incorrect, on a single-end-rod cylinder to calculate cylinder net force as piston area times  $\Delta P$  (pressure drop, psid) across the piston. This is true only for double-end-rod cylinders which have equal areas on both sides of the piston.

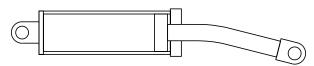


Example: The extension force is 95 PSI x 50 sq. in. = 4750 lbs. The opposing force on the rod side is 25 PSI x 40 sq. in. = 1000 lbs. Therefore, the net force which the cylinder can exert against a load in its extension direction is 4750 - 1000 = 3750 lbs. In making cylinder force calculation we sometimes assume that the opposite side of the piston is at atmospheric pressure, and that the counter-force is zero. On some kinds of loads this can lead to serious error. Note: Most designers try to eliminate back pressure to get full extend force, but there will always be back pressure.

Standard catalog cylinder models are not designed to take any appreciable side load on the piston rod. They must be mounted so the rod is not placed in a bind at any part of the stroke. If the direction of the load changes during the stroke, hinge mounting must be used on both the rod end and rear end. Use guides on the mechanism, if necessary, to assure that no side load is transmitted to the cylinder rod or piston.

#### **Rod Buckling**

Column failure or buckling of the rod may occur if the cylinder stroke is too long relative to the rod diameter. The exact ratio of rod length to rod diameter at which column failure will occur cannot be accurately calculated, but the



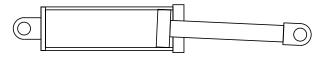
"Column Strength" table in this manual shows suggested safe ratios for normal applications.

#### **Tension and Compression Failures**

All standard cylinders have been designed with sufficiently large piston rods so failure will never occur either in tension or compression, provided the cylinder is operated within the manufacturers pressure rating.

#### **Rod Bearing Failure**

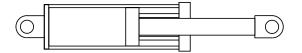
Rod bearing failures usually occur when the cylinder is at maximum extension. Failures occur more often on hinge or trunnion mount cylinders, in which the rear support point is located considerably behind the rod bearing. If space



permits, it is wise to order cylinders with longer stroke than actually required, and not permit the piston to approach to the front end while under full load.

#### Stop Collar

On those application where it is necessary to let the piston "bottom out" on the front end, the cylinder may be ordered with a stop collar. The stop collar should be especially considered on long strokes if the distance



between support exceeds 10 times the rod diameter, if the maximum thrust is required at full extension, and if the cylinder has a rear flange, clevis, tang, or trunnion mounting.

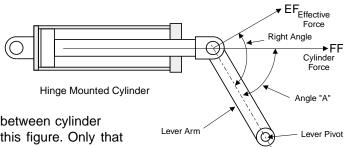
#### MINIMUM PISTON ROD DIAMETER

Figures in body of chart are suggested minimum rod diameters, in inches.

Load,	Exposed Length of Piston Rod, Inches / Rod Diameter, Inches							
Pounds	10"	20"	40"	60"	70"	80"	100"	120"
1,000			3/4	1				
1,500			13/16	1-1/16				
2,000		5/8	7/8	1-1/8	1-1/4	1-3/8		
3,000		11/16	15/16	1-3/16	1-3/8	1-1/2		
4,000		3/4	1	1-1/4	1-7/16	1-9/16	1-7/8	
6,000	13/16	7/8	1-1/8	1-3/8	1-9/16	1-5/8	1-7/8	
8,000	15/16	1	1-3/16	1-1/2	1-5/8	1-3/4	2	2-1/4
10,000	1	1-1/8	1-5/16	1-9/16	1-3/4	1-7/8	2-1/8	2-3/8
15,000	1-3/16	1-1/4	1-7/16	1-3/4	1-3/4	2	2-1/4	2-1/2
20,000	1-3/8	1-7/16	1-5/8	1-7/8	2	2-1/8	2-7/16	2-3/4
30,000	1-11/16	1-3/4	1-7/8	2-1/8	2-1/4	2-3/8	2-11/16	3
40,000	2	2	2-1/8	2-3/8	2-1/2	2-5/8	2-7/8	3-1/4
60,000	2-3/8	2-7/16	2-1/2	2-3/4	2-3/4	2-7/8	3-1/4	3-1/2
80,000	2-3/4	2-3/4	2-7/8	3	3	3-1/4	3-1/2	3-3/4
100,000	3-1/8	3-1/8	3-1/4	3-3/8	3-1/2	3-1/2	3-3/4	4
150,000	3-3/4	3-3/4	3-7/8	4	4	4-1/8	4-3/8	4-1/2
200,000	4-3/8	4-3/8	4-3/8	4-1/2	4-3/4	4-3/4	4-7/8	5
300,000	5-3/8	5-3/8	5-3/8	5-1/2	5-1/2	5-1/2	5-3/4	6

#### Cylinder Working a Rotating Lever:

A cylinder working a hinged lever can exert its maximum force on the lever only when the lever axis and cylinder axis are at right angles. When Angle "A" is greater or less than a right angle, only part of the cylinder force is effective on the lever. The cylinder force is found by multiplying the full



cylinder force times the sine (sin) of the least angle between cylinder and lever axes. Cylinder Force, FF, is horizontal in this figure. Only that portion, EF, which is at right angles to the lever axis is effective for turning the lever. The value of EF varies with the acute angle "A" between the cylinder and lever axis.

Example: Find the effective force exerted by a 3-inch bore cylinder against a lever when the cylinder is operating at 3000 PSI and when its axis is at an angle of 55 degrees with the lever axis.

First , find the full force developed by the cylinder: FF (full force) = 7.07 (piston area) x 3000 PSI = 21,210 lbs. Next, find the effective force at  $55^{\circ}$ : EF (effective force) = 21,210 x 819 (sin  $55^{\circ}$ ) = 17,371 lbs.

Since maximum cylinder force is delivered in the right angle position, the hinge points for the cylinder and lever should be located, if possible, so the right angle falls close to the lever position which requires the greatest torque (force).

Note: The working angles on a hinged units, such as a dump truck, refuse body packer blade, or a crane, are constantly changing, it may be necessary to construct a rough model on a sheet of paper, to exact scale, with cardboard arms and thumbtack hinge pins. This will show the point at which the greatest cylinder thrust is needed. An exact calculation can then be made for this condition.

					R TABLE and Cosines			
Angle,	Sine	Cosine	Angle,	Sine	Cosine	Angle,	Sine	Cosine
Degrees	(sin)	(cos)	Degrees	(sin)	(cos)	Degrees	(sin)	(cos)
1	0.0175	0.9998	31	0.5150	0.8572	61	0.8746	0.4848
2	0.0349	0.9994	32	0.5299	0.8480	62	0.8829	0.4695
3	0.0523	0.9986	33	0.5446	0.8387	63	0.8910	0.4540
4	0.0698	0.9976	34	0.5592	0.8290	64	0.8988	0.4384
5	0.0872	0.9962	35	0.5736	0.8192	65	0.9063	0.4226
6	0.1045	0.9945	36	0.5878	0.8090	66	0.9135	0.4067
7	0.1219	0.9925	37	0.6018	0.7986	67	0.9205	0.3907
8	0.1392	0.9903	38	0.6157	0.7880	68	0.9272	0.3746
9	0.1564	0.9877	39	0.6293	0.7771	69	0.9336	0.3584
10	0.1736	0.9848	40	0.6428	0.7660	70	0.9397	0.3420
11	0.1908	0.9816	41	0.6561	0.7547	71	0.9455	0.3256
12	0.2079	0.9781	42	0.6691	0.7431	72	0.9511	0.3090
13	0.2250	0.9744	43	0.6820	0.7314	73	0.9563	0.2924
14	0.2419	0.9703	44	0.6947	0.7193	74	0.9613	0.2756
15	0.2588	0.9659	45	0.7071	0.7071	75	0.9659	0.2588
16	0.2756	0.9613	46	0.7193	0.6947	76	0.9703	0.2419
17	0.2924	0.9563	47	0.7314	0.6820	77	0.9744	0.2250
18	0.3090	0.9511	48	0.7431	0.6691	78	0.9781	0.2079
19	0.3256	0.9455	49	0.7547	0.6561	79	0.9816	0.1908
20	0.3420	0.9397	50	0.7660	0.6428	80	0.9848	0.1736
21	0.3584	0.9336	51	0.7771	0.6293	81	0.9877	0.1564
22	0.3746	0.9272	52	0.7880	0.6157	82	0.9903	0.1392
23	0.3907	0.9205	53	0.7986	0.6018	83	0.9925	0.1219
24	0.4067	0.9135	54	0.8090	0.5878	84	0.9945	0.1045
25	0.4226	0.9063	55	0.8192	0.5736	85	0.9962	0.0872
26	0.4384	0.8988	56	0.8290	0.5592	86	0.9976	0.0698
27	0.4540	0.8910	57	0.8387	0.5446	87	0.9986	0.0523
28	0.4695	0.8829	58	0.8480	0.5299	88	0.9994	0.0349
29	0.4848	0.8746	59	0.8572	0.5150	89	0.9998	0.0175
30	0.5000	0.8660	60	0.8660	0.5000	90	1	0

#### Cylinders on Cranes and Beams:

Example 1: Calculation to find cylinder force required to handle 15,000 lbs. when the beam is in the position shown.

First find the force F2 at right angles to the beam which must be present to support the 15,000 lb. load.

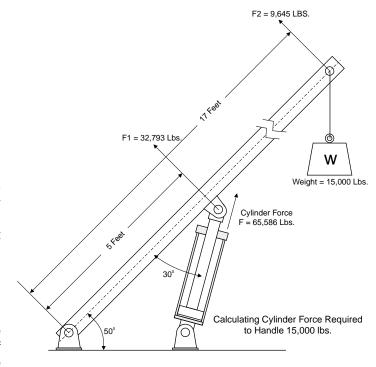
$$F2 = W \times \cos 50^{\circ} = 15,000 \times .643 = 9,645 \text{ lbs.}$$

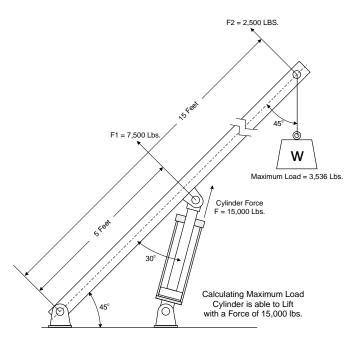
Next, find the force F1, also at right angles to the beam, which must be produced by the cylinder to support the 15,000 lb. load. This is calculated by proportion. F1 will be greater than F2 in the same ratio that arm lenght 17 feet is greater than arm lenght 5 feet.

Arm length ratio of  $17 \div 5 = 3.4$ . Therefore, F1 = 9,645 x 3.4 = 32,793 lbs.

Finally, calculate the cylinder force, at an angle of 30° to the beam, which will produce a force of 32,793 lbs. at its rod hinge point at right angles to the beam.

F (cylinder force) = F1  $\div$  sin 30° = 32,793  $\div$  .500 = 65,586 lbs.





Example 2: Calculation to find maximum load that can be lifted with a cylinder force of 15,000 lbs. when the beam is in the position shown.

First, translate the cylinder thrust, F, of 15,000 lbs. into 7,500 lbs. at right angles to the beam using power factor of 0.500 (sin) from the power factor table, for a 30° angle.

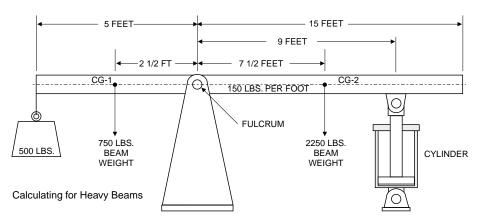
Next, translate this to F2, 2,500 lbs. at the end of beam where the weight is suspended. This is done with simple proportion by the length of each arm from the base pivot point. F2 is 1/3rd F1 since the lever arm is 3 times as long.

Finally, find the maximum hanging load that can be lifted, at a 45° angle between beam and load weight, using sin (power factor) for 45°:

W = F2 
$$\div$$
 sin 45° = 2500  $\div$  0.707 = 3535 lbs.

#### Calculations for a Heavy Beam:

On a heavy beam it is necessary to calculate not only for concentrated loads such as the suspended weights and cylinder thrust, but to figure in the weight of the beam itself. If the beam is uniform, so many pounds per foot of length, the calculation is relatively easy. In the example shown in figure "B", the beam has a uniform weight of 150 lbs.



per foot, is partially counterbalanced by a weight of 500 lbs. on the left side of the fulcrum, and must be raised by the force of a cylinder applied at a point 9 feet from the right side of the fulcrum.

The best method of solution is to use the principle of moments. A moment is a torque force consisting of (so many) pounds applied at a lever distance of (so many) feet or inches. The solution here is to find how much cylinder thrust is needed to just balance the beam. Then, by increasing the hydraulic cylinder

thrust 5 to 10% to take care of friction losses, the cylinder would be able to raise the beam.

Using the principle of moments, it is necessary to calculate all of the moment forces which are trying to turn the beam clockwise, then calculate all the moment forces trying to turn the beam counter-clockwise, then subtract the two. In this case they must be equal to balance the beam.

Clockwise moment due to the 15 feet of beam on the right side of the fulcrum: This can be considered as a concentrated weight acting at its center of gravity 7 1/2 feet from the fulcrum. Moment = 150 (lbs. per foot) x 15 feet x 7 1/2 feet = 16,875 foot pounds.

Counter-clockwise moment due to the 5 feet of beam on the left side of the fulcrum: 150 (lbs. per foot) x 5 feet x 2 1/2 feet (CG distance) = 1875 foot pounds.

Counter-clockwise moment due to hanging weight of 500 pounds: 500 x 5 feet = 2500 foot lbs.

Subtracting counter-clockwise from clockwise moments: 16,875 - 1875 - 2500 = 12,500 foot pounds that must be supplied by the cylinder for balance condition. To find cylinder thrust: 12,500 foot pounds  $\div$  9 feet (distance from fulcrum) = 1388.8 pounds.

Remember when working with moments, that only the portion of the total force which is at right angles to the beam is effective as a moment force. If the beam is at an angle to the cylinder or to the horizontal, then the effective portion of the concentrated of distributed weight, and the cylinder thrust, can be calculated with the power factors (refer to chart).

#### Designing With Cylinders Telescopic Cylinders

The great advantage telescopic cylinders have over conventional rod-type cylinders is their ability to provide an exceptionally long stroke from a compact initial package. The collapsed length of typical telescopic cylinders varies between 20% to 40% of their extended length. Thus, when mounting space is limited and the application needs a long stroke, a telescopic cylinder is a natural solution.

For example, a dump body needs to be tilted 60 degrees in order to empty completely. If the body or trailer is fitted with a conventional rod-type cylinder - with a one-piece barrel and stroke long enough to attain that angle - the dump body could not return to a horizontal orientation for highway travel because of the cylinder's length, even when fully retracted. A telescopic cylinder easily solves this problem.

Telescopic hydraulic cylinders are relatively simple devices, but their successful application requires an understanding of this component's idiosyncrasies. Knowledge of how telescopic cylinders work and which special application criteria to consider will enable you to design them safely and economically into equipment.

#### Main and Stages

As the name infers, Telescopic cylinders are constructed like a telescope. Sections of DOM (drawn over mandrel) steel tubing with successively smaller diameters nest inside each other. The largest diameter section is called the *main* or *barrel;* the smaller-diameter sections that move are called *stages;* The smallest stage is also called the *plunger*. The maximum practical number of moving stages seems to be six. Theoretically, cylinders with more stages could be designed but their stability problem would be daunting.

Telescopic cylinders normally extend from the largest stage to the smallest. This means the largest stage - with all the smaller stages nested inside it - will move first and complete its stroke before the next stage begins to move. This procedure will continue for each stage until the smallest-diameter stage is fully extended. Conversely, when retracting, the smallest-diameter stage will retract fully before the next stage starts to move. This continues until all stages are nested back in the main.

#### **Basic Cylinder Types**

As with conventional cylinders, the two basic types of telescopic hydraulic cylinders are *single-* and *double-acting*.

Single-acting telescoping cylinders extend under hydraulic pressure and rely on gravity or some external mechanical force for retraction. Single-acting cylinders are used in applications where some form of load is always on the cylinders. The classic single-acting telescopic applications are dump trucks and dump trailers. Pressurized oil extends the telescopic cylinder to raise one end of the dump body and expel its load. When pressure is released, the weight of the dump body forces oil out of the cylinder and it retracts.

Double-acting telescopic cylinders are powered hydraulically in both directions. They can be used in applications where neither gravity nor external force is available. They are well suited to noncritical positioning applications requiring out-and-back movement of a substantial load. A classic application is the packer-ejector cylinder in refuse vehicles and transfer trailers. The horizontally mounted cylinder pushes a platen to compress the load, then must retract with the platen so more material can be added. Gravity cannot help, so a double-acting cylinder is used.

#### **Bearings and Seals**

Each stage is supported within each successively larger stage by at least two bearings. One is at the bottom outside diameter or *piston end* of the stage, and the other is at the top internal diameter or *packing section* of the next larger stage. The distance between these two bearings determines the degree by which one stage overlaps the next. Generally, this distance or overlap must increase as overall stroke increases in order to resist deflection caused by the weight of extended stages and the load.

There are several designs for sealing telescopic cylinders. One of the most common designs for sealing telescopic cylinders is the use of several hinged chevron vee seals and / or one-piece, multi-lip seals with hinged lips molded in place. These seals are held in place by a stop ring or snap ring and packing nut and they use guide bearings on the sleeve piston. The internal diameter "ID" of each stage is sealed against the outer diameter "OD" of the next smaller stage nested inside it. The style and placement of these seals varies among cylinder manufactures. The style of seal also depends on its particular function. Zero-leakage, multiple-lip soft seals are usually found in the internal diameter at the packing section of the main and moving stages. Low-leakage hard seals are found on the piston end of double-acting telescopic cylinders. These piston seals allow the cylinder to retract under pressure.

Another design used on some single-acting telescopic cylinders, is the use of soft, zero leakage seals on the piston, which in turn use the full bore of the next larger stage as the effective area for extend force. These same seals contain the oil in the cylinder. The upper end of the cylinder, where the soft seals normally would be found, now contains a bearing for guidance. If any type of seal is used in the upper end of this telescopic cylinder design, it is usually a wiper/seal combination to exclude contaminants from entering the cylinders. With either type, the many sealing surfaces must compensate for normal deflection of stages as the cylinder extends.



## Designing With Cylinders Telescopic Cylinders

The cylinder design with the bearing on the piston and the seal on the other end is called a *displacement-type cylinder*. The single-acting design with a seal on the piston and a bearing at what normally would be the packing end approaches the classification of ram-type cylinder. Performance is similar to a double-acting rod-type cylinder with pressurized oil being supplied only to the piston side. All the telescopic stages would stroke in this way.

## **Double-Acting Telescopic Cylinders**

Normally extension of a double-acting telescopic cylinder occurs in the same manner as with the single-acting type. Retraction of double-acting telescopic cylinders is made possible by sealing each moving stage's piston area outside diameter with the next larger stage's inside diameter and building internal oil-transfer holes into each moving stage. The oil-transfer holes are located just above the pistons in the body of the stage. The retraction port normally is located in the top of the smallest stage. Oil flows through this port and into the smallest stage. The oil-transfer hole allows oil to enter and pressurize the volume between the next stage's internal diameter and the smaller stage's outer diameter. Pressure in this volume generates the force to move or retract the smaller stage into the larger stage.

Once this stage is fully retracted, the oil-transfer hole in the next larger stage is exposed to allow oil flow for it to retract. This retraction process continues automatically until all stages have retracted into the main. The seal on each stage selects the areas against which pressure will work.

Locating the retract port on the top of the smallest stage is the simplest way to design a double-acting telescopic cylinder, but this port location typically requires an arrangement of hoses, hose protection, and hose reels to deliver oil to the moving stage. To avoid having fluid power ports spaced far apart when the cylinder is fully extended, most double-acting telescopic cylinder designs locate both fluid ports in the smallest stage or plunger. The cylinder is then mounted so that the smallest stage or plunger is stationary and the larger and heavier stages would be the ones that move as the cylinder extends.

In some instances a double-acting telescoping cylinder can be designed where both ports are located in the stationary main barrel. Cylinder size (diameter and stroke) and the number of moving stages determine whether this is possible. If it is, the more-complicated internal passages for oil flow require a double wall and or a special trombone type telescopic design.

Piston seals on double-acting telescoping cylinders are normally manufactured from a hard substance such as cast iron, ductile iron or glass-reinforced nylon. The hard seals are needed to limit abrasion between the oil transfer holes and ports over which they must pass.

## Single- and Double-Acting Combinations

There are a few unusual types of telescoping cylinders designed for specific applications. For example, a manufacturer of oil well equipment uses a type composed of both single- and double-acting stages to position a work-over rig. The work-over rig is a derrick or tower that is transported horizontally to the well site on a trailer. There, telescopic cylinders extend to swing the rig into a vertical position. When the rig's work is done, the telescopic cylinder pulls the rig to begin the transition from vertical back to horizontal. However, once the rig has started to tilt, no more pull force is need because of the rig's weight and gravity will continue to retract the cylinder. In other words, the cylinder needs hydraulic power for the first part of its retraction stroke, but then operates as a single-acting unit.

In this type of design, the smallest moving stage is designed to be double-acting; the others are single-acting. The small stage can then provide push force to raise the rig, and pull force to start it back down. It is not unusual to design this type cylinder as a *skip-a-sleeve design*. Skip-a-sleeve design is as it's name implies, a sleeve or stage is skipped during design. Normally a telescopic stage diameter increases approximately every inch, example; sleeve diameter may be 3.75" fits into a 4.25" bore, 4.75" fitting into 5.25" bore, etc. In a skip-a-sleeve design, a sleeve is removed to increase the effective area and the retract force of the smallest sleeve or plunger, example; plunger diameter is 2.75" and fits into the 4.25" bore of the 4.75" sleeve, thus increasing effective area and retract force.

#### Constant-Thrust / Constant-Speed

A special telescopic cylinder - known as a *constant-trust/constant-speed cylinder* - is configured so that all moving stages will extend at the same time, providing an overall constant speed as well as a constant push force throughout its stroke when extending or retracting. This type of cylinder has been used to drive a drill head in underground mining, where such performance parameters are necessary and space is at a premium. The more-complicated design accomplishes the required action by trapping oil internally, matching extend and retract areas, and limiting the number of moving stages.



## Designing With Cylinders Telescopic Cylinders

## **Design Considerations**

Three familiar formulas determine the general operating characteristics of telescoping cylinders and can be manipulated to calculate the cylinder size required for a given cycle time or load. These formulas are:

 $F = A \times P$  S = 19.2 Q/A T = V/231Q where: where:

F - force, lb S - speed, fpm T - cycle time, min

A - area, in<sup>2</sup> Q - flow rate, gpm V - cylinder volume (area X stroke), in<sup>3</sup>

P - operating pressure, psi

The basic formulas for force, speed, and cycle time that apply to conventional rod-type cylinders also can be used with telescopic cylinders. To successfully apply these formulas, the designer must know which of the multiple areas and diameters to use. To calculate the force of any stage, you must decide which area will be substituted into the formulas. This area is determined by the placement of the seals that describe the boundaries of the area. For example: the extend area of a double-acting stage is determined by the seals on the pistons. Thus, the appropriate area would be calculated from the internal diameter of the next larger stage. On retraction, the area of any double-acting stage is the difference between that stage's outside diameter and the inside diameter of the next larger stage.

Designers must remember that the extend area for each stage is different, so the extend force for each stage also is different. The differences in areas mean that in an application with a constant-displacement pump supplying the hydraulic system, each stage will move at a different speed. This speed difference for each stage also holds true during retraction of double-acting telescopic cylinders because each stage's retract area is different.

In both types of telescopic cylinders, the smallest stage determines the force capacity of the cylinder. This stage will usually have the smallest extend and retract area. During extension, this stage will generate the cylinder's minimum force; during retraction, this stage normally generates the maximum force. A double-acting telescopic cylinder can exert no more retraction force than the smallest retract area provides.

After determining the effective diameter of each stage, volume can be approximated by dividing stroke by the number of stages and multiplying the quotient by each effective area. The sum of these volumes equals the approximate volume of oil to extend the cylinder. Reservoir volume should equal the cylinder's extended volume plus an initial volume of oil to fill the fully retracted cylinder and an adequate reserve for make-up oil.

Pump capacity is determined by applying the formula for speed to solve for Q (flow rate, gpm) in each stage. Inlet porting at the cylinder must be sized to accommodate the required flow for a given extension speed, of course.

## **Special Design Considerations**

Designers should *never* treat the telescopic cylinders as structural members. These cylinders should be used to generate work forces - not to stabilize the structure. They should be considered no more rigid than the columns of oil they contain. Telescopic cylinders always should be provided with mechanical support members.

Fully extended, long stroke telescopic cylinders can become very long, slender columns, making them susceptible to buckling. The structure of a telescopic cylinder can be considered as special as a stepped column with different diameter elements, each having a different moment of inertia. Additional overlap can help stabilize such a cylinder, but more overlap increases collapsed length as well as overall column length. Sometimes a cylinder diameter larger than required for the load may be needed to keep the cylinder safe under column loading.

As stated earlier, single-acting telescopic cylinders are extended by pressure and retracted by gravity or an external force. The extend speed is determined by the pump flow and flow capacity of the control valve. The retract speed is a function of the load on the cylinder and the ability of the hydraulic fluid to return to tank. Retraction speed can be controlled by metering return-oil flow through the control valve. Light loads and restricted flow slow down the retraction stroke. Most single-acting telescopic cylinders will not retract under their own weight. This is a result of several variables, including friction of the internal seals, attitude of the cylinder, and the type of mounting. A rigid mount with a low attitude may cause enough binding so that light loads cannot force the cylinder to retract.

As with any type of cylinder, heavy side loads should be avoided. Because of telescopic cylinder's multiple moving stages, side loading can cause internal binding that could result in mis-staging and possible stalling of the cylinder's movement. Because the overlap of each successive stage must be designed and manufactured with running and machining tolerances, these areas can act like hinges, allowing some movement. Longer overlap helps limit this movement, but cannot eliminate it. This is a Catch 22 design situation: the longer the overlap, the longer the cylinder's collapsed length.



## Designing With Cylinders Telescopic Cylinders

### Flow, Pressure Control

A three-way, three-position valve can provide raise, lower and hold control for a single-acting cylinder. Retraction speed of single-acting cylinders may be controlled by manually metering flow through the valve's return port. As an alternative, some systems use an orifice in the return line, valve, or cylinder port that is sized to limit flow and, thus, limit retraction speed.

Four-way, three-position valving is needed to perform the same control functions on double-acting types. The additional pathway provides a route to tank for oil displaced from the plunger end.

### **Dealing with Intensification**

Due to its construction, double-acting telescopic cylinders will act as pressure intensifiers during extension and flow multipliers during retraction. These two phenomenon are directly related to the large difference in effective area between the extend and retract side of each stage piston. This ratio can be as high as 10:1, or even greater. During extension of a double-acting telescopic cylinder, hydraulic oil is pumped into the extend port and exhausted out the retract port. If exhaust flow is impeded or restricted, the retract side of the cylinder can be pressurized to a level equal to the extend pressure multiplied by the differential area ratio. A dead block of exhaust flow can produce pressures high enough to destroy the cylinder. If any type of holding or check valve is installed in the retract line or on the retract port, the pressure intensification phenomenon can become dangerous. In the case of a 10:1 stage, a 2000 psi main pressure would result in an intermediate plunger pressure of 20,000 psi if flow from plunger is dead blocked. A similar, though less hazardous condition often results when the plunger side outlet line is reduced for design reasons or as the result of clogging or misconnection. The circuit must be designed so that these valves open before (or simultaneously with) the application of extend pressure to the cylinder.

When a double-acting cylinder retracts, the opposite occurs. Oil is pumped into the retract port and exhausted through the extend port. The exhaust flow will be equal to the retract flow multiplied by the differential area ratio. With a 10:1 ratio, a 20-gpm retract flow becomes a 200-gpm exhaust flow. If the extend lines or valves are too small and flow is restricted, backpressure can occur in the cylinder to slow the retract speed. If the backpressure equals the pump's retract pressure, the cylinder will stall and not retract.

Telescopic cylinder manufacturers attempt to size the ports to eliminate or reduce the potential for this phenomenon, but designers should size other components in the hydraulic circuit with this in mind. Most problems relating to these phenomenon result from increasing pump flow or downsizing lines, connectors, or control valves after the cylinder has been specified for operation with larger components.

#### Seal Bypass

Piston seals in double-acting telescopic cylinders normally are manufactured from a hard substance, such as cast iron, ductile iron, or glass reinforced nylon. Hard seals are needed to resist abrasion when the seals slide across the transfer holes. However, these seals are not as efficient as soft urethane or rubber seals, so small amounts of oil can bypass them. This bypass flow actually can cause a cylinder to stall if pump flow is less than the seal's allowable leakage rate. This may become a problem if the cylinder is required to stroke at low speeds. Consequently, loading should be limited to a level slightly below the cylinder's rated force at a given pressure.

Bypass leakage also can allow a cylinder to drift in either direction while holding a load. Drift is extremely hazardous if the cylinder is holding a load on the retract area. If a piston drifts past the internal transfer holes in a stage, the retract oil will rapidly transfer to the extend area - causing the cylinder to extend abruptly. This is possible because the retract oil volume is less than the extend volume, due to the large differential area ratio. Therefore, a double-acting telescoping cylinder should not be expected to hold a load on retraction.

## **Summary**

It should now be evident that specifying telescoping cylinders requires knowledge beyond that of conventional cylinders. The best insurance to guard against unforeseen problems — especially for those lacking familiarity with telescoping cylinders — is to draw from the experience of manufacturer's application engineers.

Manufacturer's of telescopic cylinders can (and have) altered their designs to suit a variety of special application considerations. Their application engineers should be eager to provide assistance in selecting or designing the right cylinder for your specific application, and advising about circuitry to operate it safely and efficiently.



#### CYLINDER FORMULAS

Thrust or force of any cylinder:

 $F = A \times P$ 

 $P = F \div A$ 

 $A = F \div P$ 

F = Force or thrust, in pounds

A = Piston area in square inches (  $.7854 \times D^2$ )

P = PSI (Gauge pressure in pounds per square inch)

# HP = Pounds of push (or pull) x Distance (in feet) 550 x Time (in seconds)

HP = Horsepower

Circle Formula:

 $A = D \times D \times .7854$ 

 $A = D^2 \times 0.7854$ 

 $A = \pi \times R^2$ 

 $A = \pi \times D^2 \div 4$ 

Circumference =  $2 \times R \times \pi$ 

Circumference =  $\pi \times D$ 

 $D = \sqrt{A/.7854}$ 

A = Area in<sup>2</sup> (Area sq. in.)

R = Radius (1/2 of Diameter)

D = Diameter, inches

 $\pi = 3.14$ 

Hydraulic Cylinder Piston travel speed:

V1 (in/min) = CIM ÷ A

 $V2 (ft/min) = Q \times 19.25 \div A$ 

 $V3 (ft/sec) = Q \times 0.3208 \div A$ 

 $Q (GPM) = 3.117 \times V3 (ft/sec) \times A$ 

 $Q (GPM) = CIM \div 231$ 

V1 = Velocity or piston travel speed, inches per minute

V2 = Velocity or piston travel speed, feet per minute

V3 = Velocity or piston travel speed, feet per second

CIM = Flow rate in cubic inches per minute (in<sup>3</sup>)

A = Effective area in square inches (in²)

Q = GPM Gallons per minute

1 Gallon = 231 in<sup>3</sup> (cubic inch)

Volume required to move a piston a given distance:

 $V = A \times L$ 

V = Volume in cubic inches (in<sup>3</sup>)

A = Area in square inches (in<sup>2</sup>)

L = Length or stroke in inches

Regenerative Cylinder

Extend Speed = Rod Volume ÷ Flow Rate in<sup>3</sup>

Area to Retract = Area to extend - Rod Area

Cylinder Ratio = Area to extend + Area to retract

Note:

Ratio can be used to calculate pressure intensification and flow intensification.

Effective force of a cylinder working at an angle to direction of the load travel:

 $F = T \times sin A$ 

T = Total cylinder force, in pounds

F = Part of the force which is effective, in pounds

A = Least angle, in degrees, between cylinder axis and load direction.

Moment Arm Equations / Levers:

 $F \times Df = W \times Dw$ 

 $F = W \times Dw \div Df^{w}$ 

 $W = F \times Df \div Dw$ 

 $Df = W \div F \times Dw$ 

 $Dw = F \div W \times Df$ 

F = Cylinder force

Df = Cylinder force distance to pivot

W = Weight or Load Force

Dw = Weight or Load Force distance to pivot

Toggle Force:

 $T = F \times A \div 2 \times B$ 

T = Toggle Force

F = Cylinder Force

A = Distance cylinder centerline to toggle

B = Remaining stroke

Force for piercing or shearing sheet metal:

 $F = P \times T \times PSI$ 

F = Force required, in pounds

P = Perimeter around area to be sheared, in inches

T = Sheet thickness in inches

PSI = Sheer strength rating of the material in pounds per square inch.

P.O. Check Application:

## Release PSI = Cap End Area x Max. W.P. - Load Rod End Area

Max. W.P. = Pressure Rating of Components

Ratio = <u>Max Working PSI</u> Release PSI

Example;

2 to 1 Ratio = 1 square inch (in²) at 1000 psi working pressure will open when a Release pressure of 500 psi is applied to a 2 square inches (in²) area.



#### HYDRAULIC PUMP EQUATIONS

Horsepower Required to Drive Hydraulic Pump:

HP = PSI x GPM  $\div$  1714 HP = (PSI x GPM)  $\div$  (1714 x EFFICIENCY)

HP = Horsepower

PSI = Gauge pressure in pounds per square inch GPM = Oil flow in gallons per minute

EFFICIENCY = Efficiency of hydraulic pump

#### Important:

As all systems are less than 10% efficient and efficiency factor must be added to the calculated input horsepower.

### Example:

Input hp = 10 gpm x 1500 psi  $\div$  1714 (constant) = 8.75 hp x 0.85 (efficiency) = required input 10 hp

### Rule of thumb:

For every 1 HP of drive, the equivalent of 1 GPM @ 1500 PSI can be produced.

#### Rule of thumb:

To idle a pump when it is unloaded will require about 5% of its full rated horsepower.

#### Note:

1 hp = 33,000 ft lbs per min or 33,000 lbs raised 1 ft in 1 min

1 hp = 550 ft. lbs. per second

1 hp = 746 Watts or 0.746 kw

1 hp = 42.4 Btu per min

1 hp = 2545 Btu per hour

BTU = The energy to raise one pound of water one degree Fahrenheit.

### Flow Formulas:

## GPM (theoretical) = RPM x CIR ÷ 231

GPM = Oil flow in gallons per minute CIR = Cubic Inch (in³) per Revolution RPM = Pump revolutions per minute

Volume required (gpm) =  $\frac{\text{Volume Displaced x 60}}{\text{Time (s) x 231}}$ 

Flow rate (gpm) =  $\frac{\text{Velocity (ft/s)} \times \text{Area (in}^2)}{0.3208}$ 

#### Note:

Fluid is pushed or drawn into a pump

Pumps do not pump pressure, their purpose is to create flow. (Pressure is a result of resistance to flow).

## Torque and horsepower relations:

T = HP x 63025 ÷ RPM HP = T x RPM ÷ 63025 RPM = HP x 63025 ÷ T

T = Torque, inch-lbs RPM = Speed, revs / minute

HP = Horsepower

#### Note:

For Torque in foot-lbs use 5252 in place of 63025

#### Note.

Work (in lbs) = force (lbs) x distance (in)

Power = Force x Distance ÷ Time

## Theoretical Pressure = $T \times 6.28 \div CIR$

T = Torque, inch-lbs CIR = Cubic Inch (in³) per Revolution

Pump Efficiencies:

## Volumetric Efficiency = <u>Actual GPM x 100</u> Theoretical Flow

Mechanical Efficiency = <u>Actual PSI x 100</u> Theoretical Pressure

Overall Efficiency = Output HP x 100
Input HP

## Overall Efficiency = Mech. Eff. x Volumetric Eff.

Theoretical Flow = RPM x CIR  $\div$ Theoretical Pressure = T x 6.28  $\div$  CIR Input HP = PSI x GPM  $\div$ Output HP = T x RPM  $\div$ 

T = Torque, inch-lbs

CIR = Cubic Inch (in³) per Revolution GPM = Flow in gallons per minute

PSI = Gauge pressure in pounds per square inch

RPM = Pump revolutions per minute

## **Gear Displacement Calculation:**

The volumetric displacement of a gear pump or motor can be approximated by measurement of the internal parts and substituting the values in the following formula:

 $V = 6.03 \times W \times (2 \times D - L) \times (L - D \div 2)$ 

#### Where

V = displacement in in<sup>3</sup>/rev

W = gear width in inches

D = gear tip diameter in inches

L = dimension across both gears when meshed in inches



#### **HYDRAULIC MOTOR EQUATIONS**

*Note:* Hydraulic motors are typically classified as high speed motors (500 - 10,000 rpm) or low speed motors (0 - 1,000) rpm.

Relationship between displacement and torque of a hydraulic motor:

T = HP x 63025 ÷ RPM HP = T x RPM ÷ 63205 RPM = HP x 63025 ÷ T

Note:

For Torque in foot-lbs use 5252 in place of 63025

 $T = CIR \times PSI \div 6.28$   $CIR = T \div PSI \times 6.28$  $PSI = T \times 6.28 \div CIR$ 

T = (GPM x PSI x 36.77)  $\div$  6.28 GPM = (T  $\div$  PSI  $\div$  36.77) x 6.28 PSI = (T  $\div$  GPM  $\div$  36.77) x 6.28

Note:

Divide PSI by Mechanical Efficiency if required. For Torque in foot-lbs use 75.36 in place of 6.28

T = Torque, inch-lbs
CIR = Cubic Inch (in³) per Revolution
GPM = Flow in gallons per minute
PSI = Pressure difference across motor
RPM = Pump revolutions per minute

Torque General Info:

HP = Horsepower

Torque = Radius x Load

Torque (in lbs) = Lever Length (in.) x Pull (lbs.)

Radius = 1/2 of Diameter

Circumference = 3.14 x Diameter

Foot Pound = Inch Pound ÷ 12

Inch Pound = Foot Pound x 12

Motor Speed:

 $GPM = RPM \times CID \div 231$   $RPM = GPM \times 231 \div CID$  $CID = GPM \div RPM \times 231$ 

Speed = (336 x MPH) ÷ Wheel Diameter (in.)

Side load on pump or motor shaft:

 $F = (HP \times 63024) \div (RPM \times R)$ 

F = Side load, in pounds, against shaft

R = Pitch radius of sheave on pump shaft, in inches;

HP = Driving power applied to shaft.

Motor Efficiencies:

Volumetric Efficiency =  $\frac{\text{Actual Speed x 100}}{\text{Theoretical Speed}}$ 

Mechanical Efficiency = <u>Actual Torque x 100</u> Theoretical Torque

Overall Efficiency = <u>Output HP x 100</u> Input HP

Overall Efficiency = Mech. Eff. x Volumetric Eff.

Theoretical Speed = GPM x 231  $\div$  CIR Theoretical Torque (in lbs) = CIR x PSI  $\div$  6.28 Input HP = PSI x GPM  $\div$  1714 Output HP = T x RPM  $\div$  63025

T = Torque, inch-lbs
CIR = Cubic Inch (in³) per Revolution
GPM = Flow in gallons per minute
PSI = Pressure difference across motor
RPM = Pump revolutions per minute

Note:

For Torque in foot-lbs use 5252 in place of 63025

Draw Bar Pull, Moving a load up an incline:

 $F = L \times sin$ 

F = Force W = Weight or load

sin = Sin of incline or angle

Rule of thumb:

Grades less than or equal to 10° use the degree of the angle. Grades greater than 10° use sin.

Grade (% of Slope) = Rise ÷ Run

Draw Bar Pull, Friction:

 $F = W \times M$ 

F = Force

W = Weight or load

M = Coefficient of friction

Draw Bar Pull, Moving a load up an incline with friction:

F to move load =  $(W \times sin) + (W \times cos \times M)$ F to hold load =  $(W \times sin) - (W \times cos \times M)$ 

F = Force

W = Weight or load

M = Coefficient of friction sin = Sin of incline or angle

cos = Cosine of incline or angle



## Formulas

Velocity of oil flow in pipe:

 $V = GPM \times 0.3208 \div A$  $A = GPM \times 0.3208 \div V$  $GPM = A \times V \div 0.3208$ 

V = Oil velocity in feet per second GPM = Flow in gallons per minute A = Inside area of pipe in square inches.

Rule of thumb:

Pump suction lines 2 to 4 feet/second Pressure lines up to 500 PSI - 10 to 15 fps Pressure lines 500 to 3000 PSI - 15 to 20 fps Pressure lines over 3000 PSI - 25 fps All oil lines in air-over-oil system - 4 fps fps = feet per second

Barlow formula (hoop stress):

 $P = 2 \times t \times S \div D$ 

P = Working pressure in PSI with a 4:1 Design Factor t = Wall thickness, in inches

S = Allowable stress (12,500 with a 4:1 Design Factor)

D = Outside diameter, in inches.

 $D = \sqrt{A/.7854}$ 

Atmosphere:

Atmospheric pressure is 14.7 psi at sea level One Bar is equal to 14.5 psi (Atmos. - 1.01 Bar) The pressure created by one fooot of water is .433 psi

Atmospheric Ratio =  $14.7 \div PSI = 33.9 \div (X)$ 

Atmospheric will lift water 33.9 feet 1 inch Hg = .491 psi14.7 psi = 29.92 hgY inch Hg Absolute =  $(29.92 - Y) \times .491 = PSI$  $PSI = lbs \div in^2$ Hg = Inches of mercury

Filtration:

1 Micron = .000039"149 Micron = 100 Mesh 74 Micron = 200 Mesh 44 Micron = 325 Mesh Beta 75 = 98.7%Beta 100 = 99%

Beta 200 = 99.5%Gas

Beta Ratio = Upstream Count + Downstream Count

Efficiency Percent (%) = 1 - (1 ÷ Beta Ratio) x 100

Gas Formulas:

PSIG (PSI Gage) = PSIA - 14.7 PSIA (PSI Absolute) = PSIG + 14.7

Isothermal

 $P_1 \times V_1 = P_2 \times V_2$ 

P<sub>1</sub> = Pre-charge Pressure + 14.7

V<sub>1</sub> = Intial Gas Volume

P<sub>2</sub> = System Pressure + 14,7

V<sub>2</sub> = Compressed Gas Volume

 $P_1$ ,  $V_1$  are initial pressure and volume;  $P_2$  and  $V_2$  are final conditions.

Note:

Isothermal operatiion occurs when compression or expansion is slow enough to allow transfer of heat out of or into the accumulator.

Adiabatic

 $P_1 \times V_1 \times T_2 = P_2 \times V_2 \times T_1$   $P_1 \times V_1 \div T_1 = P_2 \times V_2 \div T_2$ 

P<sub>1</sub> = Pre-charge Pressure + 14.7

V<sub>1</sub> = Intial Gas Volume

P<sub>2</sub> = System Pressure + 14.7

V<sub>2</sub> = Compressed Gas Volume

 $T_1 = Initial Temp. Absolute (Rankine)$ 

T<sub>2</sub> = Increased Temp. Absolute (Rankine)

T<sub>4</sub>, P<sub>4</sub> and V<sub>4</sub> are initial temperature, pressure and volume and, T<sub>2</sub>, P<sub>2</sub> and V<sub>2</sub> are final conditions.

Note:

Adiabatic operatiion occurs when compression or expansion is rapid so that there is no transfer of heat. The adiabatic equation is used where compression or expansion occurs in less than 1 minute.

Rule of thumb:

Compressibility of hydraulic oil: Volume reduction is approximately 0.5% for every 1000 PSI pressure. Compressibility of water: Volume reduction is about 0.3% for every 1000 PSI pressure.

Rankine = Fahrenheit + 460 Kelvin = Celsius + 278

Celsius to Fahrenheit = (C + 17.78) x 1.8 = Fahrenheit Fahrenheit to Celsius = F - 32 ÷ 1.8 = Celsius

Intial Gas Volume - Compressed Gas = Usual Oil

## Formulas

### Reservoir Cooling:

HP Radiated = Sq. Ft. x TD ÷ 1000 Sq. Ft. = HP x 1000 ÷ TD TD = HP x 1000 ÷ Sq. Ft.

HP = Power radiating capacity expressed in horsepower Sq. Ft. = Surface area, in square feet

TD = Temperature difference (Delta) in °F between oil and surrounding air.

If the tank is half full, divide the answer by 2. If the tank is stainless steel (CRES), divide the answer by 2. If the tank is aluminum, multiply the answer by 2.8.

1 HP = 2545 BTU 1 HP = 746 Watts BTU = the energy to raise one pound of water one degree Farenheit

#### Rule of thumb:

Each watt will raise the temperature of 1 gallon of oil by 1 °F per hour.

### Reservoir Heating:

BTU's to heat a reservoir = Oil volume (ft³) x 62.4 Specific Heat (.5) x Specific Gravity (.89) Temp. Delta (Differential)

BTU  $\div$  2545 = HP per Hour HP x 746 = Watts

#### Note:

The following applies to petroleum based hydraulic fluids.

Hydraulic oil serves as a lubricant and is practically non-compressible. It will compress approximately 0.5% at 1000 psi.

The weight of hydraulic oil may vary with a change in viscosity, however, 55 to 58 lbs/ft<sup>3</sup> covers the viscosity range from 150 SUS to 900 SUS @ 100 degrees F.

Pressure at the bottom of a one foot column of oil will be approximately 0.4 psi.

To find the pressure at the bottom of any column of oil, multiply the height in feet by 0.4.

Atmospheric pressure equals 14.7 psia at sea level.

psia (pounds per square inch absolute).

Gauge readings to not include atmospheric pressure unless marked psia.

## **Energy Formulas:**

1 Kw = 1.3 hp 1 hp = 550 ft lbs/s Hydraulic hp = gpm x psi  $\div$  1714 Torque (in lbs) = psi x disp. (in³/rev)  $\div$  6.28 Torque (in lbs) = hp x 63025  $\div$  Rpm hp = Torque (ft lbs) x rpm  $\div$  5252 Btu (per hour) =  $\Delta$ psi x gpm x 1.5



## Formulae in SI Metric Units

Familiar fluid power formulae in English units are shown in the left column. When the industry converts to SI (International) units, these formulae will take the form shown in the right column.

#### **English Units**

### **Metric Units**

## Torque, HP, Speed Relations in Hydraulic Pumps and Motors

 $T = HP \times 5252 \div RPM$  $HP = T \times RPM \div 5252$  $RPM = HP \times 5252 \div T$ T = Torque, foot-lbs. RPM = Speed, revs/min

 $HP = PSI \times GPM \div 1714$ 

HP = Horsepower

 $RPM = Kw \times 9543 \div T$ T = Torque, Nm (Newton-meters) RPM = Speed, revs/min Kw = Power in kilowatts

 $T = Kw \times 9543 \div RPM$ 

 $Kw = T \times RPM \div 9543$ 

## **Hydraulic Power Flowing Through the Pipes**

HP = Horsepower PSI = Gauge pressure, lbs/sq. inch

GPM = Flow, gallons per minute

## $Kw = Bars x dm^3/min \div 600$

Kw = Powers in kilowatts Bars = System pressure dm<sup>3</sup>/min = Flow, cu. dm/minute

## Force Developed by an Air or Hydraulic Cylinder

 $T = A \times PSI$ 

T = Force or thrust, in lbs. A = Piston area, square inches PSI = Gauge pressure, lbs/sq. inch

### $N = A \times Bars \times 10$

N = Cylinder force in Newtons A = Piston area, sq. centimeters Bars = Gauge pressure

## **Travel Speed of a Hydraulic Cylinder Piston**

 $S = V \div A$ 

S = Travel speed, inches/minute V = Vol. of oil to cyl., cu.in/min A = Piston area, square inches

### $S = V \div 6A$

S = Travel speed, meter/sec V = Oil flow dm<sup>3</sup>/minute

A = Piston area, square centimeters

### Barlow's Formula - Burst Pressure of Pipe & Tubing

 $P = 2t \times S \div O$ 

P = Burst pressure, PSI t = Pipe wall thickness, inches S = Tensile str., pipe material, PSI O = Outside diameter of pipe, inches

## $P = 2t \times S \div O$

P = Burst pressure, bars t = Pipe wall thickness, mm S = Tensile str., pipe material, bars O = Outside diameter of pipe, mm

## **Velocity of Oil Flow in Hydraulic Lines**

 $V = GPM \times 0.3208 \div A$ 

V = Velocity, feet per second GPM = Oil flow, gallons/minute A = Inside area of pipe, sq. inches

#### $V = dm^3/min \div 6A$

V = Oil velocity, meters/second dm<sup>3</sup>/min = Oil flow, cu.dm/minute A = Inside area of pipe, sq.cm.

## Recommended Maximum Oil Velocity in Hydraulic Lines

fps = feet per second Pump suction lines - 2 to 4 fps Pres. lines to 500 PSI - 10 to 15 fps Pres. lines to 3000 PSI - 15 to 20 fps Pres. lines over 3000 PSI - 25 fps Oil lines in air/oil system - 4 fps

mps = meters per second Pump suction lines - .6 to 1.2 mps Pres. lines to 350 bar - 3 to 41/2 mps Pres. lines to 200 bar - 41/2 to 6 mps Pres. lines over 200 bar - 71/2 mps Oil lines in air/oil system - 11/4 mps

#### **LENGTH**

- 1 micron ( $\mu$ ) = 0.00004 inch (in.)
- 1 millimeter (mm) = 0.039 in.
- 1 centimeter (cm) = 0.3937 in.
- 1 decimeter (dm) = 0.3281 foot (ft.)
- 1 meter (m) = 39.37 in.
  - = 3.281 ft.
  - = 1.0937 yards (yds.)

### **AREA - SQUARE**

- 1 square millimeter = 0.00155 square inch (sq. in.)
- 1 square centimeter = 0.155 sq. in.
- 1 square decimeter = 15.5 sq. in.
  - = 0.10764 square feet (sq. ft.)

#### **AREA - CUBIC**

- 1 cubic centimeter = 0.061 cubic inch (in.3)
  - = 0.0002642 U.S. liquid gallons
- 1 cubic decimeter = 61.023 in.3

### LIQUID MEASURE

- 1 milliliter (ml) = 0.0338176 ounce (oz.)
- 1 deciliter (dl) = 3.381 oz.
- 1 liter (I) = 1.0569 quarts (qt.)
  - = 0.26417 gallon (gal.)
- 1 drop = 0.05 cubic centimeter (cc)
  - = 0.00169 oz.

#### WEIGHT

- 1 gram (g) = 0.0353 ounce (oz.)
- 1 kilogram (kg.) = 2.2046 pounds (lb.)
- 1 metric ton = 0.9842 U.S. ton

#### **TEMPERATURE**

°Celsius = 5/9 (°Fahrenheit - 32)

### **FLOW - LIQUID**

1 liter/minute (lpm) = 0.2642 U.S. gallon/minute (gpm)

### **FORCE**

1 Newton (N) = 0.225 pound (lb.)

## **FREQUENCY**

1 cycle/second (cps) = 1 Hertz (H)

## **ABSOLUTE VISCOSITY**

1 centipoise (@ 0.9 specific gravity) = 5.35 SUS

## **POWER**

- 1 kilowat (kw) = 1.34 horsepower (HP)
- 1 horsepower (HP) = 33,000 foot-pounds (ft. lbs.)/minute
  - = 550 foot-pounds (ft. lbs.)/second
  - = 42.4 BTU/minute
  - = 746 watts

### **PRESSURE**

- 1 bar = 14.5 pounds per square inch (psi) above atmospheric
  - = 33.8 foot water column
  - = 42 foot oil column
  - = 29.92 inches of mercury (in. Hg)
- 1 millimeter of mercury (mm Hg) = 0.03937 in. Hg below atmospheric
- 1 psi = 2.0416 in. Hg
  - = 27.71 in. water
- 1 foot column of water = 0.433 psi
- 1 foot column of oil = 0.390 psi

#### **TORQUE**

1 Newton-meter (Nm) = 8.88 pound-inches (lb.-in.)

## **VELOCITY**

1 meter per second (m/s) = 3.28 feet/second (fps)

## FRACTIONS, DECIMALS AND MILLIMETERS

Inches		LS AND WILLIME IERS	Inches
Inches Fractions Decimals M M	Inches Fractions Decimals M M	Inches Fractions Decimals M M	Inches Fractions Decimals M M
·			
0.0004 0.01	25/32 0.78125 19.844	2.165 55	3-11/16 3.6875 93.663
0.004 0.1	0.7874 20	2-3/16 2.1875 55.563	3.7008 94
0.01 0.25	51/64 0.79688 20.241	2.2047 56	3-23/32 3.719 94.456
1/64 0.01562 0.397	13/16 0.8125 20.638	2-7/32 2.219 56.356	3.7401 95
0.0197 0.5	0.8268 21	2.244 57	3-3/4 3.75 95.25
0.0295 0.75 1/32 0.03125 0.794	53/64 0.82812 21.034 27/32 0.84375 21.431	2-1/4 2.25 57.15 2-9/32 2.281 57.944	3.7795 96 3-25/32 3.781 96.044
0.0394 1	55/64 0.85938 21.828	2.2835 58	3-13/16 3.8125 96.838
3/64 0.04688 1.191	0.8661 22	2-5/16 2.312 58.738	3.8189 97
0.059 1.5	7/8 0.875 22.225	2.3228 59	3-27/32 3.844 97.631
1/16 0.0625 1.588	57/64 0.89062 22.622	2-11/32 2.344 59.531	3.8583 98
5/64 0.07812 1.984	0.9055 23	2.3622 60	3-7/8 3.875 98.425
0.0787 2	29/32 0.90625 23.019	2-3/8 2.375 60.325	3.8976 99
3/32 0.09375 2.381	59/64 0.92188 23.416	2.4016 61	3-29/32 3.9062 99.219
0.0984 2.5	15/16 0.9375 23.813	2-13/32 2.406 61.119	3.937 100
7/64 0.10938 2.778	0.9449 24	2-7/16 2.438 61.913	3-15/16 3.9375 100.013
0.1181 3	61/64 0.95312 24.209	2.4409 62	3-31/32 3.969 100.806
1/8 0.125 3.175	31/32 0.96875 24.606	2-15/16 2.469 62.706	3.9764 101
0.1378 3.5	0.9843 25	2.4803 63	4 4 101.6
9/64 0.14062 3.572	63/64 0.98438 25.003	2-1/2 2.5 63.5	4-1/16 4.062 103.188
5/32 0.15625 3.969	1 1 25.4	2.5197 64	4-1/8 4.125 104.775
0.1575 4	1.0236 26	2-17/32 2.531 64.294	4.1338 105
11/64 0.17188 4.366 0.177 4.5	1-1/32 1.0312 26.194 1-1/16 1.062 26.988	2.559 65 2-9/16 2.562 65.088	4-3/16 4.1875 106.363 4-1/4 4.25 107.95
3/16 0.1875 4.763	1.063 27	2-19/32 2.594 65.881	4-5/16 4.312 107.93
0.1969 5	1-3/32 1.094 27.781	2.5984 66	4.3307 110
13/64 0.20312 5.159	1.1024 25	2-5/8 2.625 66.675	4-3/8 4.375 111.125
0.2165 5.5	1-1/8 1.125 28.575	2.638 67	4-7/16 4.438 112.716
7/32 0.21875 5.556	1.1417 29	2-21/32 2.656 67.469	4-1/2 4.5 114.3
15/64 0.23438 5.953	1-5/32 1.156 29.369	2.6772 68	4.5275 115
0.2362 6	1.1811 30	2-11/16 2.6875 68.263	4-9/16 4.562 115.88
1/4 0.25 6.35	1-3/16 1.1875 30.163	2.7165 69	4-5/8 4.625 117.475
0.2559 6.5	1-7/32 1.219 30.956	2-23/32 2.719 69.056	4-11/16 4.6875 119.063
17/64 0.26562 6.747	1.2205 31	2-3/4 2.75 69.85	4.7244 120
0.2756 7	1-1/4 1.25 31.75	2.7559 70	4-3/4 4.75 120.65
9/32 0.28125 7.144	1.2598 32	2-25/32 2.781 70.643	4-13/16 4.8125 122.238
0.2953 7.5	1-9/32 1.281 32.544	2.7953 71	4-7/8 4.875 123.825 4.9212 125
19/64 0.29688 7.541 5/16 0.3125 7.938	1.2992 33 1-5/16 1.312 33.338	2-13/16 2.8125 71.437 2.8346 72	4-15/16 4.9375 125.413
0.315 8	1.3386 34	2-27/32 2.844 72.231	5 5 127
21/64 0.32812 8.334	1-11/32 1.344 34.131	2.874 73	5.1181 130
0.335 8.5	1-3/8 1.375 34.925	2-7/8 2.875 73.025	5-1/4 5.25 133.35
11/32 0.34375 8.731	1.3779 35	2-29/32 2.9062 73.819	5-1/2 5.5 139.7
0.3543 9	1-13/32 1.406 35.719	2.9134 74	5.5118 140
23/64 0.35938 9.128	1.4173 36	2-15/16 2.9375 74.613	5-3/4 5.75 146.05
0.374 9.5	1-7/16 1.438 36.513	2.9527 75	5.9055 150
3/8 0.375 9.525	1.4567 37	2-31/32 2.969 75.406	6 6 152.4
25/64 0.39062 9.922	1-15/32 1.469 37.306	2.9921 76	6-1/4 6.25 158.75
0.3937 10	1.4961 38	3 3	6.2992 160
13/32 0.40625 10.319 0.413 10.5	1-1/2 1.5 38.1 1-17/32 1.531 38.894	3-1/32 3.0312 76.994 3.0315 77	6-1/2 6.5 165.1 6.6929 170
27/64 0.42188 10.716	1.5354 39	3-1/16 3.062 77.788	6-3/4 6.75 171.45
0.4331 11	1-9/16 1.562 39.688	3.0709 78	7 7 177.8
7/16 0.4375 11.113	1.5748 40	3-3/32 3.094 75.581	7.0866 180
29/64 0.45312 11.509	1-19/32 1.594 40.481	3.1102 79	7.4803 190
15/32 0.46875 11.906	1.6142 41	3-1/8 3.125 79.375	7-1/2 7.5 190.5
0.4724 12	1-5/8 1.625 41.275	3.1495 80	7.874 200
31/64 0.48438 12.303	1.6535 42	3-5/32 3.156 80.169	8 8 203.2
0.492 12.5	1-21/32 1.6562 42.069	3-3/16 3.1875 80.963	8.2677 210
1/2 0.5 12.7	1-11/16 1.6875 42.863	3.189 81	8-1/2 8.5 215.9
0.5118 13	1.6929 43	3-7/32 3.219 81.756	8.6614 220
33/64 0.51562 13.097 17/32 0.53125 13.494	1-23/32 1.719 43.656 1.7323 44	3.2283 82 3-1/4 3.25 82.55	9 9
35/64 0.54688 13.891	1-3/4 1.75 44.45	3.2677 83	9.055 230
0.5512 14	1.7717 45	3-9/32 3.281 83.344	9-1/2 9.5 241.3
9/16 0.5625 14.288	1-25/32 1.781 45.244	3.3071 84	9.8425 250
0.571 14.5	1.811 46	3-5/16 3.312 84.137	10 10 254.01
37/64 0.57812 14.684	1-13/16 1.8125 46.038	3-11/32 3.344 84.931	10.2362 260
0.5906 15	1-27/32 1.844 46.831	3.3464 85	10.6299 270
19/32 0.59375 15.081	1.8504 47	3-3/8 3.375 85.725	11 11 279.401
39/64 0.60938 15.478	1-7/8 1.875 47.625	3.3858 86	11.0236 280
5/8 0.625 15.875	1.8898 48	3-13/32 3.406 86.519	11.4173 290
0.6299 16	1-29/32 1.9062 48.419 1.9291 49	3.4252 87	11.811 300
41/64 0.64062 16.272 0.6496 16.5	1-15/16 1.9375 49.213	3-7/16 3.438 87.313 3.4646 88	12 12 304.801 13 13 330.201
21/32 0.65625 16.669	1.9875 49.213	3-15/32 3.469 88.106	13 350.201
0.6693 17	1-31/32 1.969 50.006	3-1/2 3.5 88.9	14 14 335.601
43/64 0.67188 17.066	2 50.8	3.5039 89	15 15 381.001
11/16 0.6875 17.463	2.0079 51	3-17/32 3.531 89.694	15.748 400
45/64 0.70312 17.859	2-1/32 2.0312 51.594	3.5433 90	16 16 406.401
0.7087 18	2.0472 52	3-9/16 3.562 90.487	17 17 431.801
23/32 0.71875 18.256	2-1/16 2.062 52.388	3.5827 91	17.7165 450
0.7283 18.5	2.0866 53	3-19/32 3.594 91.281	18 18 457.201
47/64 0.73438 18.653	2-3/32 2.094 53.181	3.622 92	19 19 482.601
0.748 19	2-1/8 2.125 53.975	3-5/8 3.625 92.075	19.685 500
3/4 0.75 19.05	2.126 54 2-5/32 2.156 54.769	3-21/32 3.656 92.869	20 20 508.001
49/64 0.76562 19.447	2-5/32 2.156 54.769	3.6614 93	1

## **Conversion Factor Tables**

To convert	$\rightarrow$	$\rightarrow$	$\rightarrow$	$\rightarrow$	Into →	$\rightarrow$	$\rightarrow$	$\rightarrow$	$\rightarrow$	Multiply by
Into ←	$\leftarrow$	$\leftarrow$	$\leftarrow$	$\leftarrow$	To convert	$\leftarrow$	$\leftarrow$	$\leftarrow$	$\leftarrow$	Divide by
Unit			Symb	ol	Unit			Symb	ol	Factor
Atmospheres			_		. bar					1.01325
					. inches of merc					
					. mm of mercur					
					. pounds/square					
					. pounds/square					
					. calorie					
British thermal	unit		Btu		. joule			J		1054.35
British thermal	unit		Btu		. foot pounds			ft-lbs		777.65
British thermal	unit/hr .		Btu/hr		. kilowatts			kW		0.000293071
centimetres			cm		. inches			in		0.3937
centimetres pe	r sec		cm/se	c	. feet per minute	e		ft/min		1.969
					. feet per secon					
Celsius			°C		. Fahrenheit			°F		. (F-32) ÷ 1.8
centiStokes			cSt		. Saybolt			SUS		4.635 (>52 cSt)
cubic centimeti	res		cm³		. cubic inches			in³		0.06102
cubic feet			cu ft		. gallons US			US ga	I	. 7.481
cubic feet			cu ft		. cubic metres			m³		0.0283168
					. cubic centimet					
cubic inches			in³		. gallons US			US ga	l	0.004329
cubic yards			yd³		. cubic metres			m³		0.7646
degrees			(angle	)	. °radians			rad		0.0174533
Fahrenheit			°F		. Celsius			°C		(C x 1.8) + 32
feet			ft		. metres			m		0.3048
					. bar					
					. pounds/square					
					. inches of merc					
					. pounds/square					
					. inches of merc					
					. centimetres pe					
					. centimetres pe					
					. miles per hour					
					. miles per hour					
					. cubic centimet					
mana damede ex	•				. cubic centimet	00				29.5735
					. horsepower					
					. horsepower					
					. watts					
					. kilogram metre					
					. Newton metre					
•					. joule					
					. cubic inches					
					. gallons UK					
					. litres					
					. cubic feet					
					. litres					
					. gallons US					
					. British thermal					
					. foot pounds pe					
					. foot pounds pe					
					. kilowatts					
					. Pferde Starke					
horsepower			hp		. poncelet					0.7604



## **Conversion Factor Tables**

To convert → Into ← ←	=	→ ←	→ ←	Into → To convert	<b>→</b> ←	<b>→</b> ←	→ ← Symple	→ ←	Multiply by Divide by
Unit inches		Symbo		<u>Unit</u>			Symbo		<u><b>Factor</b></u> . 2.54
inches									
inch pounds									
inch pounds									
inches of mercury									
inches of mercury									
inches of mercury									
kilogram									
kilogram									
kilogram metre									
kilogram metre									
kilogram metre									
kilogram per square									
centimetre		kg/ciii-		. Dai			Dai		0.900003
kilopascals		kDa		har			har		0.01
kilometres									
kilometres									
litres									
litres									
metric horse power microinches									
miles									
millimetres mercury									
Newton									
Newton metre									
Newton metre									
Newtons per square centimetre									
Newtons per square		N/m <sup>2</sup>		. bar			bar		. 0.00001
metre									
Pascals									
pint UK									
pint US		US pt		. litres			l		. 0.473163
pounds									
pounds									
pounds/square inch									
pounds/square inch									
pounds/square inch									
pounds/square inch		psi		. feet of water			ft H2 O		. 2.307
pounds/square inch									
pounds/cubic foot		lb-ft³		. Kilograms/cubi	c metre		kg-m³ .		. 16.02
square inches		in²		. square centime	etres		cm²		. 6.5416
square feet									
Saybolt		SUS		. centiStokes			cSt		. See below
32 – 99 SUS									
100 – 240 SUS									
>240 SUS					,				

## **Troubleshooting Hints:**

Many of the failures in a hydraulic system show similar symptoms: a gradual or sudden loss of pressure or flow, resulting in loss of power or speed in the cylinders or hydraulic motors. In fact, the cylinders may stall under light loads or may not move at all. Often the loss of power is accompanied by an increase in pump noise, especially as the pump tries to build up pressure.

Any one of the system's components - pump, relief valve, directional valve, or cylinder could be at fault.

By following an organized step-by-step testing procedure, the problem can be traced to a general area, then if necessary, each component in that area can be tested, repaired or replaced.

Familiarize yourself with the circuitry of the hydraulic system to be tested. Review of the Service Manual is critical to learn the circuitry and location of various components: reservoir, hydraulic pump, relief valve, control valves, cylinders and hydraulic motors. The Service Manual should also provide operating specifications on fluid temperature, relief valve setting and pump delivery at specific RPMs.

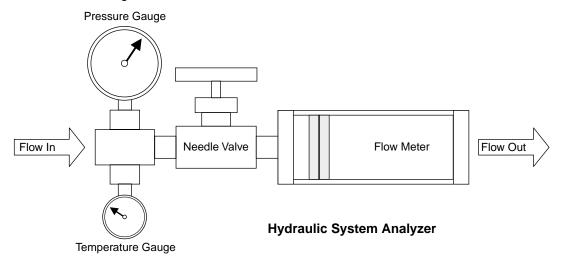
Check the obvious. Is there sufficient fluid in the reservoir? Is it dirty? Is the filtering system in proper condition? Are there any bent linkages or pinched hoses? Are quick couplers functioning properly?

Before you start troubleshooting a system, it is helpful to ask questions and find out about the problem:

- 1. Has the system been working fine and then it just guit and stopped working?
- 2. Has the system been working fine and then it started to slowly change in the way it works?
- 3. Does the system work fine when first started and then changes as it is used and as the system heats up?
- 4. Did the system stop working after something else was fixed or replaced?
- 5. Did the system ever work or work properly after it was assembled?

When troubleshooting a hydraulic system there are a few tools which will aid in finding and repairing a problem.

- 1. Pressure Gauge; To measure the System Pressure (P.S.I.).
- 2. Flow Meter; To measure Gallons per Minute (G.P.M.).
- 3. Temperature Gauge; To measure Heat.
- 4. Needle Valve; To Load / Restrict the System being tested.
- 5. And Your Senses;
  - A. Seeing; Is that suppose to be BENT?
  - B. Hearing; Is it suppose to be that LOUD?
  - C. Smell; Is it suppose to SMELL BURNT?
  - D. Touch; I can't it's too HOT!
  - E. Common; Do I want to get in there while it's WORKING? I DONT THINK SO!



With items 1, 2, 3, 4, and the necessary fittings and hoses, a Hydraulic System Analyzer can be built and most hydraulic system problems can be diagnosed and repaired.

Make sure the items are sized properly for the system being tested, don't use a 200 PSI Gauge in a 3000 PSI System or a 5 GPM Flow Meter or Needle Valve in a 50 GPM System. Verify and make sure that all the components are rated at, or exceed the pressures and flows being tested.



## Troubleshooting Hints

## A Hydraulic Analyzer can be used to check the following:

- 1. Fluid Temperatures, using the temperature gauge provided. Fluid should be flowing through the analyzer for several minutes to obtain an accurate reading.
- 2. Flow rates, using the flow meter provided. With the needle valve wide open, the monitor will show the rate at minimum pressure loss. The flow rate can be restricted by turning in on the needle valve from wide open to show the flow at various pressure loads.
- 3. System or operating pressure, by referring to the pressure gauge. To prevent possible component damage, always be sure the needle valve on your analyzer is in the wide open position prior to starting system and if possible, have a relief valve between pump and analyzer.

## **Example Test for Hydraulic Pump Performance:**

- 1. With system off and needle valve on analyzer in wide open position, install the analyzer in the line with fluid to be flowing in the direction of the arrow on the flow monitor scale.
- 2. Tighten all fittings to prevent unnecessary leakage.
- 3. Allow fluid to flow through the analyzer by turning on system.
- 4. Check the system Service Manual to see what specifications the pump is rated at. Example: 15 gpm @ 1500 psi.
- 5. With full flow flowing through the analyzer, start turning in the analyzer needle valve, gradually restricting the flow and at the same time, increasing pressure load on the pump. When reaching the rated pressure of the system pump, determine if the pump is operating efficiently (proper flow rate) or if it may need replacing or rebuilding. If the system relief valve is set below test pressure, the relief may have to be increased slightly to test the pump. Be sure to turn the relief valve back to its previous setting when tests have been completed.

## **Troubleshooting Heated Fluid:**

When analyzing a hydraulic system in which the fluid temperature is higher than normal, it should be kept in mind that hot fluid can produce unusual flow and operating characteristics. A flow monitor with a minimal sensitivity to temperature variation should be used. When fluid gets hot, the viscosity decreases (the fluid gets much thinner). This thinner fluid can pass through much smaller openings or, in other words, more fluid will pass through the same original opening.

## When the System Heats Up:

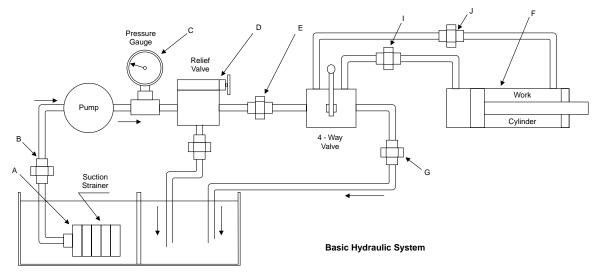
- 1. Pumps usually slip more fluid through standard clearances. High pressure settings usually cannot be obtained.
- 2. When the fluid thins down, the parts run closer together and wear faster. Particles of dirt which may not have been a problem with thicker fluid may now be very damaging.
- 3. Valves, cylinders and actuators will slip more fluid through standard clearances.

All-in-all, excessive heat in a system will not only cause excessive and faster wear, but the system will seem very sluggish because of the lack of fluid supply and operating pressure.

Knowing the potential of your analyzer, the effects of fluid temperature and pressure drop will always insure confidence in analyzing and troubleshooting any service problem areas. The analyzer is only as good as the operator and the less complicated the unit, the more it will be used.



### **Basic Troubleshooting Steps:**



### Step 1 - Pump Suction Strainer...

Probably the field trouble encountered most often is cavitation of the hydraulic pump inlet caused by restriction due to a dirt build-up on the suction strainer. This can happen on a new as well as on an older system. It produces systems such as: increased pump noise, loss of high pressure and / or speed.

If the strainer is not located in the pump suction line it will be found immersed below the oil level in the reservoir, as at Point A. Some operators of hydraulic equipment never give the equipment any attention or maintenance until it fails. Under these conditions, sooner or later, the suction strainer will probably become sufficiently restricted to cause a breakdown of the whole system and damage to the pump.

The suction strainer should be removed for inspection and should be cleaned before reinstallation. Wire mesh strainers can best be cleaned with an air hose, blowing from inside out. They can also be washed in a solvent which is compatible with the reservoir fluid. Kerosene may be used for strainers operating in petroleum based fluid. Do not use gasoline or other explosive or flammable solvents. The strainer should be cleaned even though it may not appear to be dirty. Some clogging materials cannot be seen except by close inspection. If there are holes in the mesh or if there is mechanical damage, the strainer should be replaced.

When reinstalling the strainer, inspect all joints, as at Point B for possible air leaks, particularly at union joints. There must be no air leaks in the suction line. Check the reservoir oil level to be sure it covers the top of the strainer by at least 3" at minimum oil level, which is with all cylinders extended. If it does not cover to this depth there is danger of a vortex forming which may allow air to enter the system when the pump is running.

## Step 2 - Pump and Relief Valve...

If cleaning the pump suction strainer does not correct the trouble, isolate the pump and relief valve from the rest of the circuit by disconnecting at Point E so that only the pump, relief valve, and pressure gauge remain in the pump circuit. Cap or plug both ends of the plumbing which was disconnected. The pump is now deadheaded into the relief valve. Back out relief valve pressure adjustment. Start the pump and watch for pressure buildup on the gauge while tightening the adjustment on the relief valve. If full pressure can be developed, obviously the pump and relief valve are operating correctly, and the trouble is to be found further down the line. If full pressure cannot be developed in this test, continue with Step 3.

## Step 3 - Pump or Relief Valve?...

If high pressure cannot be obtained in Step 2 by running the pump against the relief valve, further testing must be conducted to see whether the fault lies in the pump or in the relief valve. Proceed as follows:

If possible, disconnect the reservoir return line from the relief valve at Point H. Attach a short length of hose to the relief valve outlet. Hold the open end of this hose over the reservoir filler opening so the rate of oil flow can be observed. Start the pump and run the relief valve adjustment up and down while observing the flow through the hose. If the pump is bad, there will probably be a full stream of oil when relief valve adjustment is backed off, but this flow will diminish or stop as the adjustment is increased. If a flowmeter is available, the flow can be measured and compared with the pump catalog rating.



## Troubleshooting Hints

If a flowmeter is not available, the rate of flow on small pumps can be measured by discharging the hose into a bucket while timing with the sweep hand on a watch. For example, if a volume of 10 gallons is collected in 15 seconds, the pumping rate is 40 GPM, etc.

If the gauge pressure does not rise above a low value, say 100 PSI, and if the volume of flow does not substantially decrease as the relief valve adjustment is tightened, the relief valve is probably at fault, and should be cleaned or replaced as instructed in Step 5.

If the oil flow substantially decreases as the relief valve adjustment is tightened, and if only a low or moderate pressure can be developed, this indicates trouble in the pump. Proceed to Step 4.

### Step 4 - Pump...

If a full stream of oil is not obtained in Step 3, or if the stream diminishes as the relief valve adjustment is tightened, the pump is probably at fault. Assuming that the suction strainer has already been cleaned and the inlet plumbing has been examined for air leaks, as in Step 1, the oil is slipping across the pumping elements inside the pump. This can mean a worn-out pump, or too high an oil temperature. High slippage in the pump will cause the pump to run considerably hotter than the oil reservoir temperature. In normal operation, with a good pump, the pump case will probably run about 20° F above the reservoir temperature. If greater than this, excess slippage, caused by wear, may be the cause.

Check also for slipping belts, sheared shaft pin or key, broken shaft, broken coupling, or loosened set screw.

## Step 5 - Relief Valve...

If the test of Step 3 has indicated the trouble to be in the relief valve, D, the quickest remedy is to replace the valve with another one known to be good. The faulty valve may later be disassembled for inspection and cleaning. Pilot operated relief valves have small orifices which may be blocked with accumulations of dirt. Blow out all passages with an air hose and run a small wire through orifices. Check also for free movement of the spool. In a relief valve with pipe connections in the body, the spool may bind if pipe fittings are over tightened. If possible, test the spool for bind before unscrewing threaded connections from the body, or, screw in fittings tightly during inspection of the valve.

## Step 6 - Cylinder...

If the pump will deliver full pressure when operating across relief valve in Step 2, both pump and relief valve can be considered good, and the trouble is further downstream. The cylinder should be tested first for worn out or defective seals.

Run the cylinder to one end of its stroke. Disconnect the fluid line which was allowing oil to exhaust from the cylinder. Plug or cap the valve side of this disconnected line to avoid oil spillage caused by any back pressure in the tank return line. Attach a hose to the cylinder fitting where the fluid line was disconnected. Place open end of attached hose into a barrel or bucket. Start the pump and activate the valve to continue to stroke the cylinder the same direction. With the cylinder at the end of its stroke, check for any oil flowing from hose into barrel. If flow is excessive the cylinder may need repaired or replaced. Pistons with metal rings can be expected to have a small amount of leakage across the rings, and even those "leaktight" soft seals may have a small bypass during break in of new seals or after the seals are well worn. After checking, reinstall the lines and run the piston to the opposite end of the barrel and repeat the test. Occasionally a cylinder will leak at one point in its stroke due to a scratch or dent in the barrel. Check suspected positions in mid stroke by installing a positive stop at the suspected position and run the piston rod against it for testing. Once in a great while a piston seal may leak intermittently. This is usually caused by a soft packing or O-ring moving slightly or rolling into different positions on the piston, and is more likely to happen on cylinders of large bore.

#### Step 7 - Directional Control Valve...

If the cylinder has been tested (Step 6) and found to have reasonably tight piston seals, the 4-way valve should be checked next. Although it does not often happen, an excessively worn valve spool can slip enough oil to prevent build-up of maximum pressure. Symptoms of this condition are a loss of cylinder speed together with difficulty in building up to full pressure even with the relief valve adjusted to a high setting. This condition would be more likely to occur with high pressure pumps of low volume output, and would develop gradually over a long period of time.

## **Other Components**

Check other components such as by-pass flow controls, hydraulic motors, etc. Solenoid 4-way valves of the pilot operated type with tandem or open center spools may not have sufficient pilot pressure to shift the spool.



## System Inoperative:

## 1. Insufficient or No fluid in system.

Refill system with proper grade and type of fluid. Filter new oil being added as recommended. Refill oil reservoir with cylinders in closed position. If refilled while cylinders are extended the reservoir may over flow when or as the cylinders retract. Check for leaks.

## 2. Pump is not engaged.

Is pump shaft turning?

Check if PTO (power take off) is engaged. Variable control mechanism out of adjustment. Adjust to machine service manual specifications.

## 3. Slipping or broken pump drive.

Check pump drive mechanism (drive key, flex coupler) for damage.

Check for proper alignment or tension.

## 4. Pump inlet line plugged.

Drain oil and replace filter or filter element.

Check for clogged oil strainers.

Oil lines dirty or collapsed.

Check if correct inlet hose is used, inner liner may be collapsed.

Never use a pressure type hose as a pump inlet suction hose.

Check if supply shut off or gate valve is closed. Check in reservoir for other possible obstructions.

## 5. Pump speed too slow.

Check minimum drive speed.

May be too slow to prime pump.

### 6. Wrong fluid in system.

Oil viscosity too heavy for pump to pick up a prime. Drain complete system. Add new fluid of proper viscosity.

### 7. Air leaks at intake. Pump not priming.

Circuit must be tested at inlet connections. At pump intake piping joints, test by pouring oil on joints while listening for a change in sound of operation.

Determine where air is being drawn into line connection and tighten.

At pump shaft, Test by pouring oil on shaft seal while listening for a change in sound of operation. Follow manufacturer's recommendation when changing seals.

Air drawn in through intake pipe opening. Check to be certain suction and return lines are well below oil level in reservoir. Add oil to reservoir if necessary.

## 7. Worn or dirty pump.

Clean, repair or replace.

Check alignment.

Check for contaminated oil.

Drain and flush system.

## 8. Pump driven in wrong direction of rotation.

Most pump assemblies will have an arrow showing correct rotation.

On gear type pumps, the pressure port / output will be on the side where the gears come together and mesh.

Check to assure correct pump rotation was applied during assembly.

#### 9. Leakage.

Check all components, particularly the relief valve for proper settings.

Refer to technical manuals.

## 10. Broken or badly worn components (pump, valves, cylinders, etc.).

Examine and test for internal or external leakage. Analyze the conditions that brought on the failure and correct them.

Repair or replace the faulty components according to technical manual specifications.

#### 11. Excessive load.

Check unit specifications for load limits.

### **System Develops No Pressure:**

## 1. Pump not delivering fluid.

Follow the remedies mentioned above.

## 2. Incorrect valve position or setting.

Check and engage valve.

Install pressure gauge and adjust to correct pressure.

## 3. Vanes in vane pump sticking.

Check for burrs or metal particles that might hold vanes in their slots.

Repair or replace if necessary.

Clean system if contaminants are found.

## 4. Fluid recirculating back to reservoir and not going to functions.

Mechanical failure of some other part of the system, especially a relief valve.

If contamination is involved, clean and refill with proper fluid.

## 5. Piston pump or valve broken, or stuck open allowing fluid to return to inlet side.

Disassemble the pump, determine the cause and correct it.

Repair according to technical manual instructions.

## **System Operates Erratically:**

## 1. Air in system.

Check suction side for leaks. Repair.

#### 2. Cold oil.

Allow ample warm-up period.

Operate only at recommended operating temperature ranges.

## 3. Wrong fluid viscosity.

Oil viscosity too heavy.

Drain complete system. Add new fluid of proper viscosity.



## Troubleshooting Hints

### 4. Pump speed too slow.

Increase engine speed.

Check manual for recommendations.

## 5. Dirty or damaged components.

Clean or repair as necessary.

### 6. Restriction in filters or lines.

Clean and/or replace elements or lines.

## 7. Internal pump parts are sticking.

Dismantle and repair according to technical manual instructions.

Look for burrs on parts or metal particles in fluid. If contaminants are the cause, clean and refill with proper fluid.

## Distance between internal parts has incresed due to wear.

Dismantle and repair.

If wear is abnormal, determine the cause by checking the operation and maintenance records as well as by examining the pump and system.

## **System Operates Slowly:**

## 1. Oil viscosity too high, cold oil.

Allow oil to warm up before operating machine.

## 2. Low pump drive speed.

Increase engine speed (check manual for recommendations).

If clutch or belt-driven, check for proper tension.

#### 3. Low oil level.

Check reservoir and add oil as necessary.

### 4. Air in system.

Check suction side for leaks, repair.

## 5. Badly worn pump, valves, cylinders, etc.

Repair or replace as needed.

## 6. Restriction in filters or lines.

Clean and/or replace elements or lines.

## 7. Improper adjustments.

Check orifices, relief, unloading, flow control valves, etc.

Adjust per manual.

#### 8. Oil leaks.

Tighten fittings, replace seals or damaged lines.

## **System Operates Too Fast:**

## Wrong size or incorrectly adjusted restrictor or flow control.

Replace or adjust as necessary.

## 2. Engine running too fast.

Reduce engine speed.

## Overheating of Oil in System:

## 1. Oil passing thru relief valve for excessive time.

Return control valve to neutral when not in use. System stalling under load, etc.

Fluid viscosity too high.

### 2. Relief or unloading valve set too high.

Install pressure gauge and adjust to correct pressure.

## 3. Incorrect oil, low oil, dirty oil.

Use recommended oil, fill reservoir, clean oil, replace filter element.

## 4. Engine running too fast.

Reduce engine speed.

### 5. Excessive component internal leakage.

Check stall leakage past pump, valve, motor, cylinder or other components.

Repair or replace component as necessary.

## 6. Restriction in filters or lines.

Check if line I.D.'s are too small causing high velocity. Check if valvings too small, causing high velocity. Clean and/or replace elements or lines.

## 7. Malfunctioning oil cooler / heat exchanger.

Check if water is shut off, if water cooled.

Check for clogging.

Clean repair.

### 8. Insufficient heat radiation.

Check for proper air circulation around reservoir. Ambient temperature too high for system design. Clean dirt and mud from reservoir and components.

## 9. Reservoir sized too small.

Increase reservoir size.

Add oil cooler or heat exchanger.

## 10. Reservoir assembled without or insufficient baffling.

Add baffling to allow fluid time to cool.

## Foaming of Oil:

## 1. Incorrect oil, low oil, dirty oil.

Replace, clean or add as needed.

#### 2. Air leaks.

Check suction line and component seals for suction leaks. Repair or replace.

## 3. Return of tank line not below fluid level.

Repair or replace.

## 4. Inadequate baffles in reservoir.

#### 5. Lack of anti-foaming additives in oil.

Replace fluid with proper grade.



### **Noisy Pump:**

## 1. Air leak in intake, or air is being drawn through the inlet line.

Repair or make sure the inlet line is submerged in fluid in the reservoir.

To check for leaks, pour fliud around the joints and listen for a change in sound of operation.

## 2. Low oil level, incorrect oil, foamy oil.

Check if oil viscosity too high or operating temperature too low.

Replace, clean or add proper grade and type of fluid as needed.

With rare exception all return lines should be below fluid level in reservoir

## Pump inlet line or inlet screen is restricted or clogged.

Clean or replace as needed.

## 4. Reservoir breather vent clogged.

Clean or replace as needed.

### 5. Worn or damaged pump.

Check and correct cause of parts failure.

Repair or replace as needed.

### 6. Pump speed too fast.

Operate pump within recommended speed.

## 7. Drive coupling mis-aligned.

Align unit and check condition of seals and bearings. Misalignment will cause wear and subsequent high noise level in operation.

## 8. Relief or unloading valve set too high.

Use reliable gauge to check operating pressure.

Relief valve may have been set too high with a damaged pressure gauge.

Check unloading devices to see that they are properly controlling the pump delivery.

## **Excessive Pump Wear:**

## 1. Abrasive contaminants or sludge in the fluid.

Check for the cause of contaminants.

Replace or repair worn parts according to service manual.

Install or change fluid filter.

Replace fluid with recommended grade and quality.

## 2. Viscosity of fluid too low or too high.

Replace fluid with proper grade and type.

## 3. Sustained high pressure above maximum pump rating.

Check for possible relief valve malfunction or other parts failure.

## 4. Air leaks or restriction in system causing cavitation.

Eliminate any leaks in system.

## 5. Drive shaft misaligned.

Check and correct according to technical manual specifications.

### **Leaky Pump or Motor:**

## 1. Damaged or worn shaft seal.

Check and replace.

Check for misalignment.

Check that chemicals in fluid are not destroying packing or seals

### 2. Loose or damaged parts.

Tighten or replace.

## Internal Pump Parts Breakage:

## Excessive pressure above maximum limits for pump.

Check for parts malfunction and cause.

Repair according to machine technical manual.

#### 2. Seizure due to lack of fluid.

Check reservoir fluid level, as well as fluid inlet line for restriction.

Check for plugged inlet filter or strainer.

## 3. Abrasive contaminants in fluid are getting past the filter.

Check for plugged inlet filter or strainer.

Check for malfunctioning filter bypass valve.

## 4. Excessive torquing of housing bolts.

Replace damaged parts.

Torque to proper specifications.

### **Load Drops with Control Valve in Neutral:**

## 1. Leaking cylinder seals or fittings.

Replace worn parts.

#### 2. Control valve not centering when released.

Check linkage.

Check spool for binding.

## Control Valve Sticking (Binding):

## 1. Valve linkage misaligned.

Repair.

### 2. Tie-bolts too tight (stack valves).

Loosen and retighten as necessery.

### 3. Valve damaged.

Repair or replace.

## **Control Valve Leaks:**

## 1. Tie-bolts too loose (stack valves).

Tighten as necessery.

## 2. Seals damaged or worn.

Replace.

#### Relief Valve Noisy:

## 1. Relief valve setting too close to operating pressure.

Install pressure gauge and adjust to correct pressure.

## 2. Worn or scored poppet and seat.

Replace.

## 3. Spring in relief valve broken.

Replace spring and adjust to correct pressure.



## Cylinder Seal Leakage:

## 1. Slow, Uniform Leakage:

- A. Poor low-pressure sealability (especially if a lip seal).
- B. Too little initial interference (if squeeze-type seal).
- C. Loss of interference or squeeze due to wear or compression set.
- D. Seal shrinkage after installation (possibly chemically induced, or a result of leaching of plasticizers by solvent action).
- E. Possible omission or failure of static seal(s).
- F. Microscopic debris lodges under seal lip (lint, fiber, etc.).
- G.Scored lip due to passage of sharp particle under seal, leaving cut or nick.
- H. Seal lip is nicked or cut during installation (note whether leak starts immediately after seal installation).
- Non-repetitive overheating hardens compound (which loses its ability to conform to dynamic surface deviations).
- J. Off-center alignment puts all clearance on one side, all compression on the other (due to bearing wear, excessive side loads, etc.).
- K. Check static surfaces of dynamic seal (groove surfaces). They may have problems F, G, or H hidden from view, and without self-cleaning tendency.

## 2. Gradually Increasing Leakage:

- A. Progressive wear.
- B. Increasing compression set.
- C. Progressive tear or erosion from initial nick.
- D. Fine score mark on dynamic surface progressively abrades seal lip.

## 3. Sudden Copious Leakage:

- A. Extruded seal.
- B. Torn seal lip (see 1-D, -E, -F, -G, -H, & -I, and 2-D).
- C. Twisted seal.
- D. Dramatic bearing failure due to excessive side load, shock, etc.
- E. Spiral failure.
- F. Massive infusion of contamination (due to incorrect fluid added to system, or to upstream introduction of dirt or wear debris).
- G.Slow rod leakage builds up behind tight wiper, then dumps ... giving appearance of catastrophic seal failure. If leak rate continues, look for slow leak or erratic leak causes. If high leak rate continues, look for true catastrophic leak origin.
- H. Reverse-pressue blowout of piston seal due to pressure trap or failure of opposed seal.

## 4. Erratic (start-stop) Leakage:

- A. Cold start-up shrinks seal; friction/fluid heating restores size.
- B. Intermittent eccentric loading.
- C. Fibrous contamination working its way past seal lips.
- D. Unstable seal (twists and returns, cocks, etc.) *usually* caused by shock loading.
- E. Rod seal leaks slowly, tight wiper periodically dumps accumulated leakage (see 3-G).
- F. Fluid viscosity changes as temperature cycles (e.g., forklift truck alternately entering and leaving cold storage area).

## 5. Stick-slip Operation:

- A. Worn-away, low friction surface treatment.
- B. Breakdown of fluid lubricity due to contamination or deterioration of fluid.
- C. Viscosity change due to temperature.
- D. Excessive burnishing of dynamic surface to finer finish destroys ability of surface to maintain lube film (e.g., may go from 12 RMS to 4 RMS).

### 6. Seizing:

- A. Seal and bearing swell due to incompatible fluid and compound (possibly running hotter than temperature at which fluid is compatible).
- B. Thermal expansion of compound.
- C. Pressure trap between dual squeeze seals or incorrectly installed lip seals.
- D. Wedging of seal or backup device into extrusion gap (if used, it is usually the backup device that extrudes).
- E. In low-pressure systems, shock or other factors cock, cant or misorient the seals in grooves.
- F. Bent rod, cocked head, etc.

## 7. Scored Rod or Ram:

- A. Internally generated contamination.
- B. Externally introduced rod dirt, dirty makeup or disassembly/ reassembly dirt.
- C. Misoriented exclusion devices (wiper/scraper); eccentric installation.
- D. Misaligned (eccentric) loads cock ram into metalto-metal contact with head.
- E. Wiper in vertical ram forms catch-all pocket.

## 8. Drift:

- A. Inspect valve for leakage and full closure *before* disassembly. (disconnect return line on valve and inspect visually for leakage).
- B. See problems 1 and 2 as applied to piston seals.
- C. Misapplied cast-iron rings in a "hold" cylinder (right ring in the wrong job).
- D. In "retract-mode" creep, check rod seal as well as piston seal.
- E. Static internal seal may provide leakage path past piston.



## 9. Increasing Cylinder Drag:

- A. Seal swell caused by improper (incompatible) *installation* lubricant (e.g., EPR seal lubed with petro-based grease or oil).
- B. Packing of contaminants into wiper groove of vertical ram.
- C. Thermal expansion of bearings and/or seals.
- D. Apparent drag increase due to undetected flow restriction in supply or return line . . . or bypassing of pressure though improperly closing valve . . . or obstructed check valve, etc.
- E.Cocked or twisted seal bypassing fluid and wedging into extrusion gap.

## 10. Increasing Cylinder/Rod Temperature:

- A. See causes for problem 9. In their earlier stages, these problems may appear as hotter-running cylinders.
- B. Internal leakage "throttling" past seal can cause rapid heating.
- C. Decreased lubricity of fluid can boost friction and heating (hotter fluid has lower viscosity, etc.). Contaminated or deteriorated fluid can cause same cycle.
- D. Diluted fluid can boost friction, etc.
- E. Condensation in reservoirs can emulsify or hit cylinder as slugs of fluid with near-zero lubricity. Also, hot water can *swell* compounds such as urethanes, increasing friction.

## 11. Telescopic sleeve undersized, out of round or bulged.

A. Check with micrometers to see if sleeve/tube is with-in specifications.

## Cylinder will not Operate or Move:

1. Pump or PTO is not engaged, system not receiving fluid.

Engage pump, correct pump flow problem.

2. Control valve not engaged.

Engage valve, check for linkage alignment and damage.

3. Pressure too low.

Check pressure at cylinder to make sure it is to circuit requirements.

4. Cylinder bypassing internally.

Check for internal scoring, damaged or worn seals, internal cracks.

Cylinder overloaded for rated capacity. Reduce load.

- Cylinder too small or not rated for application. Install correct cylinder.
- Piston rod broken at piston end. Disassemble and replace piston rod.
- Hose quick disconnect not attached. Check if hose quick disconnect is connected properly.

## Cylinder not Holding Load or Drifts:

1. Cylinder bypassing internally.

Check for internal scoring, damaged or worn seals, internal cracks.

Pressurize one side of cylinder and disconnect fluid line at opposite port. Observe leakage. One to three cubic inches per minute is considered normal for piston rings. Virtually no leak with soft seals on piston. Replace cylinder barrel or seals as required.

### 2. Other circuit leaks.

Check for leaks thru operating valve and correct. Correct leaks in connecting lines.

## 3. Incorrect Valving.

Open center valve with conventional single rod cylinder will creep if restriction on tank port is sufficiently high. Use tandem type valve spool configuration or spool with pump dumped through one cylinder port with the other blocked.

Closed center valve can cause similar results except creep will be according to amount of clearance flow in the valve. Proper notching of valve spool can prevent building up pressure in cylinder lines between cycles.

Spools with pressure blocked and cylinder ports completely relaxed will also prevent drift if no moving element is not affected by gravity or vibration.

Pilot operated check valves can positively lock fluid in cylinder lines. Care must be exercised to insure adequate pilot pressure when rod differential may cause intensification.

## Cylinder operates erratically or chatters:

- **1. Telescopic cylinder sleeves mistaging.** Check for tight seals or bearings.
- Excessive friction due to damaged or improper / misaligned mounting.

Repair or replace as needed.

- **3. Cylinder sized too close to load requirements.** Reduce load or install larger cylinder.
- **4.** Large difference between static and kinetic friction. Install speed control valves to provide back pressure to control stroke.

### Oil Spilling Out of Tank:

1. Oil is foaming.

(Refer to Foaming of Oil section)

**2.** Oil reservoir filled while cylinders were extended. Fill while cylinders in the retract or closed position.

## Foreign Matter Sources in the Circuit:

- Pipe scale not properly removed.
   Lines need cleaned and flushed before installation.
- 2. Sealing compound (pipe dope, teflon tape) allowed to get inside fittings.

Use care when applying sealants.

3. Improperly screened fill pipes and air breathers. Repair or replace as required.



## **Troubleshooting Hints**

- **4. Burrs inside piping components.** Deburr before installation.
- Tag ends of packing coming loose. Check if packing is system compatible. Replace packing.
- 6. Seal extrusions from pressure higher than compatible with the seal or gasket.

Replace seal or gasket with compatible item.

7. Human element.

Not protecting components while being repaired. Repaired components not properly protected while stored. (Rust and other contaminants.) Lines left open and unprotected.

 Wipers or boots damage or not provided. Check cylinders or rams. Add or replace where necessary.

## Preventing repeat failures:

When a hydraulic system (pump or cylinder or other major component) has a failure, implementing this 13-step checklist can help prevent repeat failures.

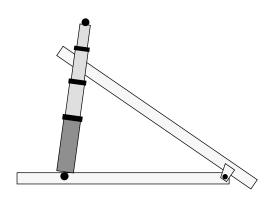
- 1. Determine cause of failure.
- 2. Eliminate cause of failure.
- 3. Retract all cylinders and drain tank.
- 4. Flush tank. Using diesel fuel under pressure, flush tank thoroughly and wipe with clean cloths.
- 5. Install new filter elements.
  - A. Check to make sure filter is 10 microns or better.
  - B. If machine does not have filtration, install a 10 micron filter on the return line.
- 6. Install new component.
- 7. Fill the tank with new oil.
  - A. Be sure recommended oil is used.
  - B. NOTE; You're filling the system, not just the oil supply tank. Pump failure due to lack of oil can result if filling is not done correctly.

    Keep a close check on the oil level as you complete the following steps.
- 8. Disconnect all lines to cylinders and/or motors at the cylinder or motor. Be sure all implements are securely blocked and all accumulators are bled before disconnecting lines. NOTE; It may be necessary to remove, inspect, and flush the fittings that are connected to the pump, valves, and/or cylinders to remove any foreign objects that may have become lodged or stuck inside them.
- Activate each circuit by moving control valve handle so lines are flushed with new oil. This flushes the lines and valves from pump to all cylinders and motors.
   Be sure to check oil level, and add new oil if necessary.

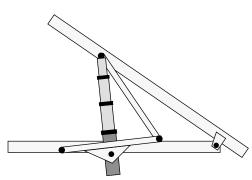
- 10. Connect lines to blind end of cylinders and all fluid motors. Leave rod end disconnected and with engine at one-fourth throttle, activate circuits slowly until cylinder bottoms out. New oil will be put in the blind end of the cylinder and old dirty oil flushed out the rod end. Do this for all cylinders on the machine.
- 11. Connect lines to rod end of cylinders. Again, check oil level and add new oil as required.
- 12. Operate all cylinders and motors alternately for 30 minutes at normal operating speed.
- Change filter element, check oil level and add oil as needed.

The above procedure, if followed, will allow you to install a new pump or cylinder with confidence, knowing that you'll get satisfactory life. Cutting short these steps can cause premature hydraulic component failure; a pump, nor a cylinder will run long on a contaminated system. In nearly all cases, a replacement component will fail in a shorter time than the original preceding it unless the system is thoroughly cleaned.

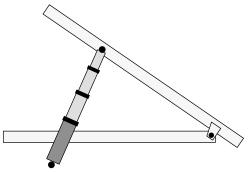
In addition, to ensure good service from your equipment, the hydraulic system must be properly maintained, including frequent oil level checks, daily inspection for leaks, filter element and oil changes at recommended intervals (using correct filters and recommended grade of oil), and finally, practicing good operating techniques.



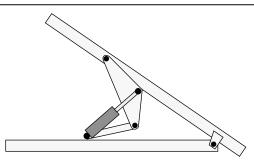
FRONT MOUNT TELESCOPIC HEAD LIFT OR BOTTOM LIFT



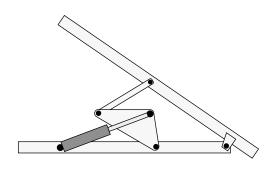
TELESCOPIC SCISSOR HINGE FORWARD OR REARWARD



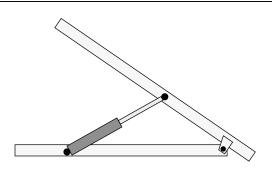
UNDERBODY TELESCOPIC
SLANT FORWARD OR SLANT REARWARD



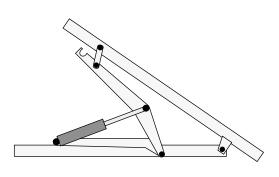
SINGLE STAGE SCISSOR



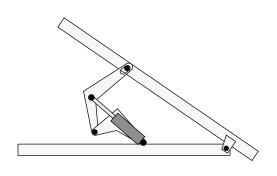
**UNDER BODY ARM HOIST** 



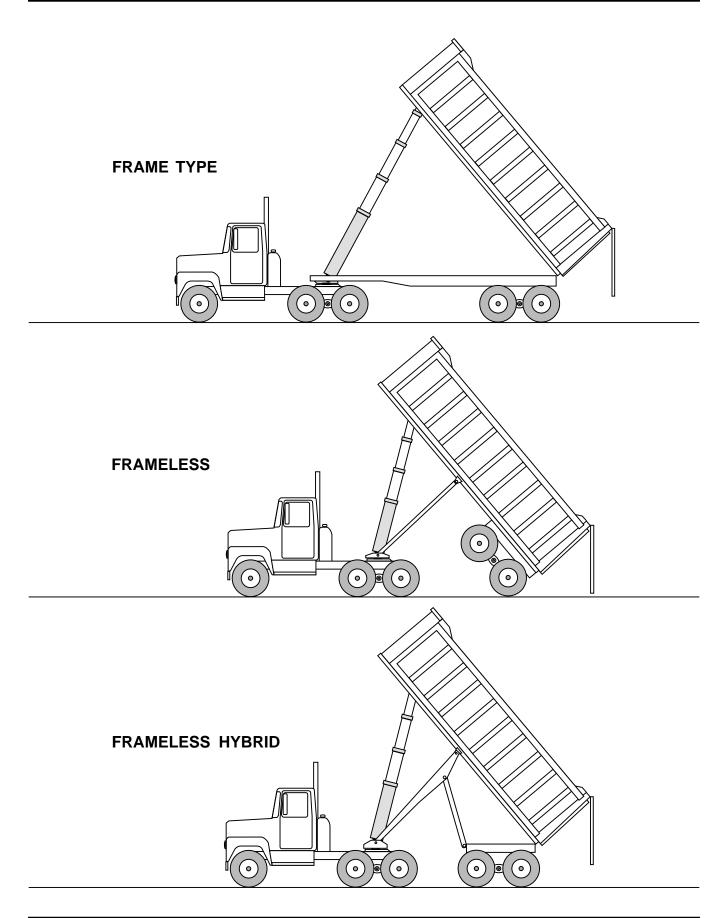
**UNDER BODY DIRECT LIFT** 



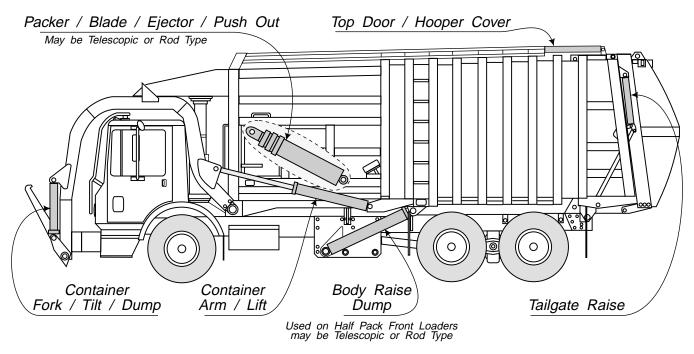
**UNDER BODY ARM - SCISSOR** 



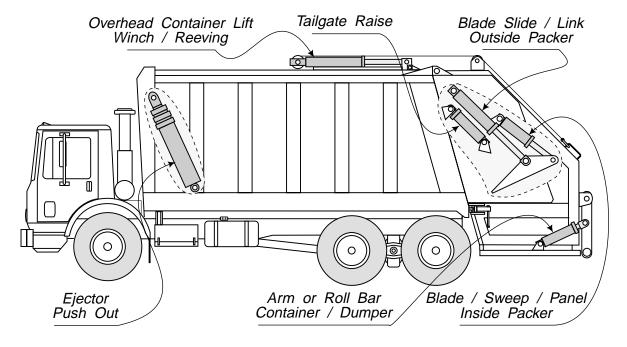
LOST MOTION SCISSOR



## Front Loader Refuse Bodies



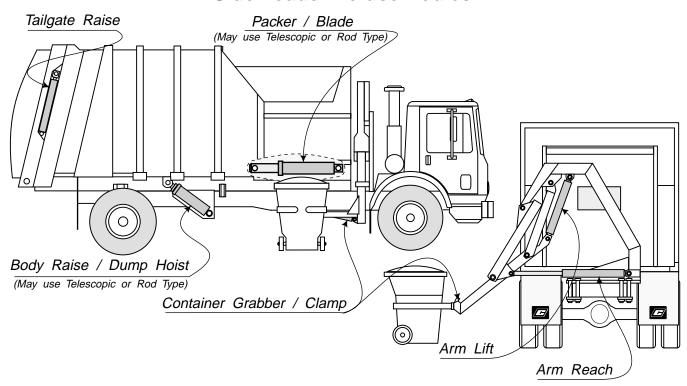
## Rear Loader Refuse Bodies



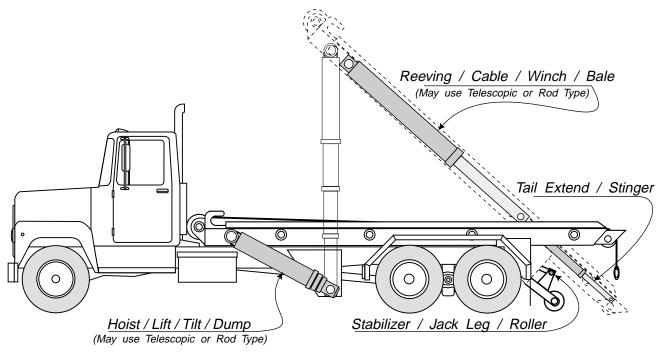
Crossover and Application Data is for Reference Only. No Other Conditions are Expressed or Implied.



## Side Loader Refuse Bodies



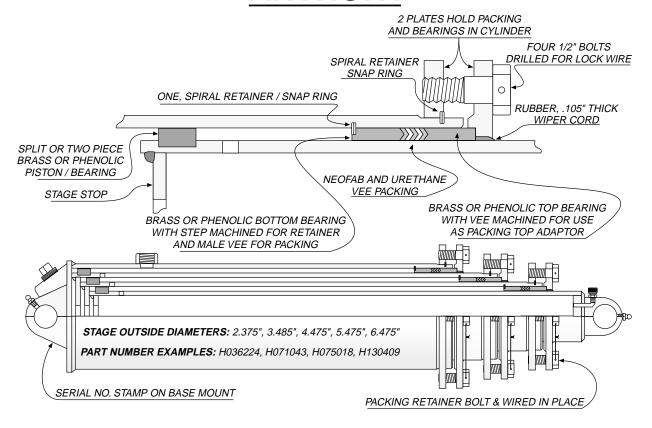
## Roll-Off & Tilt Frame Hoists



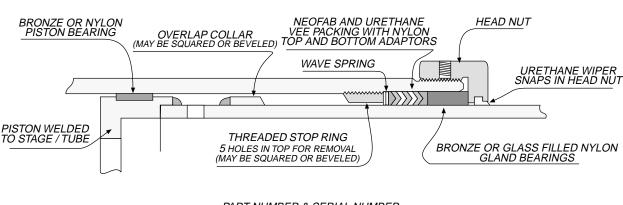
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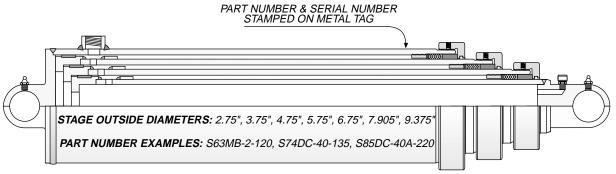


## **ANTHONY**

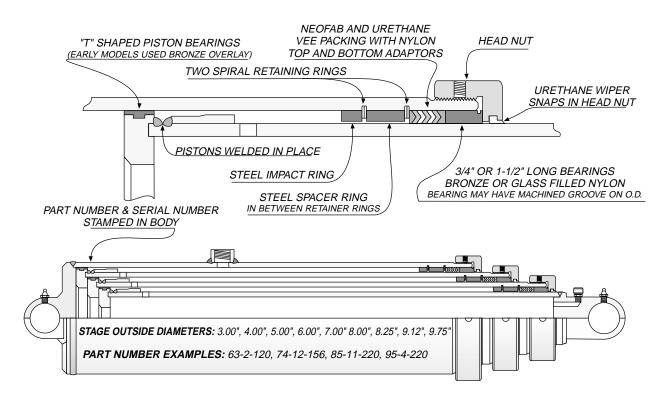


# **COMMERCIAL**

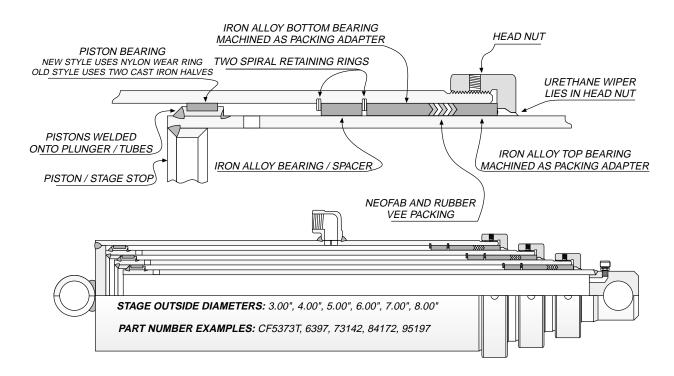




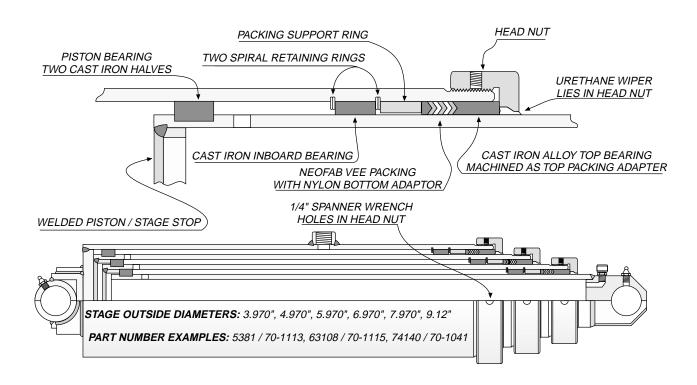
## **CUSTOM HOIST**



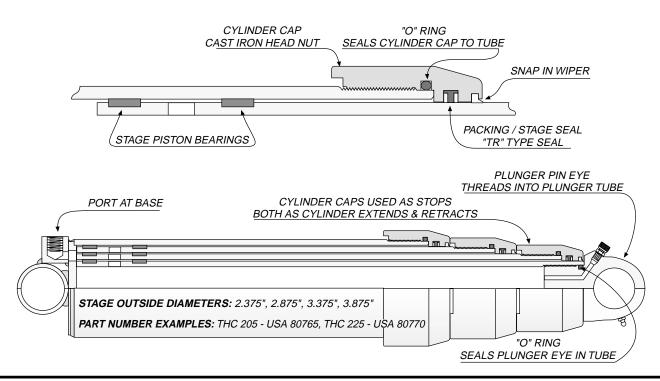
## <u>FONTAINE</u>



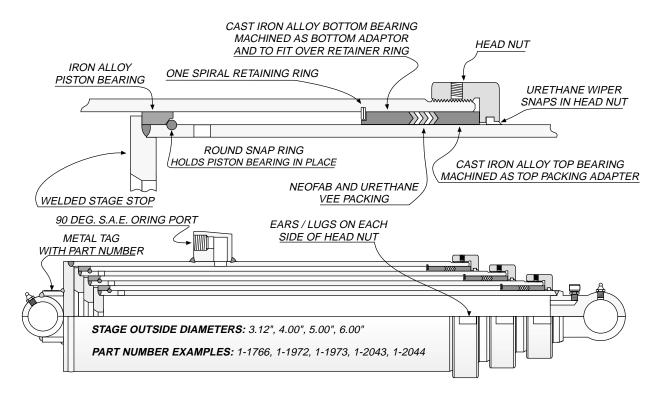
## PEABODY GALION



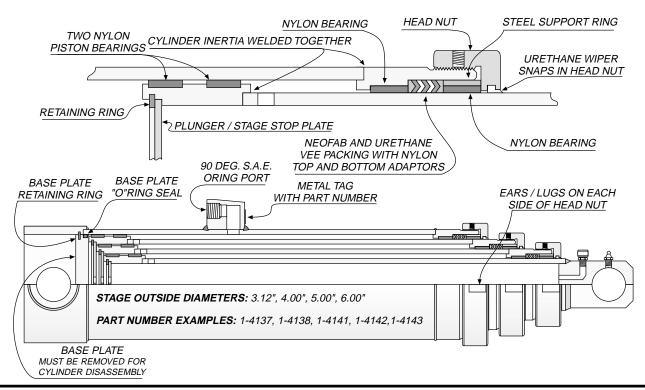
## GLENCO / FARMHAND



## **HEIL** "OLD STYLE" HPT SERIES

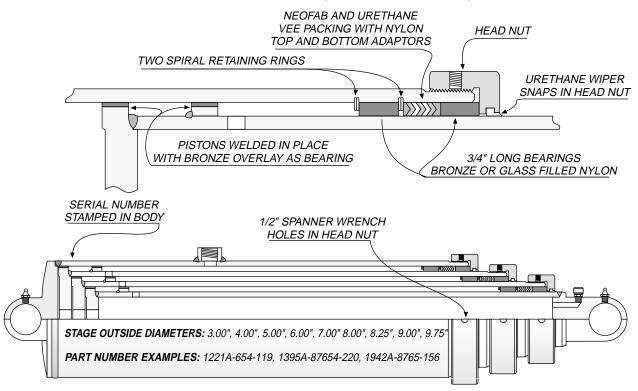


## **HEIL**"NEW STYLE" HPT SERIES



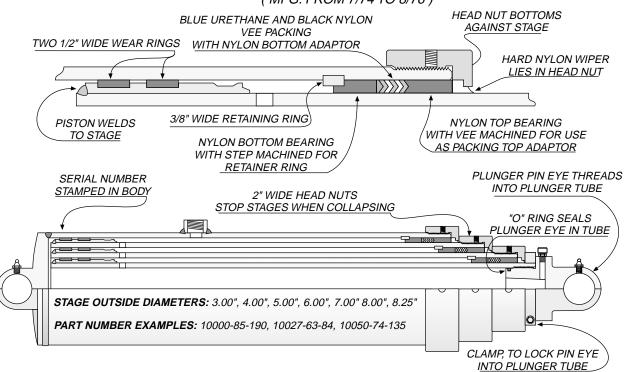
# **HYCO** 900 - 2000 SERIES

( MFG. PRIOR TO 6/74 )



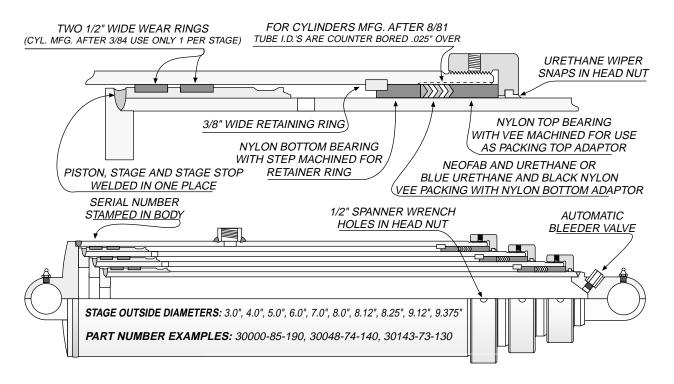
# HYCO 10,000 SERIES

( MFG. FROM 7/74 TO 8/76 )

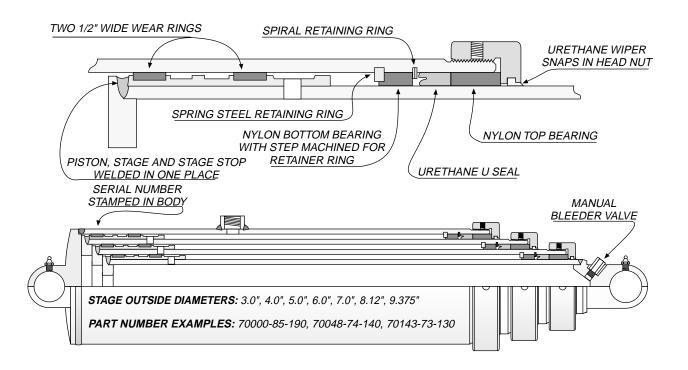


# HYCO 30,000 SERIES

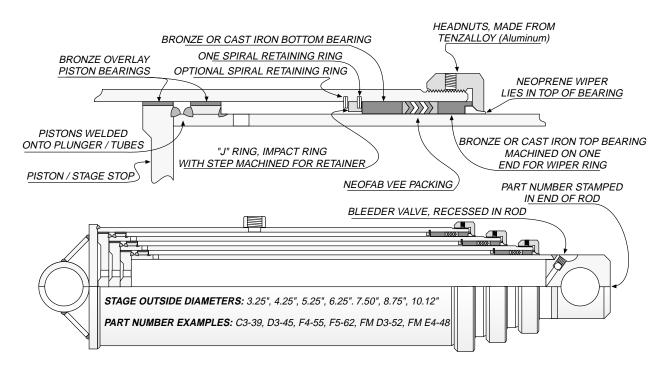
(OLD STYLE MFG. FROM 8/76 TO 7/81 ) (NEW STYLE MFG. FROM 8/81 TO PRESENT)



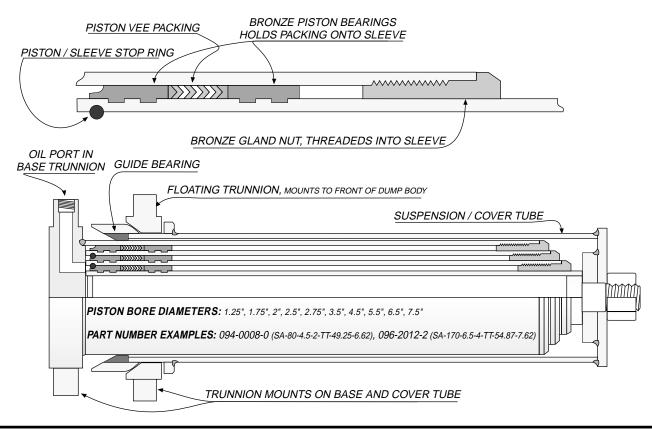
# HYCO 70,000 SERIES



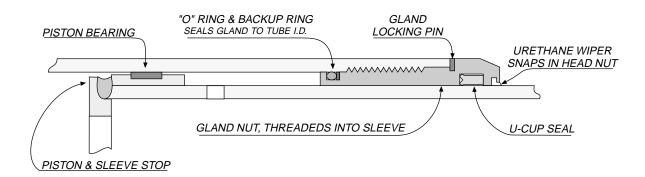
## **JOHNSON**

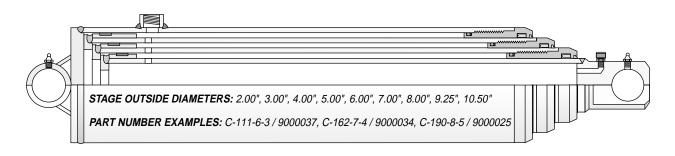


## LESSARD / DRASSEL COVER TUBE DESIGN

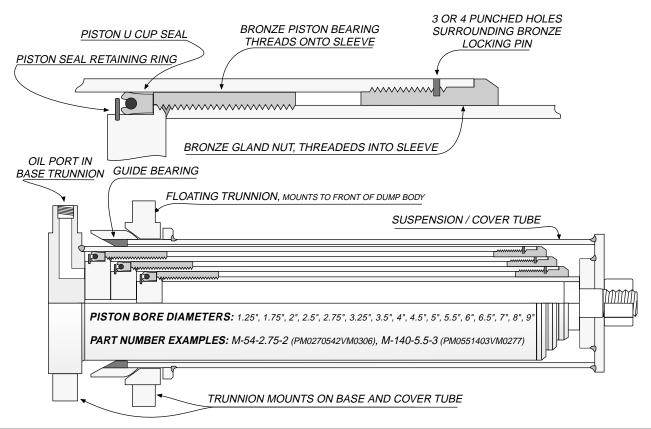


## MAILHOT / "C" MODEL

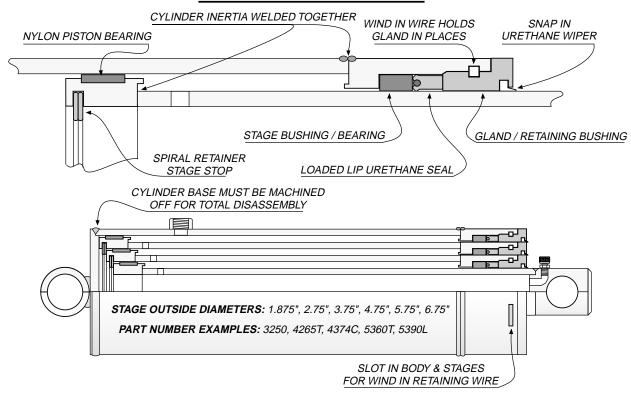




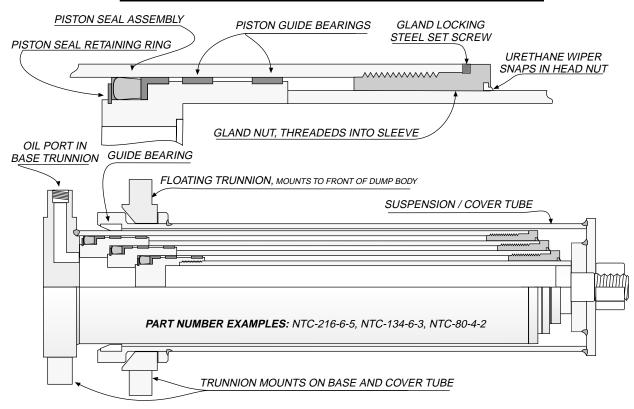
## **MAILHOT** / "M" MODEL COVER TUBE DESIGN



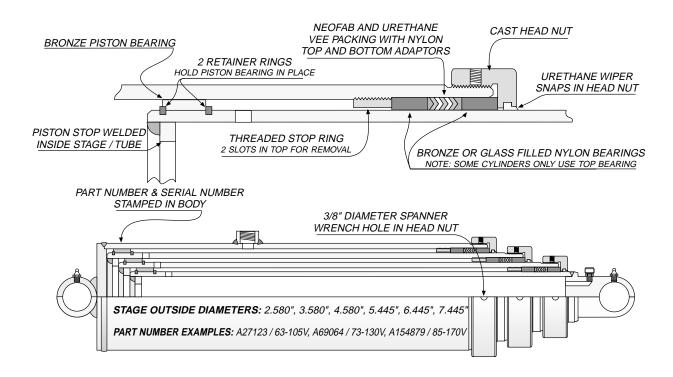
## MARION MFG.



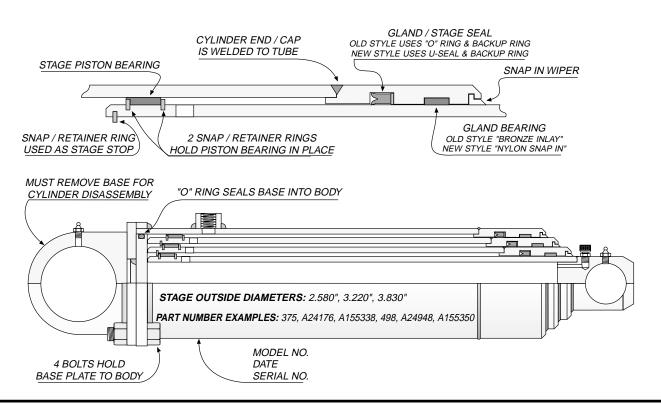
# NORDIC / NORD-SEN METAL INDUSTRIES



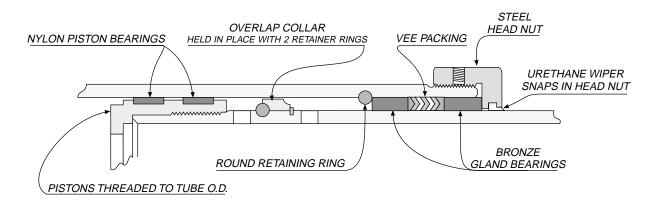
# PERFECTION

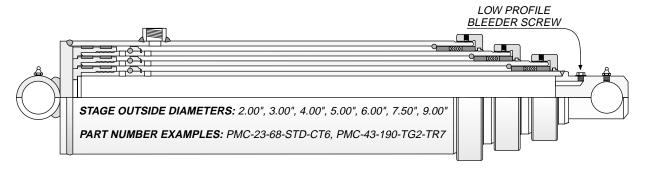


# **PERFECTION** FARM HOISTS

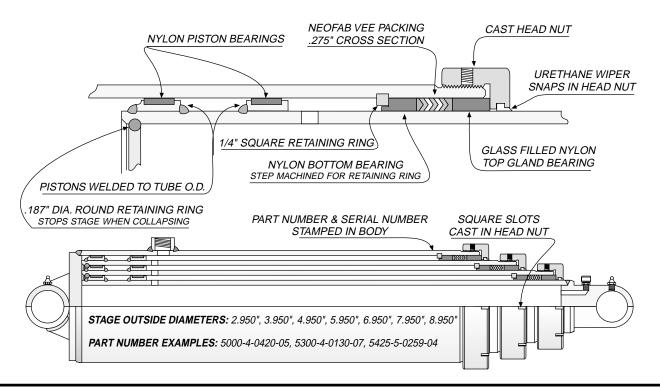


# **PRINCE**





# WARD CO



ANTHONY MIDWES	TRODY	COMMEDIAL UVD	DAULICC	COMMEDCIAL LIVERALILICS
ANTHONY MIDWES	_	COMMERCIAL HYD		COMMERCIAL HYDRAULICS
	S63DC-44-111 CM26 S73DC-40-110 CM23	S53DC-8-120		S64MB-1-140 S64MB-4-140
	S73DC-40-110 CM26	S53DC-8-130 S53DC-10-72		S73CC-17-110 S73DC-40-110
	S73DC-40-110 CM26	S53DC-10-72		S73CC-22-110 S73DC-40-110
	S73DC-40-124 CM25	S53DC-10-64		S73CC-22-124 S73DC-40-124
	S63DC-47-140 CM26	S53DC-12-104		S73DB-1-142 S73DC-40-140 CM23
11130403	003DO 47-140 OM20	S53DC-12-120		S73DC-22-110 S73DC-40-110
BENTON HARBOR		S53DC-13-104		S73DC-22-124 S73DC-40-124
P89645	D70LN-2-80	S53DC-13-126		S73DC-30-110 S73DC-40-110
P89648		S53DC-19-90		S73DC-30-124 S73DC-40-124
		S53MB-1-120		S73DC-30-129 S73DC-41-129
BIBEAU				S73DC-30-140 S73DC-40-140
CYLM100353	S43MM-1G-100	S62CC-15-69	S62CC-30-69	S73DC-32-129 S73DC-41-129
CYLM1004533		S63CC-22-80	S62DC 47.77	S73DC-32-140 S73DC-40-140
CYLM110453		S63CC-22-111		S73DC-33-124 S73DC-40-124
CYLM110553		S63CC-22-111		S73DC-35-120 S73DC-40-120
CYLM120453		S63DC-3-111		S73DC-40-129 S73DC-41-129
CYLM120553		S63DC-22-111		\$73DC-41-140 \$73DC-40-140
CYLM130553		S63DC-22-123		S73DC-64-129 S73DC-41-129 S73DC-65G-129 S73DC-41-129
CYLM130653		S63DC-22-80		373DC-03G-129 373DC-41-129
CYLM140653		S63DC-23-108		S74CC-22-120 S74DC-40A-120
CYLM150553		S63DC-23-128		S74CC-22-134 S74DC-40-135
CYLM150654 CYLM160654		S63DC-26-111	S63DC-44-111	S74CC-22-161 S74DC-40-161
CYLM170654		S63DC-27-111	S63DC-44-111 CM9	S74DC-3-134 S74DC-40-135
CYLM18274		S63DC-30-111		S74DC-5-135 S74DC-41-135
CYLM19574		S63DC-30-123	S63DC-44-123	S74DC-6-135 S74DC-40-135
CYLM90353			S63DC-47-107 CM37	S74DC-22-120 S74DC-40A-120
CYLM90452SA			S63DC-44-123 CM37	S74DC-22-134 S74DC-40-135
		S63DC-32-120		S74DC-22-161 S74DC-40-161
COMMERCIAL HYD	RAULICS	S63DC-32-126		S74DC-30-120 S74DC-40A-120
D60DB-4-72		S63DC-32-138		S74DC-30-135 S74DC-40-135
D60DD-5-69	D60DD-4-69	S63DC-32-162		\$74DC-30-138 \$74DC-40-135
D60DD-6-69		S63DC-33-105		\$74DC-30-140 \$74DC-40A-140
D62DB-5-15		S63DC-36-120 S63DC-37-82		\$74DC-30-156 \$74DC-40-156 \$74DC-30-161 \$74DC-40-161
D62DB-7-72			S63DC-47-02 S63DC-47-120 CM9	S74DC-30-161 S74DC-40-161
D62LB-2-81	D62LB-5-80		S63DC-47-120 CM9	S74DC-33-161 S74DC-40-161
D70LN-1-80	D70I N-2-80		S63DC-47-140 CM9	S74DC-40-120 S74DC-40A-120
D72DB-3-15		S63DC-43-123		S74DC-40-138 S74DC-40-135
D72DB-3-25	D72DB-7-25		S63DC-47-107 CM37	S74DC-40-140 S74DC-40A-140
D72DB-4-15			S63DC-44-123 CM37	S74DC-42-135 S74DC-40-135
D72DB-4-25			S63DC-47-120 CM9	S74DC-42-156 S74DC-40-156
D72DB-4-25-1		S63DC-48-130	S63DC-47-130 CM9	S74DC-42-161 S74DC-40-161
D72LB-6-81	D72LB-9-80		S63DC-47-140 CM9	S74DC-64G-161 S74DC-40-161
D72LP-1-80	D70LN-2-80	S63DC-52-77		S74MB-2-154 S74MB-3-154
D04DD 0.05	D04DD 7.05	S63MB-1-120		S74MM-1-160 S74MM-2G-160
D84DB-4-25		S63MB-1-140		S74MM-2-160 S74MM-2G-160
D84DB-4-25 D84DB-4-25-1		S63MB-2-120		S83CC-22-160 S83DC-40-160
D84DB-5-25		S63MB-5-108		S83DC-6-134 S83DC-26-134
D84DB-5-25-1		S63MB-8-120	S63MB-9-120	S83DC-22-160 S83DC-40-160
		S64DB-1-135	S64DB-6-135	S83DC-25-160 S83DC-40-160
S42MB-1-90	S42MB-3-90	S64DB-1-156		S83DC-30-139 S83DC-40-139
		S64DB-2-135		S83DC-30-160 S83DC-40-160
S53DB-1-103		S64DB-2-156	S64DB-6-156	S83DC-31-139 S83DC-40-139
S53DC-3-72	S53DC-14-72	S64DB-3-135	S64DB-6-135	S83DC-31-160 S83DC-40-160
S53DC-3-84	553DC-14-84	S64DB-3-156	S64DB-6-156	S83DC-32-139 S83DC-40-139
S53DC-5-107		S64DB-4-135		S83DC-33-139 S83DC-40-139 CM19
S53DC-5-126 S53DC-6-99		S64DB-4-156		00400 4 400 00400 10 100
S53DC-6-105		S64DB-5-135		S84CC-4-183 S84DC-40-188
S53DC-6-105	S53DC-15-100	S64DB-5-156		S84CC - 6-183 S84DC - 40-180
S53DC-6-123	S53DC-15-123	S64DC-1-157		S84CC-8-166 S84DC-40-161 CM38 S84CC-8-183 S84DC-40-180 CM38
S53DC-7-84		S64DC-2-157		S84CC-22-134 S84DC-40-140
S53DC-8-104		S64DC-4-156		S84CC-22-134 S84DC-40-140
	-	S64DC-5-156	30400-9-150	33.33 22 110 00400 40 140



COMMERCIAL HYD	RAULICS	COMMERCIAL HYD	RAULICS	COMMERCIAL HYDRAULICS
S84CC-22-155		S85DC-13-210		SD63CC-22-124 SD63CC-24-124
S84CC-22-172	S84DC-40-180	S85DC-22-176	S85DC-40-170	SD63DB-4-120 SD63DB-14-120
S84CC-22-183		S85DC-22-183	S85DC-40-190	SD63DB-5-120 SD63DB-14-120
S84DC-4-134		S85DC-22-190	S85DC-40-190	SD63DB-9-120 SD63DB-14-120
S84DC-4-149	S84DC-40-148	S85DC-22-197	S85DC-40-197	SD63DB-13-120 SD63DB-14-120
S84DC-4-183	S84DC-40-188	S85DC-22-220	S85DC-40A-220	SD64CC-6-132 SD64CC-7-132
S84DC-7-183	S84DC-40-180	S85DC-22-235	S85DC-40-235	SD64CC-6-152 SD64CC-7-152
S84DC-8-134		S85DC-22-250		
S84DC-8-149	S84DC-40-148	S85DC-23-220	S85DC-40A-220 CM4	SD73CC-9-120 SD73CC-25-120
S84DC-22-134		S85DC-23-235	S85DC-40-235 CM4	SD74CC-9-131 SD74CC-11-131
S84DC-22-149		S85DC-25-183		SD74DB-8-144 SD74DB-6-144
S84DC-22-155		S85DC-25-190		SD83CB-5-117 SD83CB-16-117
S84DC-22-161		S85DC-25-250		SD83CB-6-117 SD83CB-16-117
S84DC-22-172		S85DC-30-190		SD83CB-7-117 SD83CB-16-117
S84DC-22-183		S85DC-30-197		SD83CB-13-117 SD83CB-16-117
S84DC-25-134		S85DC-30-220		SD83CB-15-117 SD83CB-16-117
S84DC-25-149		S85DC-30-235		SD83CC-5-132SD83CC-24-132
S84DC-25-161		S85DC-30-250		SD83CC-5-156 SD83CC-24-156
S84DC-25-183		S85DC-31-190		SD83CC-16-126 SD83CC-27-126
S84DC-30-134		S85DC-35-250		SD83CC-25-126 SD83CC-27-126
S84DC-30-149 S84DC-30-161		S85DC-36-250 S85DC-36-265		SD83CC-26-126 SD83CC-27-126
S84DC-30-161		S85DC-36-280		SD83GF-1-156 SD83GF-2-156
S84DC-40-134		S85DC-40-220		SD83LF-3-177 SD83LF-5-177
S84DC-40-149	304DC-40-140	S85DC-40-265		SD83LF-4-177 SD83LF-5-177
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S84DC-45-134		S85DC-50-265		SD84CC-5-144 SD84CC-20-144
S84DC-45-149		S85DC-50-280		SD84CC-5-178 SD84CC-20-178
S84DC-45-183		S85DC-52-250		SD84CC-6-135
S84DC-46-134		S85DC-52-265		SD84CC-17-178 SD84CC-20-178 SD84MC-3-152 SD84MC-18-152
S84DC-46-149		S85DC-52-280		SD84MC-3-176 SD84MC-18-176
S84DC-46-183		S85DC-55-250		SD84MC-3-176 SD84MC-18-176 SD84MC-4-152 SD84MC-18-152
S84DC-47-148		S85DC-55-265		SD84MC-4-176 SD84MC-18-176
		S85DC-55-280	S85DC-56-280	SD84MC-6-176 SD84MC-21-176
S85CC-7-197	S85DC-40-197	S85MB-2-197		SD84MC-8-152 SD84MC-18-152
S85CC-7-220				SD84MC-8-176 SD84MC-18-176
S85CC-8-197			S95DC-40-190 CM4	SD84MC-9-152 SD84MC-18-152
S85CC-9-200	S85DC-40-197 CM46		S95DC-40-220 CM4	SD84MC-9-176 SD84MC-18-176
	S85DC-40A-220 CM46		S95DC-40-235 CM4	SD84MC-11-406 SD84MC-20-406
	S85DC-40-235 CM46		S95DC-40-250 CM4	SD84MC-12-406 SD84MC-20-406
S85CC-22-183 S85CC-22-190			S95DC-40-220 CM4 S95DC-40-235 CM4	SD84MC-13-176 SD84MC-21-176
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S85CC-22-197	565DC-40-197	S95DC-30-250	595DC-40-250 CM4	SD84MC-16-406 SD84MC-23-406
S85CC-22-235		S95DC-30-300		
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	S85DC-40-235 CM4	S95DC-40-320		SD85DC-1-380
	S85DC-40-197 CM46	S95DC-41-260		SD85MC-3-221 SD85MC-15-220 SD85MC-4-221 SD85MC-15-220
	S85DC-40A-220 CM46			SD85MC-8-220 SD85MC-15-220
	S85DC-40-235 CM46	SD42CC-10-111	SD42CC-19-111	SD85MC-9-220 SD85MC-15-220
S85DB-1-250	S85DC-40-250 CM46	SD42CC-13-111		SD85MC-13-220 SD85MC-16-220
S85DB-2-235	S85DC-40-235 CM22	SD43CC-4-106	SD53CC-25-106	3D03WC-11-220 3D03WC-10-220
	S85DC-40-197 CM46	SD52CC-19-111	CDF0CC 04 444	SD94CC-1-176 SD94CC-7-176
S85DB-3-220	S85DC-40A-220 CM46			SD94CC-2-176 SD94CC-7-176
S85DB-3-235	S85DC-40-235 CM46	SD52DC-6-119 SD53CC-28-182		SD94CC-3-176 SD94CC-7-176
	S85DC-40-250 CM46	SD53CC-28-182 SD53CD-1-182		SD94CC-4-190 SD94CC-8-190
S85DC-7-200		3D33CD-1-10Z	JUJJUU-Z-10Z	000000 4 400
S85DC-7-220		SD62CB-10-79	SD62CB-11-79	SD95CC-1-160 SD95CC-3-160
S85DC-7-235		SD62CB-10-94		SD95MC-1-220 SD95MC-5-220
S85DC-8-220		SD62CB-10-114		SD95MC-2-220 SD95MC-6-220
S85DC-8-235		SD63CB-5-132	SD63CB-15-132	SD96CC-1-199SD96CC-3-199
S85DC-10-197	S85DC-40-197	SD63CB-5-167		SD96MC-6-412 SD96MC-5-412 SD96MC-6-456 SD96MC-5-456
S85DC-10-220		SD63CB-6-118		SD96MC-6-480 SD96MC-5-480
S85DC-11-197		SD63CB-11-118	SD63CB-14-118	3D30IVIO-0-400 3D30IVIO-3-460
S85DC-12-220	565DC-40A-220			



## **Quick Reference Guide**

CRYSTEEL		CUSTOM HOIST		CUSTOM HOIST	
107450	S73DC-41-129	63-2-156	S63DC-47-150	73-44-120S	S73DC-40-124
107455		63-4-104		73-50-130	
107465	S63DC-47-130 CM2	63-4-120		73-55-126	
107469		63-4-160			S73DC-40-124 CM38
T63113			S63DC-47-107 CM38		S73DC-40-140 CM38
	S63DC-47-130 CM2		S63DC-47-126 CM38		S73DC-40-110 CM26
T73129			S63DC-47-77 CM11	73-81-129	S73DC-40-124 CM26
T73147	S73DC-40-150		S63DC-47-82 CM11	73-108-130	S73DC-41-129
01107011110107			S63DC-47-86 CM11	74.4.440	07450 404 400
CUSTOM HOIST	00000 4 40 01470		S63DC-47-104 CM11	74-1-118	
32-29-43			S63DC-44-111 CM11	74-1-120 74-1-124S	
42-44-60 42-75-43			S63DC-44-123 CM11	74-1-1245	
43-2-74			S63DC-47-126 CM11	74-1-126	
43-5-72			S63DC-47-140 CM11	74-1-135	
40-0-12	343DB-2-73 CW72	63-10-120 63-10-127		74-1-137	
53-2-60S	S53DC-14-72 CM4		S63DC-45-126 S63DC-47-107 CM36	74-1-140S	
53-2-72	S53DC-14-72 CM4		S63DC-47-107 CM36	74-1-144	
53-2-72S	S53DC-14-72 CM4	63-27-121		74-1-146	
53-2-74	S53DC-14-72 CM4	63-27-125		74-1-160	
53-2-84		63-27-139		74-7-135	
53-2-88		63-27-160			S74DC-40A-120 CM11
53-2-90			S63DC-47-120 CM2		S74DC-40-135 CM11
53-3-120		63-44-130	S63DC-47-130 CM2		S74DC-40A-140 CM11
53-12-99			S63DC-47-140 CM2	74-12-120	S74DC-40A-120
53-12-122			S63DC-47-130 CM2	74-27-156	S74DC-40-156 CM4
53-52-120			S63DC-47-107 CM2	74-28-120	S74DC-40A-120
53-59-99		63-83-108	S63DC-47-107 CM38	74-28-137	
53-59-105			S63DC-47-126 CM38	74-28-161	
53-59-122			S63DC-47-107 CM17		S74DC-40A-120 CM4
53-72-60S		63-105-142	S63DC-47-140 CM17		S74DC-40-135 CM2
53-72-72 53-72-72\$			S63DC-47-140 CM17		S74DC-40A-140 CM4
	S53DC-14-72 CM4		S63DC-47-77 CM22	74-37-160	
	S53DC-14-72 CM4		S63DC-47-107 CM38	74-42-118	
53-72-88			S63DC-47-126 CM38	74-42-120	
53-72-90		63-180-104		74-42-124S	
53-73-89			S63DC-44-111 CM11	74-42-126 74-42-137	
53-80-120			S63DC-44-123 CM11	74-42-137	
53-97-63			S63DC-47-140 CM11	74-42-138	
53-117-60		63-186-121 63-186-125		74-42-144	
		63-186-139		74-42-146	
	S63DC-47-130 CM43	63-186-160		74-42-160	
	S63DC-47-130 CM43		S63DC-47-130 CM2	74-65-154	
63-2-72S			S63DC-47-140 CM2	74-65-170	
63-2-74S		63-202-121			
63-2-84		63-202-125		83-4-140	
63-2-84S		63-202-139		84-2-150	
63-2-87 63-2-88		63-402-132	S63DC-47-130 CM2	84-2-160	
63-2-90		64-35-135	S64DB-6-135	84-2-165	
63-2-92S		64-35-156	S64DB-6-156	84-2-174	
	S63DC-47-104 CM2		0	84-2-181	
	S63DC-47-104 CM2		S73DC-40-124 CM23	84-2-190	
	S63DC-47-104 CM2		S73DC-40-140 CM23	84-4-140 84-4-166	
	S63DC-47-104 CM2	73-2-120		84-4-180	
	S63DC-47-107 CM2	73-3-120 73-3-130			S84DC-40-180 CM21
	S63DC-47-107 CM2	73-3-130 73-3-140		84-13-135S	
	S63DC-47-120 CM2		\$73DC-40-140 \$73DC-40-124 CM11	84-13-148	
63-2-126			\$73DC-40-124 CM11	84-13-150	
	S63DC-47-130 CM2	73-15-130		84-13-150S	
63-2-132	S63DC-47-130 CM2	73-15-135		84-13-156S	
	S63DC-47-130 CM2		S73DC-41-129	84-13-160	
	S63DC-47-140 CM2		S73DC-40-120 CM4	84-13-166	
63-2-150	S63DC-47-150	73-26-120		84-13-190	
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84-36-150S	S84DC-31-156	95-4-285		DA7-182-108 D70CC-8-108
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		95-11-200		DAT42-19-112 SD42CC-19-111
85-2-166		95-11-235	S95DC-40-235	DAT42-58-112 SD42CC-19-111
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85-2-200		95-11-260	S95DC-40-265	DAT52-9-119 SD52DC-8-119
85-2-200S		95-11-260S	S95DC-40-265	DAT52-24-112 SD52CC-24-111
85-2-225		95-11-264S	S95DC-40-265	DAT52-32-119 SD52DC-8-119
85-2-235		95-11-265	S95DC-40-265	DAT52-58-119 SD52DC-8-119
85-2-265		95-11-265S	S95DC-40-265	DAT53-8-110 SD53CC-16-111
85-2-285		95-11-280	S95DC-40-280	DAT53-105-110 SD53CC-16-111
	S85DC-40-197 CM46	95-11-285	S95DC-40-280	DAT53-117-182 SD53CD-2-182
	S85DC-40-235 CM46	95-11-300		DAT53-125-182 SD53CC-31-182
	S85DC-40A-265 CM46	95-21-190S		DAT53-130-110 SD53CC-16-111
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DAT63-145-118 SD63CB-14-118	DAT85-72-378 SD85CC-19-378	DAT9 3/4 6-7-199 SD96CC-3-199
DAT63-146-120 SD63DB-14-120	DAT85-93-220 SD85MC-16-220	DAT9 3/4 6-8-199 SD96CC-3-199
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DAT63-182-108 SD63DB-6-108	DAT95-34-220 SD95MC-6-220	AB17404 SD85CC-19-348
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DAT63-209-118 SD63CB-14-118	DAT95-36-220 SD95MC-5-220	AB17924 SD63CC-11-96
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DAT73-20-120 SD73CC-25-120	DAT96-6-416 SD96MC-5-412	AB26635TH C3810
DAT74-6-131 SD74CC-11-131	DAT96-6-456SD96MC-5-456	AB28805 SD63CD-1-177
DAT74-11-144 SD74DB-6-144	DAT96-10-200 SD96CC-3-199	BB6087 SD63CC-11-96
DAT74-15-144 SD74DB-6-144	DAT96-15-412 SD96MC-5-412	BB6089 SD74CC-11-131 BB6199 SD63CC-11-96
DAT74-21-171 SD74DB-4-171	DAT96-15-456 SD96MC-5-456	BB6200SD74CC-11-131
DAT74-22-144 SD74DB-6-144	DAT96-15-480 SD96MC-5-480	BB6201SD74CC-11-131
DAT74-28-163 SD74CC-14-163	DAT96-16-412 SD96MC-5-412	BB6888
DAT74-57-144 SD74DB-6-144	DAT96-16-456 SD96MC-5-456	BB6996
DAT74-72-132 SD74CC-11-131	DAT96-16-480 SD96MC-5-480	BB6997
DAT74-73-144 SD74DB-6-144	DAT96-17-200 SD96CC-3-199	BB8112
DAT74-113-144 SD74DB-6-144	DAT96-26-199 SD96CC-3-199	BB8133-1
DAT83-6-126 SD83CC-27-126	DAT96-27-200 SD96CC-3-199	BB8134-1 C3805
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DAT84-13-176 SD84MC-21-176	DAT96-28-480 SD96MC-5-480	BB8235 C3805
DAT84-14-152 SD84MC-18-152	DAT96-29-456 SD96MC-5-456	BB8281-1 C3842
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DAT84-18-406 SD84MC-20-406	DAT96-37-412 SD96MC-5-412	BB9042 SD86CC-2-177
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DAT84-29-406 SD84MC-23-406	DAT96-38-456 SD96MC-5-456	CB5968 SD85CC-19-348
DAT84-38-145 SD84CC-20-144	DAT96-38-480 SD96MC-5-480	CB7015 SD53CC-16-111
DAT84-38-178 SD84CC-20-178	DAT96-41-412 SD96MC-5-412	CB7478 SD72AC-1-156
DAT84-43-145 SD84CC-20-144	DAT96-41-456 SD96MC-5-456 DAT96-41-480 SD96MC-5-480	CB7599 SD96CC-3-199
DAT84-43-178 SD84CC-20-178	DAT96-45-412SD96MC-5-412	CB7661 SD72AC-1-156
DAT84-48-176 SD84MC-21-176	DAT96-45-412 SD96MC-5-412 DAT96-45-456 SD96MC-5-456	CB7770 SD63CC-11-96
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DAT84-70-406 SD84MC-23-406	DAT96-48-412SD96MC-5-412	CB7877 SD72AC-1-156
DAT85-9-384 SD85MC-20-384	DAT96-48-456SD96MC-5-456	CB7947 SD72AC-1-156
DAT85-11-348 SD85CC-19-348	DAT96-48-480 SD96MC-5-480	CB8217 SD96CC-3-199
DAT85-15-384 SD85MC-20-384	DAT96-49-456 SD96MC-5-456	CB8321 SD85CC-19-348
DAT85-18-380 SD85DC-2-380	DAT96-50-412SD96MC-5-412	DB1748 SD96CC-3-199
DAT85-25-348 SD85CC-19-348	DAT96-50-456 SD96MC-5-456	DB4753 SD96CC-3-199
DAT85-26-220 SD85MC-16-220	DAT96-50-480 SD96MC-5-480	DB5801 SD96CC-3-199 DB7694 D3838
DAT85-27-220 SD85MC-15-220	DAT96-52-412 SD96MC-5-412	DB7694 D3838 DB8042 C3805
DAT85-29-384 SD85MC-20-384	DAT96-52-456 SD96MC-5-456	DB8042 C3805 DB8043 C3810
DAT85-45-384 SD85MC-20-384	DAT96-59-170 SD86CC-1-171	DB8044
DAT85-46-220 SD85MC-16-220		DB8045SD94CC-9-190
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## **Quick Reference Guide**

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A277 D60DB-7-72	ECB-2353-7 S85DC-40A-220 CM	MHC-030665 S95DC-40-190 CM42
A3525 D70LN-2-80	ECB-2353-8 S85DC-40-190 CM2	MHC-030666 S84DC-40-170
A6430 D60DB-7-72	ECB-2353-9 S85DC-40-250	MHC-030667 SD63DB-14-120
	ECB-2353-10 S85DC-40-170	MHC-030668 \$95DC-40-220 CM42
EDBRO	ECB-2353-12 S84DC-31-156	MHC-030671 S74DC-40-135 CM4
058003223 S53DC-11-104	ECB-2353-13 \$73DC-40-120 CM4	MHC-030673 S63MB-8-140 MHC-030676 S85DC-40A-265
058003247 S53DC-11-120 058003401 S63DC-47-107	ECB-2353-14 S74DC-40-156 CM4 ECB-2353-15 S63DC-47-120 CM36	
058003440 S63DC-44-111	ECB-2353-17 S84DC-47-120 CM36	MHC-030680 S74DC-41-133 MHC-030682 S74DC-40A-120 CM4
058003488 S63DC-47-140	ECB-2353-18 S74DC-40A-140 CM	MHC-030684 S95DC-40-250 CM42
058003820 S85DC-40-235	ECB-2353-19 S84DC-40-148 CM2	MHC-030685 S63MB-11-108
TE11-3-2660 S53DC-11-104	ECB-2353-20 S63DC-47-107 CM36	
TE11-3-2960 S53DC-11-120	ECB-2353-21 S63DC-47-104 CM36	MHC-030692 \$73DC-40-140 CM4
TE14-3-2670 S63DC-47-107	ECB-2353-22 S84DC-40-140	
TE14-3-2850 S63DC-44-111	ECB-2353-23 S95DC-40-250	TWC-1391-X S85DC-40-235
TE14-3-3450 S63DC-47-140	ECB-2353-24 \$95DC-40-265	TWC-1391-1 S85DC-40A-220
TE18-5-6170 S85DC-40-235	ECB-2353-25 S85DC-40-250	TWC-1391-2 S85DC-40-190
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<b>FONTAINE</b> 5385 S53DB-13-90	ECB-2353-27S85DC-40A-220 ECB-2353-28S95DC-40-250	TWC-1391-5 S85DC-40-250 CM2 TWC-1391-6 S85DC-40-235 CM2
53110 S53DB-13-90	ECB-2353-26 \$95DC-40-250 ECB-2353-29 \$85DC-40A-265	TWC-1391-6 S85DC-40-235 CM2
6385 S63DC-47-86 CM17	ECB-2353-30 S85DC-40-235 CM2	TWC-1391-8 S85DC-40-190 CM2
6397 S63DC-47-92 CM17	ECB-2353-31 S85DC-40-250 CM2	TWC-1391-9 S85DC-40-250
63110 S63DC-47-107 CM17	ECB-2353-32 S85DC-40A-265 CM	
63126 S63DC-44-123 CM63	ECB-2353-33 S95DC-40-265	TWC-1391-12 S84DC-31-156
63142 S63DC-47-140 CM17	ECB-2353-35 S96DC-1-285	TWC-1391-13 S73DC-40-120 CM4
73110 S73DC-40-110 CM20	ECB-2353-37 S85DC-40A-265 CM	
73115 \$73DC-40-110 CM20	ECB-2353-38 S84DC-40-148	TWC-1391-15 \$63DC-47-120 CM36
73126 \$73DC-40-124 CM20	ECB-2353-39 S85DC-40A-265	TWC-1391-17 S84DC-40-140
73142 \$73DC-40-140 CM20	ECB-2353-41 \$63DC-47-120 CM36	
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84164 S84DC-31-156 CM20	ECB-2353-43 S85DC-40-190 CM2	
84172 S84DC-40-170 CM20	ECB-2353-45 S96DC-1-285	TWC-1391-22 S84DC-40-140
84188 S84DC-40-180 CM20	ECB-2353-46 S95DC-40-250 CM42	
85172 S85DC-51-160 CM18		TWC-1391-29 S85DC-40A-265
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85197 S85DC-40-190 CM53	ECB-2353-48 S95DC-40-235	TWC-1392-X S85DC-40-235
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85197 S85DC-40-190 CM53 DCH-31-351 S85DC-40-190 CM53 DCH-31-359 S85DC-51-160 CM18	ECB-2353-48	TWC-1392-X S85DC-40-235 TWC-1392-2 S85DC-40-190 TWC-1392-3 S84DC-40-148
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85197       S85DC-40-190 CM53         DCH-31-351       S85DC-40-190 CM53         DCH-31-359       S85DC-51-160 CM18         DCH-31-374       S84DC-31-156 CM20         DCH-31-388       S84DC-40-148 CM20         DCH-31-413       S84DC-40-170 CM20         DCH-31-427       S73DC-40-140 CM20         DCH-31-433       S73DC-40-124 CM20         DCH-31-439       S73DC-40-110 CM20         DCH-31-462       S84DC-40-140 CM20         DCH-31-476       S63DC-47-140 CM17         DCH-31-483       S63DC-44-123 CM63	ECB-2353-48       S95DC-40-235         ECB-2353-52       S95DC-40-280         ECB-2353-53       S95DC-40-250 CM42         ECB-2353-54       S85DC-40-190         ECB-2353-55       S85DC-40A-220         ECB-2353-56       S85DC-40-190 CM2         ECB-2353-57       S85DC-40A-220 CM2         ECB-2353-101       S85DC-40A-220         ECB-2353-706       S85DC-40-235         ECC-3366-1       S63DC-47-107 CM16         ECC-3366-2       S63DC-47-120 CM16	TWC-1392-X
85197       S85DC-40-190 CM53         DCH-31-351       S85DC-40-190 CM53         DCH-31-359       S85DC-51-160 CM18         DCH-31-374       S84DC-31-156 CM20         DCH-31-388       S84DC-40-148 CM20         DCH-31-413       S84DC-40-170 CM20         DCH-31-427       S73DC-40-140 CM20         DCH-31-433       S73DC-40-110 CM20         DCH-31-439       S73DC-40-110 CM20         DCH-31-462       S84DC-40-140 CM20         DCH-31-476       S63DC-47-140 CM17         DCH-31-483       S63DC-44-123 CM63         DCH-31-487       S63DC-47-107 CM17	ECB-2353-48	TWC-1392-X
85197       S85DC-40-190 CM53         DCH-31-351       S85DC-40-190 CM53         DCH-31-359       S85DC-51-160 CM18         DCH-31-374       S84DC-31-156 CM20         DCH-31-388       S84DC-40-148 CM20         DCH-31-413       S84DC-40-170 CM20         DCH-31-427       S73DC-40-140 CM20         DCH-31-433       S73DC-40-110 CM20         DCH-31-439       S73DC-40-110 CM20         DCH-31-462       S84DC-40-140 CM20         DCH-31-476       S63DC-47-140 CM17         DCH-31-483       S63DC-44-123 CM63         DCH-31-487       S63DC-47-107 CM17         DCH-31-491       S63DC-47-92 CM17	ECB-2353-48       S95DC-40-235         ECB-2353-52       S95DC-40-280         ECB-2353-53       S95DC-40-250 CM42         ECB-2353-54       S85DC-40-190         ECB-2353-55       S85DC-40A-220         ECB-2353-56       S85DC-40-190 CM2         ECB-2353-57       S85DC-40A-220 CM2         ECB-2353-101       S85DC-40A-220         ECB-2353-706       S85DC-40-235         ECC-3366-1       S63DC-47-107 CM16         ECC-3366-2       S63DC-47-130 CM16         ECC-3366-3       S63DC-47-130 CM16	TWC-1392-X
85197       S85DC-40-190 CM53         DCH-31-351       S85DC-40-190 CM53         DCH-31-359       S85DC-51-160 CM18         DCH-31-374       S84DC-31-156 CM20         DCH-31-388       S84DC-40-148 CM20         DCH-31-413       S84DC-40-170 CM20         DCH-31-427       S73DC-40-140 CM20         DCH-31-433       S73DC-40-110 CM20         DCH-31-439       S73DC-40-110 CM20         DCH-31-462       S84DC-40-140 CM20         DCH-31-476       S63DC-47-140 CM17         DCH-31-483       S63DC-47-107 CM17         DCH-31-491       S63DC-47-92 CM17         DCH-31-495       S63DC-47-86 CM17	ECB-2353-48       S95DC-40-235         ECB-2353-52       S95DC-40-280         ECB-2353-53       S95DC-40-250 CM42         ECB-2353-54       S85DC-40-190         ECB-2353-55       S85DC-40-190 CM2         ECB-2353-56       S85DC-40-190 CM2         ECB-2353-57       S85DC-40A-220 CM2         ECB-2353-101       S85DC-40A-220         ECB-2353-706       S85DC-40-235         ECC-3366-1       S63DC-47-107 CM16         ECC-3366-2       S63DC-47-120 CM16         ECC-3366-3       S63DC-47-130 CM16         ECE-7681-1       S84DC-31-156	TWC-1392-X
85197       S85DC-40-190 CM53         DCH-31-351       S85DC-40-190 CM53         DCH-31-359       S85DC-51-160 CM18         DCH-31-374       S84DC-31-156 CM20         DCH-31-388       S84DC-40-148 CM20         DCH-31-413       S84DC-40-170 CM20         DCH-31-427       S73DC-40-140 CM20         DCH-31-433       S73DC-40-124 CM20         DCH-31-499       S73DC-40-110 CM20         DCH-31-462       S84DC-40-140 CM20         DCH-31-476       S63DC-47-140 CM17         DCH-31-483       S63DC-44-123 CM63         DCH-31-487       S63DC-47-107 CM17         DCH-31-491       S63DC-47-86 CM17         DCH-31-499       S53DC-11-107 CM27	ECB-2353-48       S95DC-40-235         ECB-2353-52       S95DC-40-280         ECB-2353-53       S95DC-40-250 CM42         ECB-2353-54       S85DC-40-190         ECB-2353-55       S85DC-40A-220         ECB-2353-56       S85DC-40-190 CM2         ECB-2353-57       S85DC-40A-220 CM3         ECB-2353-101       S85DC-40A-220         ECB-2353-706       S85DC-40-235         ECC-3366-1       S63DC-47-107 CM16         ECC-3366-2       S63DC-47-130 CM16         ECC-3366-3       S63DC-47-130 CM16         ECE-7681-1       S84DC-31-156         ECE-7681-2       S73DC-40-120 CM4	TWC-1392-X
85197       S85DC-40-190 CM53         DCH-31-351       S85DC-40-190 CM53         DCH-31-359       S85DC-51-160 CM18         DCH-31-374       S84DC-31-156 CM20         DCH-31-388       S84DC-40-148 CM20         DCH-31-413       S84DC-40-170 CM20         DCH-31-427       S73DC-40-140 CM20         DCH-31-433       S73DC-40-124 CM20         DCH-31-439       S73DC-40-110 CM20         DCH-31-462       S84DC-40-140 CM20         DCH-31-476       S63DC-47-140 CM17         DCH-31-483       S63DC-47-107 CM17         DCH-31-491       S63DC-47-107 CM17         DCH-31-495       S63DC-47-86 CM17         DCH-31-499       S53DC-11-107 CM27         DCH-31-507       S53DB-13-90	ECB-2353-48       S95DC-40-235         ECB-2353-52       S95DC-40-280         ECB-2353-53       S95DC-40-250 CM42         ECB-2353-54       S85DC-40-190         ECB-2353-55       S85DC-40-190 CM2         ECB-2353-56       S85DC-40-190 CM2         ECB-2353-57       S85DC-40A-220 CM3         ECB-2353-101       S85DC-40A-220         ECB-2353-706       S85DC-40-235         ECC-3366-1       S63DC-47-107 CM16         ECC-3366-2       S63DC-47-130 CM16         ECC-3366-3       S63DC-47-130 CM16         ECE-7681-1       S84DC-31-156         ECE-7681-2       S73DC-40-120 CM4         ECE-7681-3       S74DC-40-156 CM4	TWC-1392-X
85197       S85DC-40-190 CM53         DCH-31-351       S85DC-40-190 CM53         DCH-31-359       S85DC-51-160 CM18         DCH-31-374       S84DC-31-156 CM20         DCH-31-388       S84DC-40-148 CM20         DCH-31-413       S84DC-40-170 CM20         DCH-31-427       S73DC-40-140 CM20         DCH-31-433       S73DC-40-124 CM20         DCH-31-439       S73DC-40-110 CM20         DCH-31-462       S84DC-40-140 CM20         DCH-31-476       S63DC-47-140 CM17         DCH-31-483       S63DC-47-107 CM17         DCH-31-491       S63DC-47-92 CM17         DCH-31-495       S63DC-47-86 CM17         DCH-31-499       S53DC-11-107 CM27         DCH-31-507       S53DB-13-90         DCH-31-519       S84DC-40-180 CM20	ECB-2353-48	TWC-1392-X
85197       S85DC-40-190 CM53         DCH-31-351       S85DC-40-190 CM53         DCH-31-359       S85DC-51-160 CM18         DCH-31-374       S84DC-31-156 CM20         DCH-31-388       S84DC-40-148 CM20         DCH-31-413       S84DC-40-170 CM20         DCH-31-427       S73DC-40-140 CM20         DCH-31-433       S73DC-40-124 CM20         DCH-31-439       S73DC-40-110 CM20         DCH-31-462       S84DC-40-140 CM20         DCH-31-476       S63DC-47-140 CM17         DCH-31-483       S63DC-47-107 CM17         DCH-31-491       S63DC-47-107 CM17         DCH-31-495       S63DC-47-86 CM17         DCH-31-499       S53DC-11-107 CM27         DCH-31-507       S53DB-13-90	ECB-2353-48	TWC-1392-X
85197       S85DC-40-190 CM53         DCH-31-351       S85DC-40-190 CM53         DCH-31-359       S85DC-51-160 CM18         DCH-31-374       S84DC-31-156 CM20         DCH-31-388       S84DC-40-148 CM20         DCH-31-413       S84DC-40-170 CM20         DCH-31-427       S73DC-40-140 CM20         DCH-31-433       S73DC-40-124 CM20         DCH-31-439       S73DC-40-110 CM20         DCH-31-462       S84DC-40-140 CM20         DCH-31-476       S63DC-47-140 CM17         DCH-31-483       S63DC-47-107 CM17         DCH-31-491       S63DC-47-92 CM17         DCH-31-495       S63DC-47-86 CM17         DCH-31-499       S53DC-11-107 CM27         DCH-31-507       S53DB-13-90         DCH-31-519       S84DC-40-180 CM20	ECB-2353-48	TWC-1392-X
85197       S85DC-40-190 CM53         DCH-31-351       S85DC-40-190 CM53         DCH-31-359       S85DC-51-160 CM18         DCH-31-374       S84DC-31-156 CM20         DCH-31-388       S84DC-40-148 CM20         DCH-31-413       S84DC-40-170 CM20         DCH-31-427       S73DC-40-140 CM20         DCH-31-433       S73DC-40-110 CM20         DCH-31-439       S73DC-40-110 CM20         DCH-31-462       S84DC-40-140 CM20         DCH-31-476       S63DC-47-140 CM17         DCH-31-483       S63DC-47-140 CM17         DCH-31-491       S63DC-47-107 CM17         DCH-31-495       S63DC-47-92 CM17         DCH-31-499       S53DC-11-107 CM27         DCH-31-507       S53DB-13-90         DCH-31-519       S84DC-40-180 CM20         DCH-31-536       S73DC-40-110 CM20         FRUEHAUF         251147       S84DC-40-170	ECB-2353-48	TWC-1392-X S85DC-40-235 TWC-1392-2 S85DC-40-190 TWC-1392-3 S84DC-40-148 TWC-1392-9 S85DC-40-250 TWC-1411-1 S84DC-31-156 TWC-1411-2 S73DC-40-120 CM4 TWC-1411-3 S74DC-40-156 CM4 TWC-1411-5 S84DC-47-120 CM36 TWC-1411-5 S84DC-47-120 CM36 TWC-1411-6 S74DC-40-135 CM4 TWC-1411-7 S63DC-47-107 CM36  GALBREATH 1039 D60DB-7-72 1043 D60DB-7-72 1111 SD63CB-14-118 1126 D60DB-7-72 2419 SD62CB-11-114 2420 SD62CB-11-79 4 2460 D70LN-2-80 5 2494 SD62CB-11-94
85197       S85DC-40-190 CM53         DCH-31-351       S85DC-40-190 CM53         DCH-31-359       S85DC-51-160 CM18         DCH-31-374       S84DC-31-156 CM20         DCH-31-388       S84DC-40-148 CM20         DCH-31-413       S84DC-40-170 CM20         DCH-31-427       S73DC-40-140 CM20         DCH-31-439       S73DC-40-124 CM20         DCH-31-439       S73DC-40-110 CM20         DCH-31-462       S84DC-40-140 CM17         DCH-31-476       S63DC-47-140 CM17         DCH-31-483       S63DC-47-107 CM17         DCH-31-491       S63DC-47-92 CM17         DCH-31-495       S63DC-47-86 CM17         DCH-31-499       S53DC-11-107 CM27         DCH-31-519       S84DC-40-180 CM20         DCH-31-536       S73DC-40-110 CM20         FRUEHAUF         251147       S84DC-40-170         ECA-8950       S85MC-48-220	ECB-2353-48       S95DC-40-235         ECB-2353-52       S95DC-40-280         ECB-2353-53       S95DC-40-250 CM42         ECB-2353-54       S85DC-40-190         ECB-2353-55       S85DC-40-190 CM2         ECB-2353-56       S85DC-40-190 CM2         ECB-2353-101       S85DC-40A-220 CM2         ECB-2353-101       S85DC-40-235         ECC-3366-1       S63DC-47-107 CM16         ECC-3366-2       S63DC-47-120 CM16         ECC-3366-3       S63DC-47-130 CM16         ECE-7681-1       S84DC-31-156         ECE-7681-2       S73DC-40-120 CM4         ECE-7681-3       S74DC-40-156 CM4         ECE-7681-5       S84DC-40-140         ECE-7681-6       S74DC-40-140 CM-140         ECE-7681-7       S63DC-47-107 CM36         ECE-7681-8       S74DC-40A-140 CM-140 CM-140         ECE-7681-8       S74DC-40A-140 CM-140         ECE-7681-8       S74DC-40A-140 CM-140 CM-140	TWC-1392-X S85DC-40-235 TWC-1392-2 S85DC-40-190 TWC-1392-3 S84DC-40-148 TWC-1392-9 S85DC-40-250 TWC-1411-1 S84DC-31-156 TWC-1411-2 S73DC-40-120 CM4 TWC-1411-3 S74DC-40-156 CM4 TWC-1411-5 S84DC-47-120 CM36 TWC-1411-5 S84DC-47-120 CM36 TWC-1411-6 S74DC-40-135 CM4 TWC-1411-7 S63DC-47-107 CM36  GALBREATH 1039 D60DB-7-72 1043 D60DB-7-72 1111 SD63CB-14-118 1126 D60DB-7-72 2419 SD62CB-11-114 2420 SD62CB-11-79 4 2460 D70LN-2-80 5 2494 SD62CB-11-94
85197       \$85DC-40-190 CM53         DCH-31-351       \$85DC-40-190 CM53         DCH-31-359       \$85DC-51-160 CM18         DCH-31-374       \$84DC-31-156 CM20         DCH-31-388       \$84DC-40-148 CM20         DCH-31-413       \$84DC-40-170 CM20         DCH-31-427       \$73DC-40-140 CM20         DCH-31-439       \$73DC-40-110 CM20         DCH-31-439       \$73DC-40-110 CM20         DCH-31-462       \$84DC-40-140 CM17         DCH-31-476       \$63DC-47-140 CM17         DCH-31-483       \$63DC-47-107 CM17         DCH-31-491       \$63DC-47-92 CM17         DCH-31-495       \$63DC-47-86 CM17         DCH-31-499       \$53DC-11-107 CM27         DCH-31-507       \$53DB-13-90         DCH-31-536       \$73DC-40-110 CM20         FRUEHAUF         251147       \$84DC-40-170         ECA-8950       \$85MC-48-220         ECA-8951       \$85MC-48-235	ECB-2353-48	TWC-1392-X S85DC-40-235 TWC-1392-2 S85DC-40-190 TWC-1392-3 S84DC-40-148 TWC-1392-9 S85DC-40-250 TWC-1411-1 S84DC-31-156 TWC-1411-2 S73DC-40-120 CM4 TWC-1411-3 S74DC-40-156 CM4 TWC-1411-5 S84DC-47-120 CM36 TWC-1411-5 S84DC-47-120 CM36 TWC-1411-6 S74DC-40-135 CM4 TWC-1411-7 S63DC-47-107 CM36  GALBREATH 1039 D60DB-7-72 1043 D60DB-7-72 1111 SD63CB-14-118 1126 D60DB-7-72 1111 SD63CB-11-114 2420 SD62CB-11-114 2420 SD62CB-11-194 3116 SD62CB-11-94 3131 SD62CB-11-114
85197       S85DC-40-190 CM53         DCH-31-351       S85DC-40-190 CM53         DCH-31-359       S85DC-51-160 CM18         DCH-31-374       S84DC-31-156 CM20         DCH-31-388       S84DC-40-148 CM20         DCH-31-413       S84DC-40-170 CM20         DCH-31-427       S73DC-40-140 CM20         DCH-31-433       S73DC-40-110 CM20         DCH-31-439       S73DC-40-110 CM20         DCH-31-462       S84DC-40-140 CM20         DCH-31-476       S63DC-47-140 CM17         DCH-31-483       S63DC-47-107 CM17         DCH-31-491       S63DC-47-107 CM17         DCH-31-495       S63DC-47-86 CM17         DCH-31-499       S53DC-11-107 CM27         DCH-31-519       S84DC-40-180 CM20         DCH-31-536       S73DC-40-110 CM20         FRUEHAUF         251147       S84DC-40-170         ECA-8950       S85MC-48-220         ECA-8951       S85MC-48-235         ECA-9287       S85MC-48-200	ECB-2353-48	TWC-1392-X S85DC-40-235 TWC-1392-2 S85DC-40-190 TWC-1392-3 S84DC-40-148 TWC-1392-9 S85DC-40-250 TWC-1411-1 S84DC-31-156 TWC-1411-2 S73DC-40-120 CM4 TWC-1411-3 S74DC-40-156 CM4 TWC-1411-5 S84DC-47-120 CM36 TWC-1411-5 S84DC-47-120 CM36 TWC-1411-7 S63DC-47-107 CM36  GALBREATH 1039 D60DB-7-72 1043 D60DB-7-72 1111 SD63CB-11-114 1126 D60DB-7-72 1111 SD63CB-14-118 1126 D60DB-7-72 1111 SD63CB-11-114 2420 SD62CB-11-114 2420 SD62CB-11-114 2420 SD62CB-11-114 3131 SD62CB-11-94 3139 SD62CB-11-94 3139 SD62CB-11-114
85197       S85DC-40-190 CM53         DCH-31-351       S85DC-40-190 CM53         DCH-31-359       S85DC-51-160 CM18         DCH-31-374       S84DC-31-156 CM20         DCH-31-388       S84DC-40-148 CM20         DCH-31-413       S84DC-40-170 CM20         DCH-31-427       S73DC-40-140 CM20         DCH-31-433       S73DC-40-110 CM20         DCH-31-439       S73DC-40-110 CM20         DCH-31-462       S84DC-40-140 CM20         DCH-31-476       S63DC-47-140 CM17         DCH-31-483       S63DC-47-107 CM17         DCH-31-487       S63DC-47-107 CM17         DCH-31-491       S63DC-47-86 CM17         DCH-31-499       S53DC-11-107 CM27         DCH-31-507       S53DB-13-90         DCH-31-519       S84DC-40-180 CM20         DCH-31-536       S73DC-40-110 CM20         FRUEHAUF         251147       S84DC-40-170         ECA-8950       S85MC-48-220         ECA-8951       S85MC-48-235         ECA-9287       S85MC-48-200         ECB-2353 X       S85DC-40-235	ECB-2353-48	TWC-1392-X S85DC-40-235 TWC-1392-2 S85DC-40-190 TWC-1392-3 S84DC-40-148 TWC-1392-9 S85DC-40-250 TWC-1411-1 S84DC-31-156 TWC-1411-2 S73DC-40-120 CM4 TWC-1411-3 S74DC-40-156 CM4 TWC-1411-5 S84DC-47-120 CM36 TWC-1411-5 S84DC-47-120 CM36 TWC-1411-6 S74DC-40-135 CM4 TWC-1411-7 S63DC-47-107 CM36  GALBREATH 1039 D60DB-7-72 1043 D60DB-7-72 1111 SD63CB-14-118 1126 D60DB-7-72 1111 SD63CB-11-114 2420 SD62CB-11-79 12460 D70LN-2-80 12494 SD62CB-11-94 13131 SD62CB-11-94 13131 SD62CB-11-94 1319 SD62CB-11-114 13242 SD62CB-11-94 1319 SD62CB-11-114
85197       S85DC-40-190 CM53         DCH-31-351       S85DC-40-190 CM53         DCH-31-359       S85DC-51-160 CM18         DCH-31-374       S84DC-31-156 CM20         DCH-31-388       S84DC-40-148 CM20         DCH-31-413       S84DC-40-170 CM20         DCH-31-427       S73DC-40-140 CM20         DCH-31-433       S73DC-40-124 CM20         DCH-31-439       S73DC-40-110 CM20         DCH-31-462       S84DC-40-140 CM20         DCH-31-476       S63DC-47-140 CM17         DCH-31-487       S63DC-47-107 CM17         DCH-31-489       S63DC-47-92 CM17         DCH-31-491       S63DC-47-86 CM17         DCH-31-499       S53DC-11-107 CM27         DCH-31-507       S53DB-13-90         DCH-31-519       S84DC-40-180 CM20         DCH-31-536       S73DC-40-110 CM20         FRUEHAUF         251147       S84DC-40-170         ECA-8950       S85MC-48-220         ECA-8951       S85MC-48-235         ECA-9287       S85MC-48-200         ECB-2353 X       S85DC-40-235         ECB-2353-1       S85DC-40-220	ECB-2353-48	TWC-1392-X S85DC-40-235 TWC-1392-2 S85DC-40-190 TWC-1392-3 S84DC-40-148 TWC-1392-9 S85DC-40-250 TWC-1411-1 S84DC-31-156 TWC-1411-2 S73DC-40-120 CM4 TWC-1411-3 S74DC-40-156 CM4 TWC-1411-5 S84DC-47-120 CM36 TWC-1411-5 S84DC-47-120 CM36 TWC-1411-6 S74DC-40-135 CM4 TWC-1411-7 S63DC-47-107 CM36  GALBREATH 1039 D60DB-7-72 1043 D60DB-7-72 1111 SD63CB-14-118 1126 D60DB-7-72 1111 SD63CB-11-114 2420 SD62CB-11-79 4 2460 D70LN-2-80 5 2494 SD62CB-11-94 3116 SD62CB-11-94 3139 SD62CB-11-94 3139 SD62CB-11-14 3242 SD62CB-11-94 3412 D60DB-7-72 3902 SD62CB-11-114
85197       S85DC-40-190 CM53         DCH-31-351       S85DC-40-190 CM53         DCH-31-359       S85DC-51-160 CM18         DCH-31-374       S84DC-31-156 CM20         DCH-31-388       S84DC-40-148 CM20         DCH-31-413       S84DC-40-170 CM20         DCH-31-427       S73DC-40-140 CM20         DCH-31-433       S73DC-40-110 CM20         DCH-31-439       S73DC-40-110 CM20         DCH-31-462       S84DC-40-140 CM20         DCH-31-476       S63DC-47-140 CM17         DCH-31-483       S63DC-47-107 CM17         DCH-31-487       S63DC-47-92 CM17         DCH-31-491       S63DC-47-86 CM17         DCH-31-499       S53DC-11-107 CM27         DCH-31-507       S53DB-13-90         DCH-31-519       S84DC-40-180 CM20         DCH-31-536       S73DC-40-110 CM20         FRUEHAUF         251147       S84DC-40-170         ECA-8950       S85MC-48-220         ECA-8951       S85MC-48-235         ECA-9287       S85MC-48-200         ECB-2353-1       S85DC-40-235         ECB-2353-2       S85DC-40-190	ECB-2353-48	TWC-1392-X S85DC-40-235 TWC-1392-2 S85DC-40-190 TWC-1392-3 S84DC-40-148 TWC-1392-9 S85DC-40-250 TWC-1411-1 S84DC-31-156 TWC-1411-2 S73DC-40-120 CM4 TWC-1411-3 S74DC-40-156 CM4 TWC-1411-4 S63DC-47-120 CM36 TWC-1411-5 S84DC-40-140 TWC-1411-6 S74DC-40-135 CM4 TWC-1411-7 S63DC-47-107 CM36  GALBREATH 1039 D60DB-7-72 1043 D60DB-7-72 1111 SD63CB-14-118 1126 D60DB-7-72 1111 SD63CB-11-114 2420 SD62CB-11-79 4 2460 D70LN-2-80 5 2494 SD62CB-11-94 3 3131 SD62CB-11-14 3131 SD62CB-11-14 3131 SD62CB-11-14 3139 SD62CB-11-14 3139 SD62CB-11-94 3139 SD62CB-11-94 3412 D60DB-7-72 3902 SD62CB-11-114 8973 SD62CB-11-79
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70611-63-138			S74DC-40-135 CM23	093-1014-0	
70611-63-162			S84DC-40-140 CM66	093-1015-0	
70614-73-119	S73DC-40-120 CM38		S83DC-40-139 CM66	094-0008-0	
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70804-53-120T			S84DC-31-156 CM22	095-1026-0	
70805-64-156			S84DC-40-161 CM22	095-1027-0	
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097-2027-0		SA220-8-5-BC-60		C250-8-5	
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SA120-4.5-3-TT-49.25-		9000030		PM0351003VM0114.	
SA120-5.5-3-TT-54.62-		9000031		PM0400822VM0021.	
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SA130-6.5-3-TT-54.87-		9000035		PM0451003VM0174.	
SA130-6-3-TT-59.88-6.		9000036		PM0451003VM0115.	
SA140-5.5-3-TT-57.75-		9000037		PM0451203VM0117.	
SA140-6.5-3-TT-58.25-		9000038		PM0551103VM0175.	
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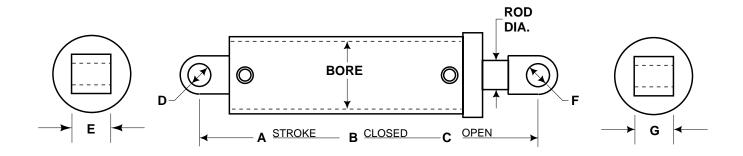
## **Quick Reference Guide**

MAILHOT		PERFECTION	
PM0551303VM0119 \$		A69070	
PM0551403VM0277 \$		A69071	
PM0601504VM0078 \$		A69072	
PM0651303VM0127 \$		A69073	
PM0651403VM0123 \$		A76479	
PM0651504VM0124 \$	S74MM-2G-150	A80314	
PM0651604VM0125 \$	S74MM-2G-160	A92097	
PM0651704VM0126 \$	S74MM-2G-170	A94464	
PM0651804VM0280 \$	S74MM-2G-180	A98278	
PM0701724VM0176 \$	S74MM-4G-172	A99108	
PM0701824VM0010 \$	S74MM-4G-182	A154781	
		A154875	
MARION		A154877	
3243C	S32DB-1-43 CM72	A154879	S85DC-40-170
3250C	S32DB-1-50 CM72		
4243C	S42DB-5-43 CM72	PRINCE	
4250C	S42DB-5-51 CM72	J1335	
4250T		S1062	
4375C	S43DB-2-75 CM72	S373	
5360C		S588	D70LN-2-80
5374T	S53MB-5-74		
5390C	S53DB-13-90	R & S BODY	
5390T		100586	
A218630-119		100610	
A218630-135	S74DC-40-135 CM26	100628	
		100655	
MAXON		100659	S95DC-40A-300 CM22
202104	C3831	100661	
222740		100665	
401216		101055	S95DC-40-250 CM22
401222	D60DD-4-69		
404614	C3694	TESCO WILLIAMSEN	
	C3694	<b>TESCO WILLIAMSEN</b> 327-01-100	
404614	C3694 C3831		S42MB-3-90
404614	C3694 C3831	327-01-100	S42MB-3-90 S53MB-3-120
404614	C3694 C3831 C3831	327-01-100 327-01-103 327-01-104 327-01-106	\$42MB-3-90 \$53MB-3-120 \$63MB-9-120 \$63MB-8-140
404614	C3694 C3831 C3831 S52MM-1G-82	327-01-100	\$42MB-3-90 \$53MB-3-120 \$63MB-9-120 \$63MB-8-140 \$74MB-3-154
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404614	C3694 C3831 C3831 S52MM-1G-82 S53MM-1G-100 S53MM-1G-120 S73MM-1G-140 S63MM-1G-140 S63MM-1G-140 S74MM-4G-182 S53DC-11-120 S63DC-47-107 S63DC-47-107 S63DC-47-120 S53DC-11-104 S53DC-11-120 S53DC-11-120 S53DC-11-120 S53DC-11-120	327-01-100	\$42MB-3-90 \$53MB-3-120 \$63MB-9-120 \$63MB-8-140 \$74MB-3-154 \$74MB-3-172 \$63MB-8-140 \$42MB-3-90 \$53MB-3-120 \$63MB-9-120 \$63MB-8-140 \$63MB-8-140 \$74MB-3-154 \$74MB-3-152 \$42MB-3-90 \$53MB-3-120 \$53MB-3-120 \$63MB-8-140 \$74MB-3-154
## A04614 ## A00000 ## A000000 ## A000000 ## A000000 ## A00000000	C3694 C3831 C3831 C3831 S52MM-1G-82 S53MM-1G-100 S53MM-1G-120 S73MM-1G-140 S63MM-1G-140 S63MM-1G-140 S74MM-4G-182 S53DC-11-120 S63DC-47-107 S63DC-47-107 S63DC-47-120 S53DC-11-120 S53DC-11-120 S53DC-11-84 S63DC-47-86	327-01-100	\$42MB-3-90 \$53MB-3-120 \$63MB-9-120 \$63MB-8-140 \$74MB-3-154 \$74MB-3-172 \$63MB-8-140 \$42MB-3-90 \$53MB-3-120 \$63MB-9-120 \$63MB-8-140 \$63MB-8-140 \$74MB-3-154 \$74MB-3-152 \$42MB-3-90 \$53MB-3-120 \$53MB-3-120 \$63MB-8-140 \$74MB-3-154
## A04614 ## A00000 ## A00000 ## A00000 ## A00000 ## A000000 ## A0000000 ## A00000000	C3694 C3831 C3831 C3831 S52MM-1G-82 S53MM-1G-100 S53MM-1G-120 S73MM-1G-140 S63MM-1G-140 S63MM-1G-140 S74MM-4G-182 S53DC-11-120 S63DC-47-107 S63DC-47-100 S53DC-11-104 S53DC-11-120 S53DC-11-84 S63DC-47-86 S63DC-47-86	327-01-100	\$42MB-3-90 \$53MB-3-120 \$63MB-9-120 \$63MB-8-140 \$74MB-3-154 \$74MB-3-172 \$63MB-8-140 \$42MB-3-90 \$53MB-3-120 \$63MB-9-120 \$63MB-8-140 \$63MB-8-140 \$74MB-3-154 \$74MB-3-152 \$42MB-3-90 \$53MB-3-120 \$53MB-3-120 \$63MB-8-140 \$74MB-3-154
404614	C3694 C3831 C3831 C3831 S52MM-1G-82 S53MM-1G-100 S53MM-1G-120 S73MM-1G-140 S63MM-1G-140 S63MM-1G-140 S74MM-4G-182 S53DC-11-120 S63DC-47-107 S63DC-47-107 S63DC-47-120 S53DC-11-120 S53DC-11-120 S53DC-11-84 S63DC-47-86 S63DC-47-86 S74DC-40-135	327-01-100	\$42MB-3-90 \$53MB-3-120 \$63MB-9-120 \$63MB-8-140 \$74MB-3-154 \$74MB-3-172 \$63MB-8-140 \$42MB-3-90 \$53MB-3-120 \$63MB-8-140 \$63MB-8-140 \$63MB-8-140 \$74MB-3-154 \$74MB-3-172 \$42MB-3-90 \$53MB-3-120 \$53MB-3-120 \$53MB-8-140 \$74MB-3-154 \$74MB-3-154 \$74MB-3-154
## A04614 ## A00000 ## A00000 ## A00000 ## A00000 ## A000000 ## A0000000 ## A00000000	C3694 C3831 C3831 C3831 S52MM-1G-82 S53MM-1G-100 S53MM-1G-120 S73MM-1G-140 S63MM-1G-140 S63MM-1G-140 S74MM-4G-182 S53DC-11-120 S63DC-47-107 S63DC-47-107 S63DC-47-120 S53DC-11-120 S53DC-11-120 S53DC-11-120 S53DC-14-84 S63DC-47-86 S63DC-47-86 S74DC-40-135 S53DC-14-72 CM69	327-01-100	\$42MB-3-90 \$53MB-3-120 \$63MB-9-120 \$63MB-8-140 \$74MB-3-154 \$74MB-3-172 \$63MB-8-140 \$42MB-3-90 \$53MB-3-120 \$63MB-9-120 \$63MB-8-140 \$74MB-3-154 \$74MB-3-172 \$42MB-3-172 \$42MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-172
## A04614 ## A00000 ## A00000 ## A00000 ## A00000 ## A000000 ## A0000000 ## A00000000	C3694 C3831 C3831 C3831 S52MM-1G-82 S53MM-1G-100 S53MM-1G-120 S73MM-1G-140 S63MM-1G-140 S63MM-1G-140 S74MM-4G-182 S53DC-11-120 S63DC-47-107 S63DC-47-107 S63DC-11-104 S53DC-11-104 S63DC-47-86 S63DC-47-86 S74DC-40-135 S53DC-14-72 CM69 S84DC-40-170	327-01-100	\$42MB-3-90 \$53MB-3-120 \$63MB-9-120 \$63MB-8-140 \$74MB-3-154 \$74MB-3-172 \$63MB-8-140 \$42MB-3-90 \$53MB-3-120 \$63MB-9-120 \$63MB-8-140 \$74MB-3-154 \$74MB-3-172 \$42MB-3-172 \$42MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-172
## A04614 ## A00000 ## A00000 ## A00000 ## A00000 ## A000000 ## A0000000 ## A00000000	C3694 C3831 C3831 C3831 S52MM-1G-82 S53MM-1G-100 S53MM-1G-120 S73MM-1G-140 S63MM-1G-140 S74MM-4G-182 S53DC-11-120 S63DC-47-107 S63DC-47-107 S63DC-11-104 S53DC-11-104 S53DC-11-120 S53DC-11-84 S63DC-47-86 S63DC-47-86 S74DC-40-135 S53DC-14-72 CM69 S84DC-40-170 S85DC-40-170	327-01-100	\$42MB-3-90 \$53MB-3-120 \$63MB-9-120 \$63MB-8-140 \$74MB-3-154 \$74MB-3-172 \$63MB-8-140 \$42MB-3-90 \$53MB-3-120 \$63MB-9-120 \$63MB-8-140 \$74MB-3-154 \$74MB-3-172 \$42MB-3-172 \$42MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-172
## A04614 ## A00000 ## A00000 ## A00000 ## A00000 ## A000000 ## A0000000 ## A00000000	C3694 C3831 C3831 C3831 S52MM-1G-82 S53MM-1G-100 S53MM-1G-120 S73MM-1G-140 S63MM-1G-140 S63MM-1G-140 S74MM-4G-182 S53DC-11-120 S63DC-47-107 S63DC-47-104 S53DC-11-104 S53DC-11-120 S53DC-11-120 S53DC-14-84 S63DC-47-86 S63DC-47-86 S74DC-40-135 S53DC-14-72 CM69 S84DC-40-170 S85DC-40-170 S95DC-40-220	327-01-100	\$42MB-3-90 \$53MB-3-120 \$63MB-9-120 \$63MB-8-140 \$74MB-3-154 \$74MB-3-172 \$63MB-8-140 \$42MB-3-90 \$53MB-3-120 \$63MB-9-120 \$63MB-8-140 \$74MB-3-154 \$74MB-3-172 \$42MB-3-172 \$42MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-172
## A04614 ## A00000 ## A00000 ## A00000 ## A00000 ## A000000 ## A0000000 ## A00000000	C3694 C3831 C3831 C3831 C3831 S52MM-1G-82 S53MM-1G-100 S53MM-1G-120 S73MM-1G-140 S63MM-1G-140 S63MM-1G-140 S74MM-4G-182 S53DC-11-120 S63DC-47-107 S63DC-47-104 S53DC-11-120 S53DC-11-120 S53DC-11-120 S53DC-14-84 S63DC-47-86 S63DC-47-86 S74DC-40-135 S53DC-14-72 CM69 S84DC-40-170 S85DC-40-170 S95DC-40-220 S95DC-40-220	327-01-100	\$42MB-3-90 \$53MB-3-120 \$63MB-9-120 \$63MB-8-140 \$74MB-3-154 \$74MB-3-172 \$63MB-8-140 \$42MB-3-90 \$53MB-3-120 \$63MB-9-120 \$63MB-8-140 \$74MB-3-154 \$74MB-3-172 \$42MB-3-172 \$42MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-172
## A04614 ## A00000 ## A000000 ## A000000 ## A000000 ## A00000000	C3694 C3831 C3831 C3831 S52MM-1G-82 S53MM-1G-100 S53MM-1G-120 S73MM-1G-140 S63MM-1G-140 S63MM-1G-140 S63MM-1G-140 S74MM-4G-182 S53DC-11-120 S63DC-47-107 S63DC-47-120 S53DC-11-104 S53DC-11-120 S53DC-11-120 S53DC-14-84 S63DC-47-86 S63DC-47-86 S63DC-47-86 S63DC-47-86 S74DC-40-170 S85DC-14-72 CM69 S84DC-40-170 S95DC-40-170 S95DC-40-220 S95DC-40-250 S84DC-40-140	327-01-100	\$42MB-3-90 \$53MB-3-120 \$63MB-9-120 \$63MB-8-140 \$74MB-3-154 \$74MB-3-172 \$63MB-8-140 \$42MB-3-90 \$53MB-3-120 \$63MB-9-120 \$63MB-8-140 \$74MB-3-154 \$74MB-3-172 \$42MB-3-172 \$42MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-172
## A04614 ## A06000 ## A06	C3694 C3831 C3831 C3831 C3831 S52MM-1G-82 S53MM-1G-100 S53MM-1G-120 S73MM-1G-140 S63MM-1G-140 S63MM-1G-140 S63MM-1G-140 S63DC-47-107 S63DC-47-107 S63DC-47-120 S53DC-11-120 S53DC-11-120 S53DC-11-120 S53DC-14-84 S63DC-47-86 S63DC-47-86 S63DC-47-86 S74DC-40-135 S53DC-14-72 CM69 S84DC-40-170 S95DC-40-220 S95DC-40-220 S95DC-40-250 S84DC-40-140 S63DC-47-130	327-01-100	\$42MB-3-90 \$53MB-3-120 \$63MB-9-120 \$63MB-8-140 \$74MB-3-154 \$74MB-3-172 \$63MB-8-140 \$42MB-3-90 \$53MB-3-120 \$63MB-9-120 \$63MB-8-140 \$74MB-3-154 \$74MB-3-172 \$42MB-3-172 \$42MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-172
## A04614 ## A04614 ## A04614 ## A04614 ## A06614 ## A06	C3694 C3831 C3831 C3831 S52MM-1G-82 S53MM-1G-100 S53MM-1G-120 S73MM-1G-140 S63MM-1G-140 S63MM-1G-140 S63MM-1G-140 S74MM-4G-182 S53DC-11-120 S63DC-47-107 S63DC-47-107 S63DC-47-120 S53DC-11-120 S53DC-11-120 S53DC-14-84 S63DC-47-86 S63DC-47-86 S74DC-40-135 S53DC-14-72 CM69 S84DC-40-170 S85DC-40-170 S85DC-40-170 S95DC-40-220 S95DC-40-250 S84DC-40-140 S63DC-47-130 S73DC-40-120	327-01-100	\$42MB-3-90 \$53MB-3-120 \$63MB-9-120 \$63MB-8-140 \$74MB-3-154 \$74MB-3-172 \$63MB-8-140 \$42MB-3-90 \$53MB-3-120 \$63MB-9-120 \$63MB-8-140 \$74MB-3-154 \$74MB-3-172 \$42MB-3-172 \$42MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-172
## A04614 ## A04614 ## A04614 ## A04614 ## A06614 ## A0666 ## A69066 ## A06004 ## A69066 ## A06004 ## A69066 ## A06004 ## A69066 ## ACC ##	C3694 C3831 C3831 C3831 S52MM-1G-82 S53MM-1G-100 S53MM-1G-120 S73MM-1G-140 S63MM-1G-140 S63MM-1G-140 S63MM-1G-140 S74MM-4G-182 S53DC-11-120 S63DC-47-107 S63DC-47-120 S53DC-11-120 S53DC-11-120 S53DC-11-120 S53DC-14-84 S63DC-47-86 S63DC-47-86 S74DC-40-170 S85DC-14-72 CM69 S84DC-40-170 S95DC-40-250 S95DC-40-250 S84DC-40-140 S63DC-47-130 S73DC-40-120 S74DC-40A-120	327-01-100	\$42MB-3-90 \$53MB-3-120 \$63MB-9-120 \$63MB-8-140 \$74MB-3-154 \$74MB-3-172 \$63MB-8-140 \$42MB-3-90 \$53MB-3-120 \$63MB-9-120 \$63MB-8-140 \$74MB-3-154 \$74MB-3-172 \$42MB-3-172 \$42MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-172
## A04614 ## A04614 ## A04614 ## A04614 ## A06614 ## A06	C3694 C3831 C3831 C3831 S52MM-1G-82 S53MM-1G-100 S53MM-1G-120 S73MM-1G-140 S63MM-1G-140 S63MM-1G-140 S63MM-1G-140 S74MM-4G-182 S53DC-11-120 S63DC-47-107 S63DC-47-120 S53DC-11-120 S53DC-11-120 S53DC-11-120 S53DC-14-84 S63DC-47-86 S63DC-47-86 S74DC-40-135 S53DC-14-72 CM69 S84DC-40-170 S95DC-40-220 S95DC-40-250 S84DC-40-170 S95DC-40-120 S73DC-40-120 S74DC-40-120 S74DC-40-135	327-01-100	\$42MB-3-90 \$53MB-3-120 \$63MB-9-120 \$63MB-8-140 \$74MB-3-154 \$74MB-3-172 \$63MB-8-140 \$42MB-3-90 \$53MB-3-120 \$63MB-9-120 \$63MB-8-140 \$74MB-3-154 \$74MB-3-172 \$42MB-3-172 \$42MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-120 \$53MB-3-172

Crossover and Application Data is for Reference Purpose Only. No Other Conditions are Expressed or Implied. Note: In some applications, modifications may be required for proper cylinder installation.

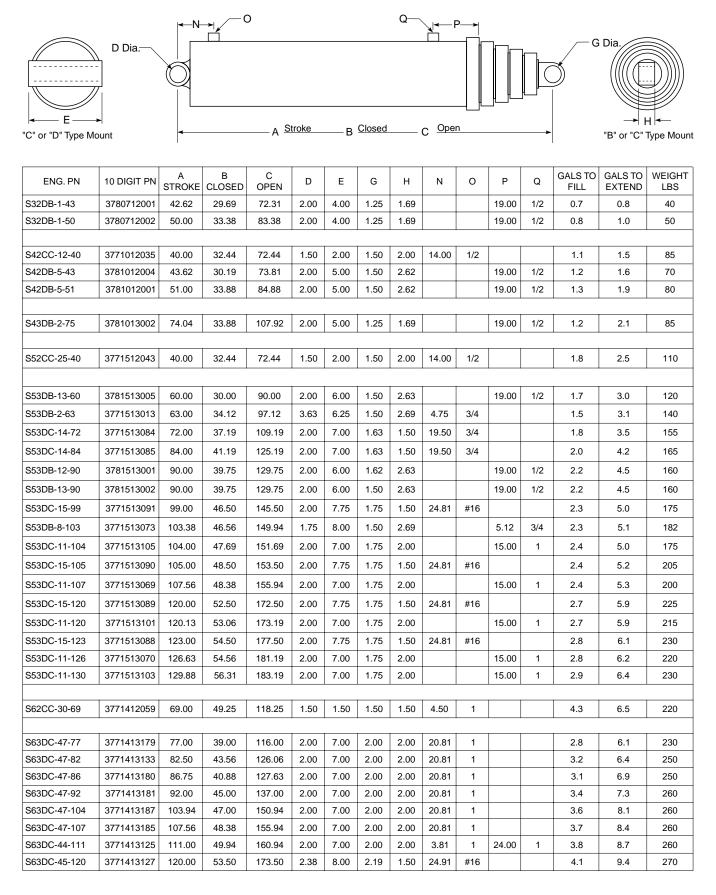


A69069 ...... S85DC-40-170



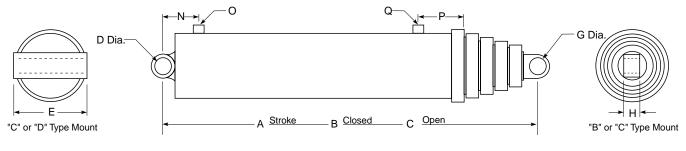
Cylinder Model	Cylinder Part Number	Bore Dia	Rod Dia	Straoke A	Closed B	Open C	Base D	Mount E	Plunger F	Mount G	Extend Port Size	Retract Port Size	Shpg. Wt.
C3603 C4438	C3603 C4438	3.00 3.00	1.50 1.50	11.00 25.25	23.12 34.25	34.12 59.50	1.00 1.00	2.00 1.00	1.00 1.13	2.00 2.00	#12 #8	#12 #8	115 55
C4438	C4438 C4439	3.00	1.50	25.25 25.25	34.25 34.25	59.50 59.50	1.00	0.75	1.13	2.00	#6 #6	#6 #6	55 55
B5162	B5162	3.00	1.75	28.19	38.00	66.19	1.00	1.00	1.13	1.50	#8	#8	75
C3602	C3602	3.00	1.50	30.50	43.88	74.38	1.00	1.50	1.00	1.50	#12	#12	70
B5347	B5347	3.00	2.50	31.25	38.75	70.00	1.00	1.50	1.00	0.88	"SA"	1/2	90
B3151	B3151	3.50	2.25	21.63	32.63	54.26	1.50	2.00	1.50	2.00	#12	#12	70
C3842	C3842	3.50	2.50	37.00	50.50	87.50	1.75	1.50	1.75	1.00	1/4	"S.A."	135
C3847	C3847	4.00	2.50	21.63	32.63	54.26	1.50	2.00	1.50	2.00	#12	#12	90
C3926	C3926	4.00	2.00	22.00	36.00	58.00	1.50	2.88	1.50	2.50	#10	#10	110
C3807	C3807	4.00	2.00	24.00	43.50	67.50	1.50	1.50	1.50	3.00	1/2	1/2	85
C3877	C3877	4.00	3.50	24.75	36.00	60.75	1.25	1.00	1.25	1.25	"SA"	1/2	95
C3831	C3831	4.00	2.50	29.75	39.00	68.75	1.50	4.50	1.38	2.50	3/4	3/4	95
C3863	C3863	4.00	2.50	35.31	51.44	86.75	1.25	2.75	1.25	2.50	3/8	3/8	130
C3850	C3850	4.50	2.00	26.50	40.62	67.12	1.50	2.00	1.50	2.50	#12	#12	105
B3152	B3152	4.50	2.00	26.94	40.63	67.57	1.50	2.00	1.50	2.00	#12	#12	105
C3933	C3933	4.50	2.50	41.50	59.38	100.88	2.00	2.00	2.00	2.88	#12	#12	183
C4622	C4622	4.50	2.50	41.50	57.00	98.50	2.00	2.00	THRD	2.25-12	#12	#12	170
C3843	C3843	5.00	3.00	20.00	33.00	53.00	2.00	2.00	2.00	2.00	#12	#12	130
C3805	C3805	5.00	3.00	27.00	39.00	66.00	1.75	3.00	1.75	3.00	3/4	3/4	137
C3872	C3872	5.00	2.50	36.00	49.25	85.25	2.00	1.88	2.63	1.88	#16	#16	185
C3813	C3813	5.00	2.50	43.75	57.62	101.37	2.00	2.00	THRD	2.25-12	3/4	3/4	175
C3694	C3694	5.00	2.50	51.00	64.50	115.50	1.75	2.50	1.75	2.50	1	1	225
C3563	C3563	5.50	4.00	54.13	75.00	129.13	1.25	1.25	1.25	2.00	#16	#16	305
C4793	C4793	5.50	4.00	63.00	81.00	144.00	2.00	3.00	2.00	3.00	#16	#16	260
D4088	D4088	5.50	4.00	63.00	81.12	144.12	2.00	6.50	2.00	3.88	#16	#16	245
C3810	C3810	6.00	2.50	34.00	51.00	85.00	1.75	3.00	1.75	3.00	1	1	205
C3814	C3814	6.00	2.50	34.00	48.50	82.50	1.75	3.00	THRD	2.50-12		1	176
D3855	D3855	6.00	4.00	63.00	75.00	138.00	2.00	7.00	2.00	3.88	1	1	300
D60DD-4-69	3771421090	6.00	3.75	69.00	84.31	153.31	3.50	7.00	2.75	5.00	1	1	390
D60DB-7-72	3771421147	6.00	4.50	72.00	90.00	162.00	2.50	7.00	2.50	4.38	#12	#12	400
D3838	D3838	6.00	3.00	76.00	89.62	165.62	3.50	7.25	3.50	3.50	#16	#16	356
D62DB-9-15	3771421118	6.25	2.00	15.00	26.69	41.69	2.00	7.00	0.75	2.00	3/4	3/4	121
D62DB-7-72	3771421102	6.25	4.75	72.00	90.00	162.00	2.50	7.00	2.50	4.38	#12	#12	410
D62LB-5-80	3771421180	6.25	4.75	80.00	98.75	178.75	2.50	6.53	2.50	4.69	3/4	3/4	320
D70LN-2-80	3773721025	7.00 7.00	3.00	80.00	92.50	172.50 244.12	SPL	LUGS 2.00	0.75	3.00	#12	#12	500
D70CC-8-108 D70CC-8-131	3773721018	7.00	5.50 5.50	108.00	136.12 159.12	290.12	2.00	2.00	2.00 2.00	2.00 2.00	1 1	1 1	455 690
D70CC-6-131	3773721012 3772521071	7.00 7.25	2.75	131.00 15.00	26.69	41.69	2.00	8.25	0.75	2.75	3/4	3/4	176
D72DB-7-15 D72DB-7-25	3772521071	7.25 7.25	2.75	25.00	36.69	61.69	2.00	8.25	0.75	2.75	3/4	3/4	228
D72DB-7-25-1		7.25 7.25	2.75	25.00	47.69	72.69	2.00	8.25	0.75	2.75	3/4	3/4	230
D72LB-9-80	3772521074	7.25 7.25	4.75	80.00	98.75	178.75	2.50	6.53	2.50	4.69	3/4	3/4	420
D84DB-7-25	3772921006	8.44	2.75	25.00	36.69	61.69	2.00	9.50	0.75	2.75	3/4	3/4	260
D84DB-7-25-1		8.44	2.75	25.00	47.69	72.69	2.00	9.50	0.75	2.75	3/4	3/4	280
DUTDU-1-20*1	0112021001	U. <del>74</del>	2.10	20.00	₹1.03	12.00	2.00	5.50	0.75	2.10	5/4	5/4	200

## Stock Single-Acting Telescopic Cylinders With CC, DB, and DC Mounts





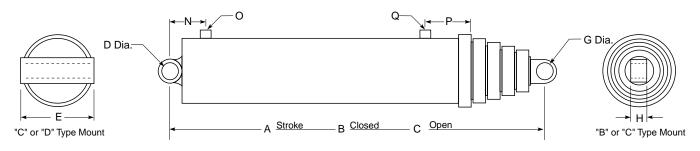
## Stock Single-Acting Telescopic Cylinders With CC, DB, and DC Mounts



ENG. PN	10 DIGIT PN	A STROKE	B CLOSED	C OPEN	D	Е	G	Н	N	0	Р	Q	GALS TO FILL	GALS TO EXTEND	WEIGHT LBS
S63DC-47-120	3771413183	120.06	52.62	172.68	2.00	7.00	2.00	2.00	20.81	1			4.1	9.4	270
S63DC-44-123	3771413126	123.00	54.94	177.94	2.00	7.00	2.00	2.00	3.81	1	24.00	1	4.2	9.6	270
S63DC-45-126	3771413128	126.00	55.50	181.50	2.38	8.00	2.19	1.50	24.91	#16			4.3	9.9	275
S63DC-47-126	3771413186	126.62	54.56	181.18	2.00	7.00	2.00	2.00	20.81	1			4.3	9.9	275
S63DC-47-130	3771413184	130.06	56.00	186.06	2.00	7.00	2.00	2.00	20.81	1			4.3	9.9	295
S63DC-45-138	3771413129	138.00	59.50	197.50	2.38	8.00	2.19	1.50	24.91	#16			4.6	11.1	300
S63DC-47-140	3771413149	140.25	59.81	200.06	2.00	7.00	2.00	2.00	20.81	1			4.7	11.1	295
S63DC-47-150	3771413182	150.00	63.00	213.00	2.00	7.00	2.00	2.00	20.81	1			5.0	11.9	310
S63DC-45-162	3771413130	162.00	67.50	229.50	2.38	8.00	2.19	1.50	24.91	#16			5.3	11.9	330
													•		
S64DB-6-135	3771414034	135.00	47.19	182.19	1.75	8.00	1.50	2.69			5.50	2, 3/4	3.4	8.1	260
S64DB-6-156	3771414035	156.00	53.19	209.19	1.75	8.00	1.50	2.69			5.50	2, 3/4	3.5	10.1	293
S64DC-9-156	3771414040	156.00	53.62	209.62	2.00	7.25	2.00	1.75	6.31	1			3.5	10.1	285
S64DC-7-157	3771414036	157.00	58.56	215.56	1.50	7.00	1.50	2.00	4.31	1			3.9	10.1	310
												l			
S73DC-40-110	3772513068	110.63	50.06	160.69	2.00	8.25	2.00	2.00	7.06	1	4.62	1	5.3	12.6	335
S73DC-40-120	3772513089	120.00	53.12	173.12	2.00	8.25	2.00	2.00	7.06	1	4.62	1	5.7	13.7	370
S73DC-40-124	3772513069	124.88	54.81	179.69	2.00	8.25	2.00	2.00	7.06	1	4.62	1	5.9	14.2	380
S73DC-41-129	3772513072	129.00	56.50	185.50	2.00	8.25	2.00	2.00	8.56	1	4.62	1	6.1	14.7	390
S73DC-40-140	3772513070	140.44	60.00	200.44	2.00	8.25	2.00	2.00	7.06	1	4.62	1	6.5	16.1	400
S73DC-40-150	3772513083	150.00	63.50	213.50	2.00	8.25	2.00	2.00	7.06	1	4.62	1	6.5	17.1	410
S74DC-40A-120	3772514056	120.00	44.12	164.12	2.00	8.25	2.00	2.00	7.06	1	4.62	1	4.1	11.5	335
S74DC-41-135	3772514052	135.00	48.31	183.31	2.00	8.25	1.63	1.50	18.31	3/4			4.5	13.1	370
S74DC-40-135	3772514047	135.00	48.44	183.44	2.00	8.25	2.00	2.00	7.06	1	4.62	1	4.5	13.1	370
S74DC-40A-140	3772514057	140.00	49.75	189.75	2.00	8.25	2.00	2.00	7.06	1	4.62	1	4.6	13.5	380
S74DC-40-156	3772514050	156.00	53.75	209.75	2.00	8.25	2.00	2.00	7.06	1	4.62	1	5.1	15.1	390
S74DC-40-161	3772514051	161.75	55.31	217.06	2.00	8.25	2.00	2.00	7.06	1	4.62	1	5.2	15.5	405
S74DC-40-180	3772514063	180.00	61.12	241.12	2.00	8.25	2.00	2.00	7.06	1	4.62	1	5.9	17.4	430
S83DC-26-134	3772913033	134.63	57.63	192.25	2.00	9.50	1.63	1.50	5.62	1			9.0	21.0	520
S83DC-40-139	3772913022	138.00	60.12	198.12	2.00	9.50	2.00	2.00	6.50	1	5.62	1	9.1	21.9	530
S83DC-51-150	3772913040	150.00	64.00	214.00	2.00	9.50	2.00	2.00	6.50	1	17.25	1	9.8	23.9	590
S83DC-40-160	3772913023	159.00	69.12	228.12	2.00	9.50	2.00	2.00	6.50	1	5.62	1	10.6	25.3	601



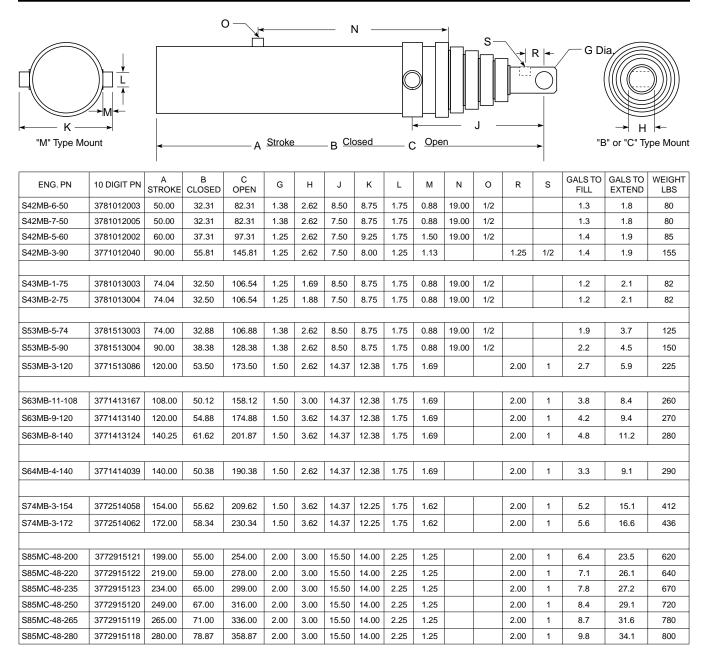
## Stock Single-Acting Telescopic Cylinders With CC, DB, and DC Mounts



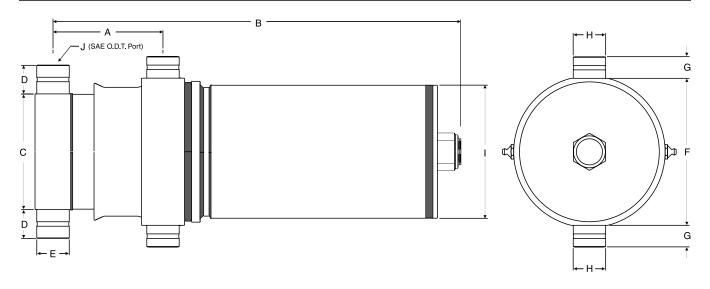
ENG. PN	10 DIGIT PN	A STROKE	B CLOSED	C OPEN	D	Е	G	Н	N	0	Р	Q	GALS TO FILL	GALS TO EXTEND	WEIGHT LBS
S84DC-40-140	3772914051	140.00	49.75	189.75	2.00	9.50	2.00	2.00	6.50	1	5.62	1	6.5	19.2	485
S84DC-40-148	3772914052	147.50	51.50	199.25	2.00	9.50	2.00	2.00	6.50	1	5.62	1	6.7	20.3	530
S84DC-31-156	3772914048	156.00	53.75	209.75	2.00	9.50	2.00	2.00	5.56	1	5.62	1	7.1	21.5	550
S84DC-40-161	3772914014	160.00	55.75	215.75	2.00	9.50	2.00	2.00	6.50	1	5.62	1	7.4	21.9	565
S84DC-40-170	3772914050	170.00	57.25	227.25	2.00	9.50	2.00	2.00	6.50	1	5.62	1	7.6	23.3	575
S84DC-40-180	3772914053	180.00	59.75	239.75	2.00	9.50	2.00	2.00	6.50	1	5.62	1	8.0	24.7	580
S84DC-40-188	3772914057	188.00	61.75	249.75	2.00	9.50	2.00	2.00	6.50	1	5.62	1	8.3	25.8	585
S85DC-51-160	3772915096	160.00	48.00	208.00	2.00	9.50	2.00	2.00	6.56	1	15.50	1	5.5	18.5	560
S85DC-40-170	3772915077	170.00	49.88	219.88	2.00	9.50	2.00	2.00	6.50	1	5.62	1	5.8	20.1	570
S85DC-40-190	3772915022	189.00	54.62	243.62	2.00	9.50	2.00	2.00	6.50	1	5.62	1	6.4	22.3	580
S85DC-40-197	3772915021	196.13	55.25	251.38	2.00	9.50	2.00	2.00	6.50	1	5.62	1	6.5	22.8	590
S85DC-40A-220	3772915081	220.00	60.00	280.00	2.00	9.50	2.00	2.00	6.50	1	5.62	1	7.2	25.5	610
S85DC-40-235	3772915019	234.00	64.62	298.62	2.00	9.50	2.00	2.00	6.50	1	5.62	1	7.8	27.2	660
S85DC-40-250	3772915018	249.00	68.62	317.62	2.00	9.50	2.00	2.00	6.50	1	5.62	1	8.4	29.1	705
S85DC-40A-265	3772915080	265.00	71.00	336.00	2.00	9.50	2.00	2.00	6.50	1	5.62	1	8.7	31.6	776
S85DC-40-285	3772915078	285.00	78.50	363.50	2.00	9.50	2.00	2.00	6.50	1	5.62	1	9.8	34.1	800
	•	•													
S95DC-40-190	3773115015	190.00	56.00	246.00	2.00	10.88	2.00	2.00	6.50	1	5.75	1	6.4	22.4	835
S95DC-40-220	3773115011	218.00	62.44	280.44	2.00	10.88	2.00	2.00	6.50	1	5.75	1	9.7	36.2	855
S95DC-40-235	3773115010	233.00	65.44	298.44	2.00	10.88	2.00	2.00	6.50	1	5.75	1	10.2	38.7	880
S95DC-40-250	3773115009	248.00	68.44	316.44	2.00	10.88	2.00	2.00	6.50	1	5.75	1	10.8	41.3	1100
S95DC-40-265	3773115013	265.00	72.62	337.62	2.00	10.88	2.00	2.00	6.50	1	5.75	1	11.6	44.6	1160
S95DC-40-280	3773115016	280.00	78.62	358.62	2.00	10.88	2.00	2.00	6.50	1	5.75	1	12.8	47.6	1200
S95DC-40A-300	3773115022	300.00	79.00	379.00	2.00	10.88	2.00	2.00	6.50	1	5.75	1	13.5	50.2	1255
S95DC-40A-320	3773115023	320.00	83.00	403.00	2.00	10.88	2.00	2.00	6.50	1	5.75	1	13.9	54.1	1300
S95DC-40A-340	3773115040	340.00	87.00	427.00	2.00	10.88	2.00	2.00	6.50	1	5.75	1	14.5	59.9	1350
S96DC-1-285	3773116001	285.00	70.63	355.63	2.00	10.88	2.00	2.00	6.50	1	5.75	1	10.3	41.8	1045



## Stock Single-Acting Telescopic Cylinders With MB and MC Mounts







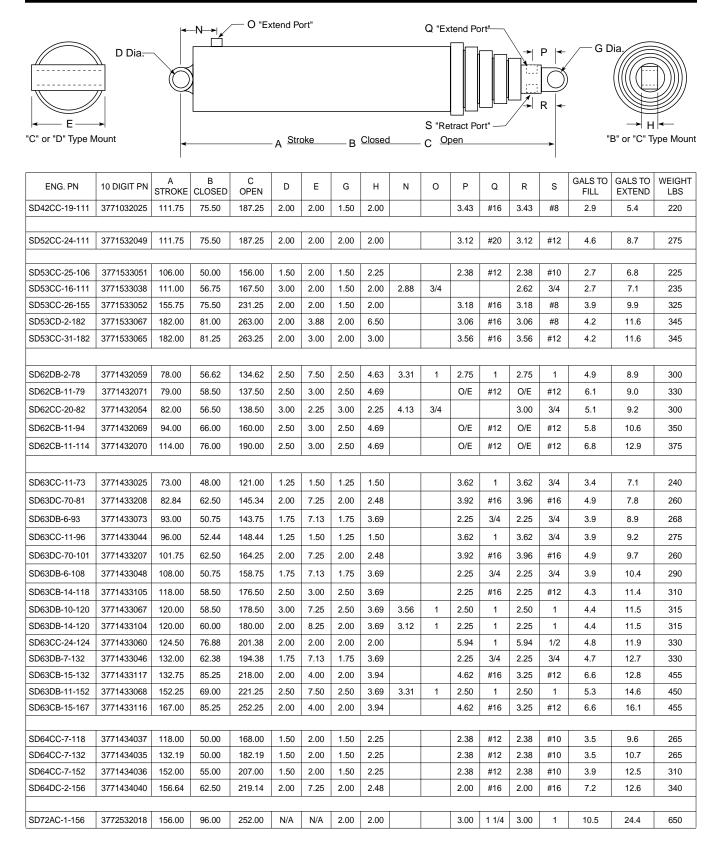
ENG. PN	10 DIGIT PN	STROKE	А	В	С	D	Е	F	G	Н	I	J	WEIGHT LBS	GALS TO EXTEND
S43MM-1G-80	3771013007	80	5.50	38.13	4.75	2.00	1.88	6.75	1.25	1.88	5.75	#10	145	4.6
S43MM-1G-90	3771013008	90	5.50	45.38	4.75	2.00	1.88	6.75	1.25	1.88	5.75	#10	150	5.3
S43MM-1G-100	3771013009	100	5.50	45.38	4.75	2.00	1.88	6.75	1.25	1.88	5.75	#10	150	5.7
S52MM-1G-82	3771512062	82	5.69	53.88	5.75	2.00	1.88	8.00	1.25	1.88	6.75	#12	220	6.7
S52MM-1G-90	3771512063	90	5.50	55.88	5.75	2.00	1.88	8.00	1.25	1.88	6.75	#12	220	7.3
S53MM-1G-90	3771513134	90	5.25	53.38	5.75	2.00	1.88	8.00	1.25	1.88	6.75	#12	320	6.6
S53MM-1G-100	3771513131	100	5.50	55.06	5.75	2.00	1.88	8.00	1.25	1.88	6.75	#12	320	7.7
S53MM-1G-110	3771513132	110	5.50	55.06	5.75	2.00	1.88	8.00	1.25	1.88	6.75	#12	320	8.0
S53MM-1G-120	3771513133	120	5.50	55.06	5.75	2.00	1.88	8.00	1.25	1.88	6.75	#12	320	8.2
S63MM-1G-110	3771413209	110	5.88	57.75	7.00	2.00	1.88	9.00	1.25	1.88	8.00	#12	450	12.4
S63MM-1G-120	3771413210	120	6.38	57.25	7.00	2.00	1.88	9.00	1.25	1.88	8.00	#12	450	12.8
S63MM-1G-130	3771413211	130	6.38	57.25	7.00	2.00	1.88	9.00	1.25	1.88	8.00	#12	450	13.3
S63MM-1G-140	3771413212	140	6.38	60.25	7.00	2.00	1.88	9.00	1.25	1.88	8.00	#12	460	14.1
S73MM-1G-130	3772513110	130	7.63	62.88	8.00	2.00	2.25	10.19	1.50	2.25	9.17	#16	575	19.1
S73MM-1G-140	3772513111	140	7.63	62.88	8.00	2.00	2.25	10.19	1.50	2.25	9.17	#16	575	19.9
S74MM-2G-150	3772514086	150	7.63	62.88	8.00	2.00	2.25	10.19	1.50	2.25	9.17	#16	660	19.9
S74MM-2G-160	3772514085	160	7.63	62.88	8.00	2.00	2.25	10.19	1.50	2.25	9.17	#16	660	20.3
S74MM-2G-170	3772514087	170	7.63	62.88	8.00	2.00	2.25	10.19	1.50	2.25	9.17	#16	660	20.9
S74MM-2G-180	3772514091	180	7.63	62.88	8.00	2.00	2.25	10.19	1.50	2.25	9.17	#16	660	21.4
S74MM-4G-172	3772514088	172	9.63	65.13	9.00	2.00	2.25	11.19	1.25	2.25	9.17	#16	680	22.8
S74MM-4G-182	3772514089	182	9.63	65.13	9.00	2.00	2.25	11.19	1.25	2.25	9.17	#16	680	23.3
S74MM-4G-195	3772514090	195	9.63	65.13	9.00	2.00	2.25	11.19	1.25	2.25	9.17	#16	680	23.9

### **OUR DESIGN ADVANTAGES INCLUDE:**

- \* Longer sleeve overlap for improved stability.
- \* Threaded steel stop rings for easier servicing and more reliable stopping action.
- \* Replaceable, hardened bushings on base trunnion mounts.
- \* Floating trunnion on a true spherical bearing which guarantees self-alignment while maintaining full contact with surface when extending.
- \* Wave springs and chevron packing for self-compensating seals.
- \* Hytrel rod wipers that resist higher temperatures without extrusion.
- \* Snap-on, glass-filled bearings that absorb contaminants without damaging cylinder walls.
- \* Internal packing nuts give added support to the tube exterior while making service procedures easier.

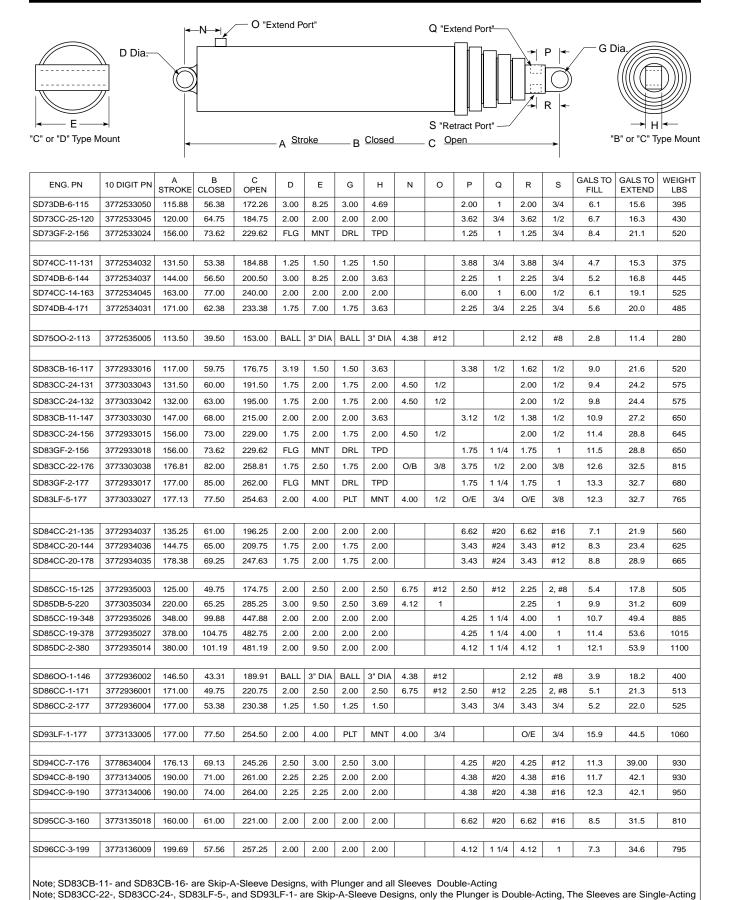


## Stock Double-Acting Telescopic Cylinders Non-Trunnion Mount Designs



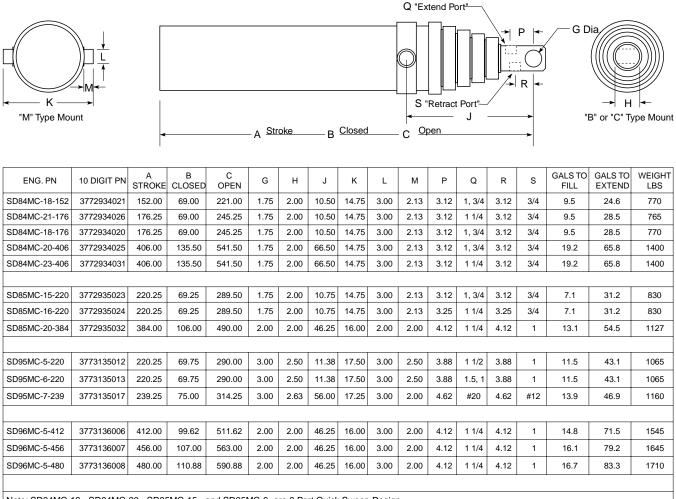


## Stock Double-Acting Telescopic Cylinders Non-Trunnion Mount Designs



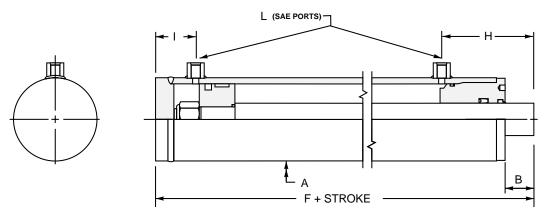


# Stock Double-Acting Telescopic Cylinders Trunnion Mount Designs



Note; SD84MC-18-, SD84MC-20-, SD85MC-15-, and SD95MC-6- are 3 Port Quick Sweep Design.





# 2500 PSI STANDARD DUTY 100 SERIES CYLINDER FEATURES

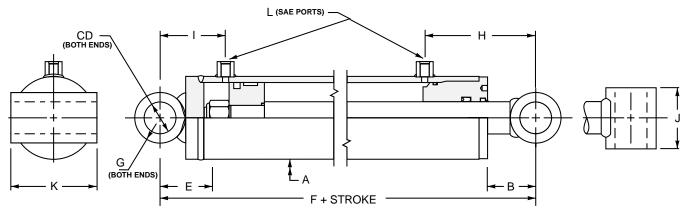
- \*COLD DRAWN (HIGH IMPACT) 75,000 MIN. YIELD D.O.M. TUBING
- \*GROUND & POLISHED, HARD CHROME PLATED RODS (75,000 min. yeild)
- \*WELDED STYLE CONSTRUCTION CERTIFIED TO A.W.S. B2.1
- \*INTERNALLY THREADED HEAD DESIGN WITH BUTTRESS THREADS

- \*HIGHEST QUALITY SEAL CONFIGURATIONS COMPATIBLE WITH PETROLEUM BASE FLUIDS
- \*DUCTILE IRON HEAD GLAND & PISTON
- \*PISTON UTILIZES WEAR BEARINGS
- \*NYLON INSERTED LOCK NUT
- \*STANDARD PAINT; GREY PRIMER

Bore	Rod	HREADS A	В	F	н	ı	L	Maximum Stroke	Part#
	.75	2.00	1.38	5.75	3.31	1.31	#4	18	104-**.**
1.50	1.00	2.00	1.50	6.00	3.56	1.31	#4	34	106-**.**
	1.00	2.50	1.38	6.25	3.62	1.38	#6	25	110-**.**
2.00	1.12	2.50	1.50	6.25	3.62	1.38	#6	31	112-**.**
	1.25	2.50	1.50	6.50	3.88	1.38	#6	39	114-**.**
0.50	1.25	3.00	1.50	6.50	3.62	1.62	#6	31	118-**.**
2.50	1.50	3.00	1.56	7.00	4.06	1.69	#6	45	120-**.**
	1.25	3.50	1.56	7.00	4.00	1.75	#8	26	124-**.**
2.00	1.50	3.50	1.44	7.00	3.88	1.88	#8	38	126-**.**
3.00	1.75	3.50	1.44	7.00	3.88	1.88	#8	52	128-**.**
	2.00	3.50	1.44	7.25	4.12	1.88	#8	66	130-**.**
	1.50	4.00	1.56	7.25	4.00	2.00	#8	32	134-**.**
3.50	1.75	4.00	1.56	7.25	4.00	2.00	#8	44	136-**.**
	2.00	4.00	1.56	7.25	4.00	2.00	#8	58	138-**.**
	1.50	4.50	1.44	7.25	3.88	2.12	#8	28	142-**.**
4.00	1.75	4.50	1.50	7.50	3.94	2.31	#8	39	144-**.**
4.00	2.00	4.50	1.50	7.50	3.94	2.31	#8	51	146-**.**
	2.50	4.50	1.50	7.75	4.19	2.31	#8	78	148-**.**
	1.75	5.00	1.38	7.75	3.81	2.44	#8	34	152-**.**
4.50	2.00	5.00	1.38	7.75	3.81	2.44	#8	45	154-**.**
	2.25	5.00	1.38	7.75	3.81	2.44	#8	58	156-**.**
	2.00	5.62	1.50	8.25	3.94	2.81	#8	40	160-**.**
5.00	2.50	5.62	1.50	8.50	4.19	2.81	#8	62	162-**.**
	3.00	5.62	1.50	8.50	4.19	2.81	#8	89	164-**.**

- \* For stroke lengths longer than 60", add 1" for every 10" of stroke to the "F" dimension.
- \* Other port sizes and locations available upon request.
- \* Consult factory for longer stroke and or higher pressure requirements.
- \* Maximum stroke based on full load at full extension.





# 2500 PSI STANDARD DUTY 200 SERIES CYLINDER FEATURES

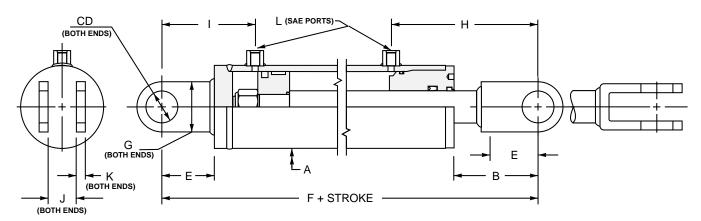
- \*COLD DRAWN (HIGH IMPACT) 75,000 MIN. YIELD D.O.M. TUBING
- \*GROUND & POLISHED, HARD CHROME PLATED RODS (75,000 min. yeild)
- \*WELDED STYLE CONSTRUCTION CERTIFIED TO A.W.S. B2.1
- \*INTERNALLY THREADED HEAD DESIGN WITH BUTTRESS THREADS

- \*HIGHEST QUALITY SEAL CONFIGURATIONS COMPATIBLE WITH PETROLEUM BASE FLUIDS
- \*DUCTILE IRON HEAD GLAND & PISTON
- \*PISTON UTILIZES WEAR BEARINGS
- \*NYLON INSERTED LOCK NUT
- \*STANDARD PAINT; GREY PRIMER

Bore	Rod	Α	В	CD	E	F	G	Н	I	J	K	L	Maximum Stroke	Part#
1.50	.75	2.00	1.31	.75	.56	6.25	.62	3.25	1.88	2.50	2.50	#4	18	204-**.**
1.50	1.00	2.00	1.19	.75	.56	6.25	.62	3.25	1.88	2.50	2.50	#4	34	206-**.**
	1.00	2.50	1.44	1.00	.69	7.00	.75	3.69	2.06	2.50	3.00	#6	25	210-**.**
2.00	1.12	2.50	1.56	1.00	.69	7.00	.75	3.69	2.06	2.50	3.00	#6	31	212-**.**
	1.25	2.50	1.31	1.00	.69	7.00	.75	3.69	2.06	2.50	3.00	#6	39	214-**.**
2.50	1.25	3.00	1.69	1.00	.81	7.50	.88	3.81	2.44	2.50	3.25	#6	31	218-**.**
2.50	1.50	3.00	1.50	1.00	.81	7.75	.88	4.00	2.50	2.50	3.25	#6	45	220-**.**
	1.25	3.50	1.50	1.00	.81	7.75	.88	3.94	2.56	2.50	3.75	#8	26	224-**.**
2.00	1.50	3.50	1.38	1.00	.81	7.75	.88	3.81	2.69	2.50	3.75	#8	38	226-**.**
3.00	1.75	3.50	1.38	1.00	.81	7.75	.88	3.81	2.69	2.50	3.75	#8	52	228-**.**
	2.00	3.50	1.38	1.00	.81	8.00	.88	4.06	2.69	2.50	3.75	#8	66	230-**.**
	1.50	4.00	1.44	1.25	.88	8.00	1.00	3.88	2.88	2.75	4.25	#8	32	234-**.**
3.50	1.75	4.00	1.44	1.25	.88	8.00	1.00	3.88	2.88	2.75	4.25	#8	44	236-**.**
	2.00	4.00	1.44	1.25	.88	8.00	1.00	3.88	2.88	2.75	4.25	#8	58	238-**.**
	1.50	4.50	1.56	1.25	.88	8.25	1.00	4.00	3.00	2.75	4.75	#8	28	242-**.**
	1.75	4.50	1.62	1.25	.88	8.50	1.00	4.06	3.19	2.75	4.75	#8	39	244-**.**
4.00	2.00	4.50	1.62	1.25	.88	8.50	1.00	4.06	3.19	2.75	4.75	#8	51	246-**.**
	2.50	4.50	1.62	1.25	.88	8.75	1.00	4.31	3.19	2.75	4.75	#8	78	248-**.**
	1.75	5.00	1.50	1.25	.88	8.75	1.00	3.94	3.31	2.75	5.25	#8	34	252-**.**
4.50	2.00	5.00	1.50	1.25	.88	8.75	1.00	3.94	3.31	2.75	5.25	#8	45	254-**.**
	2.25	5.00	1.50	1.25	.88	8.75	1.00	3.94	3.31	2.75	5.25	#8	58	256-**.**
	2.00	5.62	1.88	1.50	1.12	9.75	1.25	4.31	3.94	2.75	6.00	#8	40	260-**.**
5.00	2.50	5.62	1.88	1.50	1.12	10.00	1.25	4.56	3.94	2.75	6.00	#8	62	262-**.**
	3.00	5.62	1.88	1.50	1.12	10.00	1.25	4.56	3.94	4.25	6.00	#8	89	264-**.**

- \* For stroke lengths longer than 60", add 1" for every 10" of stroke to the "F" dimension.
- \* Other port sizes and locations available upon request.
- \* Consult factory for longer stroke and or higher pressure requirements.
- \* Maximum stroke based on full load at full extension.





### 2500 PSI STANDARD DUTY 300 SERIES CYLINDER **FEATURES**

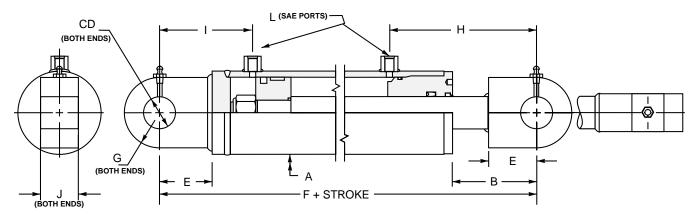
- \*COLD DRAWN (HIGH IMPACT) 75,000 MIN. YIELD D.O.M. TUBING
- \*GROUND & POLISHED, HARD CHROME PLATED RODS (75,000 min. yeild)
- \*WELDED STYLE CONSTRUCTION CERTIFIED TO A.W.S. B2.1
- \*INTERNALLY THREADED HEAD DESIGN WITH

- \*HIGHEST QUALITY SEAL CONFIGURATIONS COMPATIBLE WITH PETROLEUM BASE FLUIDS
- \*DUCTILE IRON HEAD GLAND & PISTON
- \*PISTON UTILIZES WEAR BEARINGS
- \*NYLON INSERTED LOCK NUT
- \*STANDARD PAINT; GREY PRIMER

BOI	TRESS	THRE	ADS											
Bore	Rod	Α	В	CD	E	F	G	Н	ı	J	K	L	Maximum Stroke	Part#
4.50	.75	2.00	3.00	.75	1.62	9.00	1.75	4.94	2.94	1.06	.38	#4	18	304-**.**
1.50	1.00	2.00	2.88	.75	1.62	9.00	1.75	4.94	2.94	1.06	.38	#4	34	306-**.**
	1.00	2.50	3.88	1.00	2.00	10.25	2.00	5.62	3.38	1.25	.50	#6	25	310-**.**
2.00	1.12	2.50	3.50	1.00	2.00	10.25	2.00	5.62	3.38	1.25	.50	#6	31	312-**.**
	1.25	2.50	3.25	1.00	2.00	10.25	2.00	5.62	3.38	1.25	.50	#6	39	314-**.**
2.50	1.25	3.00	3.25	1.00	2.00	10.25	2.00	5.38	3.62	1.25	.50	#6	31	318-**.**
2.50	1.50	3.00	3.06	1.00	2.00	10.25	2.00	5.56	3.44	1.25	.50	#6	45	320-**.**
	1.25	3.50	3.06	1.00	2.00	10.25	2.00	5.50	3.50	1.25	.50	#8	26	324-**.**
2.00	1.50	3.50	3.06	1.00	2.00	10.25	2.00	5.50	3.50	1.25	.50	#8	38	326-**.**
3.00	1.75	3.50	3.06	1.00	2.00	10.25	2.00	5.50	3.50	1.25	.50	#8	52	328-**.**
	2.00	3.50	3.06	1.00	2.00	10.25	2.00	5.75	3.25	1.25	.50	#8	66	330-**.**
	1.50	4.00	3.06	1.00	2.00	10.25	2.00	5.50	3.50	1.25	.50	#8	32	334-**.**
3.50	1.75	4.00	3.06	1.00	2.00	10.25	2.00	5.50	3.50	1.25	.50	#8	44	336-**.**
	2.00	4.00	3.06	1.00	2.00	10.25	2.00	5.50	3.50	1.25	.50	#8	58	338-**.**
	1.50	4.50	3.06	1.00	2.00	10.25	2.00	5.50	3.50	1.25	.50	#8	28	342-**.**
4.00	1.75	4.50	3.06	1.00	2.00	10.25	2.00	5.50	3.50	1.25	.50	#8	39	344-**.**
4.00	2.00	4.50	3.06	1.00	2.00	10.25	2.00	5.50	3.50	1.25	.50	#8	51	346-**.**
	2.50	4.50	3.25	1.00	2.00	11.25	2.00	5.94	4.06	1.25	.50	#8	78	348-**.**
	1.75	5.00	3.38	1.25	2.00	11.75	2.50	5.81	4.44	1.62	.75	#8	34	352-**.**
4.50	2.00	5.00	3.38	1.25	2.00	11.75	2.50	5.81	4.44	1.62	.75	#8	45	354-**.**
	2.25	5.00	3.38	1.25	2.00	11.75	2.50	5.81	4.44	1.62	.75	#8	58	356-**.**
	2.00	5.62	4.25	1.50	2.50	13.50	3.00	6.69	5.31	2.12	1.00	#8	40	360-**.**
5.00	2.50	5.62	4.25	1.50	2.50	13.75	3.00	6.94	5.31	2.12	1.00	#8	62	362-**.**
	3.00	5.62	4.25	1.50	2.50	13.75	3.00	6.94	5.31	2.12	1.00	#8	89	364-**.**

- \* For stroke lengths longer than 60", add 1" for every 10" of stroke to the "F" dimension.
- \* Other port sizes and locations available upon request.
- \* Consult factory for longer stroke and or higher pressure requirements.
- \* Maximum stroke based on full load at full extension.





# 2500 PSI STANDARD DUTY 400 SERIES CYLINDER FEATURES

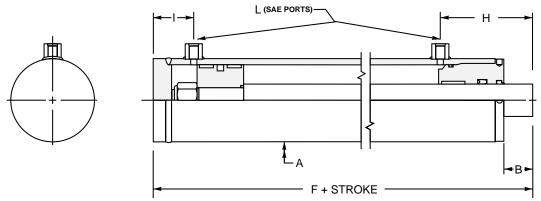
- \*COLD DRAWN (HIGH IMPACT) 75,000 MIN. YIELD D.O.M. TUBING
- \*GROUND & POLISHED, HARD CHROME PLATED RODS (75,000 min. yeild)
- \*WELDED STYLE CONSTRUCTION CERTIFIED TO A.W.S. B2.1
- \*INTERNALLY THREADED HEAD DESIGN WITH BUTTRESS THREADS

- \*HIGHEST QUALITY SEAL CONFIGURATIONS COMPATIBLE WITH PETROLEUM BASE FLUIDS
- \*DUCTILE IRON HEAD GLAND & PISTON
- \*PISTON UTILIZES WEAR BEARINGS
- \*NYLON INSERTED LOCK NUT
- \*STANDARD PAINT; GREY PRIMER

Bore	Rod	A	В	CD	E	F	G	н	1	J	L	Maximum Stroke	Part#
4.50	.75	2.00	2.12	.75	1.50	8.00	.75	4.06	2.81	.75	#4	18	404-**.**
1.50	1.00	2.00	2.00	.75	1.50	8.00	.75	4.06	2.81	1.00	#4	34	406-**.**
	1.00	2.50	2.62	1.00	2.00	9.50	1.00	4.88	3.38	1.00	#6	25	410-**.**
2.00	1.12	2.50	2.75	1.00	2.00	9.50	1.00	4.88	3.38	1.25	#6	31	412-**.**
	1.25	2.50	2.50	1.00	2.00	9.50	1.00	4.88	3.38	1.25	#6	39	414-**.**
2.50	1.25	3.00	2.75	1.00	2.00	9.75	1.00	4.88	3.62	1.25	#6	31	418-**.**
2.50	1.50	3.00	2.56	1.00	2.00	10.00	1.00	5.06	3.69	1.50	#6	45	420-**.**
	1.25	3.50	2.81	1.00	2.00	10.25	1.00	5.25	3.75	1.25	#8	26	424-**.**
	1.50	3.50	2.69	1.00	2.00	10.25	1.00	5.12	3.88	1.50	#8	38	426-**.**
3.00	1.75	3.50	2.69	1.00	2.00	10.25	1.00	5.12	3.88	1.75	#8	52	428-**.**
	2.00	3.50	2.69	1.00	2.00	10.50	1.00	5.38	3.88	2.00	#8	66	430-**.**
	1.50	4.00	3.06	1.25	2.50	11.25	1.25	5.50	4.50	1.50	#8	32	434-**.**
3.50	1.75	4.00	3.06	1.25	2.50	11.25	1.25	5.50	4.50	1.75	#8	44	436-**.**
	2.00	4.00	3.06	1.25	2.50	11.25	1.25	5.50	4.50	2.00	#8	58	438-**.**
	1.50	4.50	3.19	1.25	2.50	11.50	1.25	5.62	4.62	1.50	#8	28	442-**.**
4.00	1.75	4.50	3.25	1.25	2.50	11.75	1.25	5.69	4.81	1.75	#8	39	444-**.**
4.00	2.00	4.50	3.25	1.25	2.50	11.75	1.25	5.69	4.81	2.00	#8	51	446-**.**
	2.50	4.50	3.25	1.25	2.50	12.00	1.25	5.94	4.81	2.50	#8	78	448-**.**
	1.75	5.00	3.12	1.25	2.50	12.00	1.25	5.56	4.94	1.75	#8	34	452-**.**
4.50	2.00	5.00	3.12	1.25	2.50	12.00	1.25	5.56	4.94	2.00	#8	45	454-**.**
	2.25	5.00	3.12	1.25	2.50	12.00	1.25	5.56	4.94	2.50	#8	58	456-**.**
	2.00	5.62	3.25	1.50	2.50	12.50	1.50	5.69	5.31	2.00	#8	40	460-**.**
5.00	2.50	5.62	3.25	1.50	2.50	12.75	1.50	5.94	5.31	2.50	#8	62	462-**.**
	3.00	5.62	3.25	1.50	2.50	12.75	1.50	5.94	5.31	3.00	#8	89	464-**.**

- \* For stroke lengths longer than 60", add 1" for every 10" of stroke to the "F" dimension.
- \* Other port sizes and locations available upon request.
- \* Consult factory for longer stroke and or higher pressure requirements.
- \* Maximum stroke based on full load at full extension.





### 3000 PSI HEAVY DUTY 600 SERIES CYLINDER **FEATURES**

- \*PISTON AND HEAD GLAND HAVE INCREASED BEARING SURFACE AREA
- \*COLD DRAWN (HIGH IMPACT) 75,000 MIN. YIELD D.O.M. TÜBING
- \*GROUND & POLISHED, HARD CHROME PLATED RODS (75,000 min. yeild)
- \*WELDED STYLE CONSTRUCTION CERTIFIED TO A.W.S. B2.1

- \*INTERNALLY THREADED HEAD DESIGN WITH **BUTTRESS THREADS**
- \*HIGHEST QUALITY SEAL CONFIGURATIONS COMPATIBLE WITH PETROLEUM BASE FLUIDS
- \*DUCTILE IRON HEAD GLAND & PISTON
- \*PISTON UTILIZES WEAR BEARINGS
- \*NYLON INSERTED LOCK NUT
- \*STANDARD PAINT; GREY PRIMER

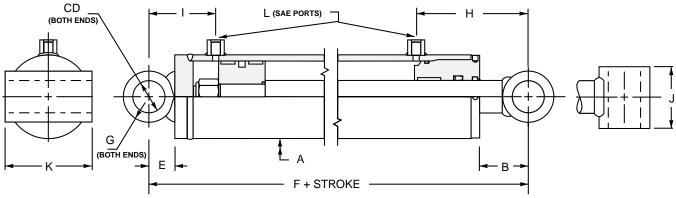
10 /	7. VV. O. D2	<u> 1</u>		1	1		TAINT, GILL	1	
Bore	Rod	Α	В	F	н	I	L	Maximum Stroke	Part#
3.00	1.75	3.50	1.56	8.25	3.94	1.94	#6	47	604-**.**
3.00	2.00	3.50	1.56	8.25	3.94	1.94	#6	53	606-**.**
3.50	1.75	4.00	1.44	8.25	3.88	2.00	#8	40	610-**.**
3.50	2.00	4.00	1.50	8.50	3.94	2.19	#8	52	612-**.**
	2.00	4.62	1.50	9.25	4.56	2.31	#8	46	616-**.**
4.00	2.50	4.62	1.50	9.25	4.56	2.31	#8	72	618-**.**
	3.00	4.62	1.62	10.00	5.44	2.56	#8	99	620-**.**
	2.00	5.12	1.50	9.50	4.62	2.50	#10	40	624-**.**
4.50	2.50	5.12	1.50	9.75	4.62	2.75	#10	64	626-**.**
4.50	3.00	5.12	1.50	9.75	4.62	2.75	#10	90	628-**.**
	3.50	5.12	1.50	10.00	5.38	2.62	#10	115	630-**.**
	2.00	5.62	1.50	9.25	4.44	2.44	#12	36	634-**.**
	2.50	5.62	1.62	10.25	5.00	2.69	#12	57	636-**.**
5.00	3.00	5.62	1.62	10.25	5.00	2.69	#12	82	638-**.**
	3.50	5.62	1.50	10.00	4.88	2.56	#12	108	640-**.**
	4.00	5.62	1.62	10.25	5.56	2.69	#12	130	642-**.**
	2.50	6.12	1.44	10.25	4.81	2.75	#12	52	646-**.**
	3.00	6.12	1.56	10.25	4.94	2.62	#12	75	648-**.**
5.50	3.50	6.12	1.56	10.25	4.94	2.62	#12	100	650-**.**
	4.00	6.12	1.69	10.50	5.06	2.75	#12	124	652-**.**
	4.50	6.12	1.56	11.00	5.50	3.38	#12	144	654-**.**
	2.50	6.75	1.62	10.50	5.00	2.81	#12	48	658-**.**
6.00	3.00	6.75	1.50	10.25	4.88	2.69	#12	70	660-**.**
6.00	3.50	6.75	1.50	10.25	4.88	2.69	#12	94	662-**.**
	4.00	6.75	1.62	10.50	5.00	2.81	#12	120	664-**.**
	2.50	8.00	1.69	11.25	5.62	2.75	#16	40	668-**.**
7.00	3.00	8.00	1.69	11.25	5.62	2.75	#16	60	670-**.**
7.00	3.50	8.00	1.69	11.25	5.62	2.75	#16	82	672-**.**
	4.00	8.00	1.56	11.75	5.50	3.38	#16	107	674-**.**
	3.50	9.00	1.50	13.00	5.69	3.44	#16	71	678-**.**
8.00	4.00	9.00	1.50	13.00	5.69	3.44	#16	94	680-**.**
	4.50	9.00	1.50	13.00	5.69	3.44	#16	118	682-**.**

Ordering Information: \* TO COMPLETE PART#; REPLACE (\*\*.\*\*) WITH STROKE REQUIRED

\* For stroke lengths longer than 60", add 1" for every 10" of stroke to the "F" dimension.

- - \* Other port sizes and locations available upon request.
  - \* Consult factory for longer stroke and or higher pressure requirements.
  - \* Maximum stroke based on full load at full extension.





### 3000 PSI HEAVY DUTY 700 SERIES CYLINDER **FEATURES**

- \*PISTON AND HEAD GLAND HAVE INCREASED BEARING SURFACE AREA
- \*COLD DRAWN (HIGH IMPACT) 75,000 MIN. YIELD D.O.M. TUBING
- \*GROUND & POLISHED, HARD CHROME PLATED RODS (75,000 min. yeild)
- \*WELDED STYLE CONSTRUCTION CERTIFIED
- \*INTERNALLY THREADED HEAD DESIGN WITH **BUTTRESS THREADS**
- \*HIGHEST QUALITY SEAL CONFIGURATIONS COMPATIBLE WITH PETROLEUM BASE FLUIDS
- \*DUCTILE IRON HEAD GLAND & PISTON
- \*PISTON UTILIZES WEAR BEARINGS
- \*NYLON INSERTED LOCK NUT

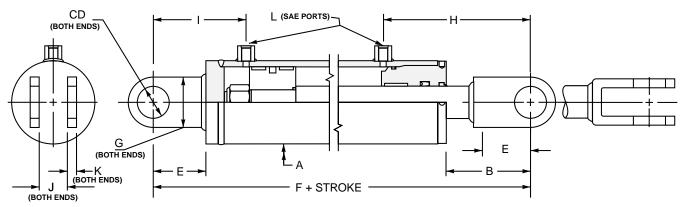
3.00 2 3.50 2	Rod 1.75 2.00 1.75	A 3.50 3.50	B 1.50	CD	E	F	G		_			-	Maximum	
3.00 2 3.50 1 2	2.00 1.75		1.50			-	G	н		J	K	L	Stroke	Part#
3.50	1.75	3.50		1.00	.81	9.00	.88	3.88	2.75	2.75	3.75	#6	47	704-**.**
3.50			1.50	1.00	.81	9.00	.88	3.88	2.75	2.75	3.75	#6	53	706-**.**
2	2 22	4.00	1.56	1.25	.88	9.25	1.00	4.00	2.88	3.25	4.25	#8	40	710-**.**
	2.00	4.00	1.62	1.25	.88	9.50	1.00	4.06	3.06	3.25	4.25	#8	52	712-**.**
2	2.00	4.62	2.12	1.50	1.12	11.00	1.25	5.19	3.44	3.00	4.75	#8	46	716-**.**
4.00 2	2.50	4.62	2.12	1.50	1.12	11.00	1.25	5.19	3.44	3.00	4.75	#8	72	718-**.**
3	3.00	4.62	2.00	1.50	1.12	11.50	1.25	5.81	3.69	4.25	4.75	#8	99	720-**.**
2	2.00	5.12	1.88	1.50	1.12	11.00	1.25	5.00	3.62	3.00	5.50	#10	40	724-**.**
4.50	2.50	5.12	2.12	1.50	1.12	11.50	1.25	5.25	3.88	3.00	5.50	#10	64	726-**.**
4.50	3.00	5.12	2.12	1.50	1.12	11.50	1.25	5.25	3.88	4.25	5.50	#10	90	728-**.**
3	3.50	5.12	2.12	1.50	1.12	11.75	1.25	6.00	3.75	4.25	5.50	#10	115	730-**.**
2	2.00	5.62	2.25	1.75	1.25	11.25	1.38	5.19	3.69	3.25	6.00	#12	36	734-**.**
2	2.50	5.62	2.12	1.75	1.25	12.00	1.38	5.50	3.94	3.25	6.00	#12	57	736-**.**
5.00	3.00	5.62	2.12	1.75	1.25	12.00	1.38	5.50	3.94	4.75	6.00	#12	82	738-**.**
3	3.50	5.62	2.00	1.75	1.25	11.75	1.38	5.38	3.81	4.75	6.00	#12	108	740-**.**
4	4.00	5.62	2.12	1.75	1.25	12.00	1.38	6.06	3.94	4.75	6.00	#12	130	742-**.**
2	2.50	6.12	2.19	1.75	1.25	12.25	1.38	5.56	4.00	3.25	7.00	#12	52	746-**.**
3	3.00	6.12	2.06	1.75	1.25	12.00	1.38	5.44	3.88	4.75	7.00	#12	75	748-**.**
5.50 3	3.50	6.12	2.06	1.75	1.25	12.00	1.38	5.44	3.88	4.75	7.00	#12	100	750-**.**
4	4.00	6.12	2.19	1.75	1.25	12.25	1.38	5.56	4.00	4.75	7.00	#12	124	752-**.**
4	4.50	6.12	2.06	1.75	1.25	12.75	1.38	6.00	4.62	5.50	7.00	#12	144	754-**.**
2	2.50	6.75	2.25	2.00	1.38	12.50	1.50	5.62	4.19	3.50	7.00	#12	48	758-**.**
6.00	3.00	6.75	2.12	2.00	1.38	12.25	1.50	5.50	4.06	3.50	7.00	#12	70	760-**.**
8.00	3.50	6.75	2.12	2.00	1.38	12.25	1.50	5.50	4.06	4.75	7.00	#12	94	762-**.**
4	4.00	6.75	2.25	2.00	1.38	12.50	1.50	5.62	4.19	5.50	7.00	#12	120	764-**.**
2	2.50	8.00	2.81	2.50	1.62	14.00	1.75	6.75	4.38	5.00	8.25	#16	40	768-**.**
7.00	3.00	8.00	2.81	2.50	1.62	14.00	1.75	6.75	4.38	5.00	8.25	#16	60	770-**.**
7.00	3.50	8.00	2.81	2.50	1.62	14.00	1.75	6.75	4.38	5.00	8.25	#16	82	772-**.**
4	4.00	8.00	2.69	2.50	1.62	14.50	1.75	6.62	5.00	5.00	8.25	#16	107	774-**.**
3	3.50	9.00	2.88	3.00	1.88	16.25	2.00	7.06	5.31	5.00	9.25	#16	71	778-**.**
8.00 4	4.00	9.00	2.88	3.00	1.88	16.25	2.00	7.06	5.31	5.00	9.25	#16	94	780-**.**
4	4.50	9.00	2.88	3.00	1.88	16.25	2.00	7.06	5.31	5.00	9.25	#16	118	782-**.**

Ordering Information: \* TO COMPLETE PART#; REPLACE (\*\*.\*\*) WITH STROKE REQUIRED

\* For stroke lengths longer than 60", add 1" for every 10" of stroke to the "F" dimension.

- \* Other port sizes and locations available upon request.
- \* Consult factory for longer stroke and or higher pressure requirements.
- \* Maximum stroke based on full load at full extension.





# 3000 PSI HEAVY DUTY 800 SERIES CYLINDER FEATURES

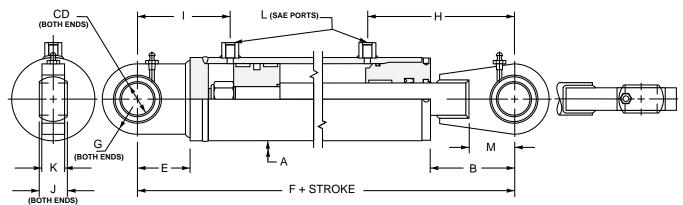
- \*PISTON AND HEAD GLAND HAVE INCREASED BEARING SURFACE AREA
- \*COLD DRAWN (HIGH IMPACT) 75,000 MIN. YIELD D.O.M. TUBING
- \*GROUND & POLISHED, HARD CHROME PLATED RODS (75,000 min. yeild)
- \*WELDED STYLÈ CONSTRUCTION CERTIFIED TO A.W.S. B2.1

- \*INTERNALLY THREADED HEAD DESIGN WITH BUTTRESS THREADS
- \*HIGHEST QUALITY SEAL CONFIGURATIONS COMPATIBLE WITH PETROLEUM BASE FLUIDS
- \*DUCTILE IRON HEAD GLAND & PISTON
- \*PISTON UTILIZES WEAR BEARINGS
- \*NYLON INSERTED LOCK NUT
- \*STANDARD PAINT; GREY PRIMER

Bore	Rod	Α	В	CD	E	F	G	Н	ı	J	к	L	Maximum Stroke	Part#
2.00	1.75	3.50	3.31	1.00	2.00	12.00	2.00	5.75	3.94	1.25	.50	#6	47	804-**.**
3.00	2.00	3.50	3.31	1.00	2.00	12.00	2.00	5.75	3.94	1.25	.50	#6	53	806-**.**
2.50	1.75	4.00	3.81	1.25	2.00	12.50	2.50	4.50	4.50	1.62	.75	#8	40	810-**.**
3.50	2.00	4.00	3.50	1.25	2.00	12.50	2.50	5.94	4.69	1.62	.75	#8	52	812-**.**
	2.00	4.62	4.25	1.50	2.00	14.00	3.00	7.31	4.31	1.62	1.00	#8	46	816-**.**
4.00	2.50	4.62	4.25	1.50	2.00	14.00	3.00	7.31	4.31	1.62	1.00	#8	72	818-**.**
	3.00	4.62	4.12	1.50	2.00	14.50	3.00	7.94	4.56	1.62	1.00	#8	99	820-**.**
	2.00	5.12	4.25	1.50	2.00	14.25	3.00	7.38	4.50	1.62	1.00	#10	40	824-**.**
4.50	2.50	5.12	4.00	1.50	2.00	14.25	3.00	7.12	4.75	1.62	1.00	#10	64	826-**.**
4.50	3.00	5.12	4.00	1.50	2.00	14.25	3.00	7.12	4.75	1.62	1.00	#10	90	828-**.**
	3.50	5.12	4.00	1.50	2.00	14.50	3.00	7.88	4.62	1.62	1.00	#10	115	830-**.**
	2.00	5.62	4.25	1.75	2.25	14.25	3.50	7.19	4.69	2.12	1.00	#12	36	834-**.**
	2.50	5.62	4.12	1.75	2.25	15.00	3.50	7.50	4.94	2.12	1.00	#12	57	836-**.**
5.00	3.00	5.62	4.12	1.75	2.25	15.00	3.50	7.50	4.94	2.12	1.00	#12	82	838-**.**
	3.50	5.62	4.25	1.75	2.25	15.00	3.50	7.62	4.81	2.12	1.00	#12	108	840-**.**
	4.00	5.62	4.12	1.75	2.25	15.00	3.50	8.06	4.94	2.12	1.00	#12	130	842-**.**
	2.50	6.12	4.44	1.75	2.25	15.50	3.50	7.81	5.00	2.12	1.25	#12	52	846-**.**
	3.00	6.12	4.56	1.75	2.25	15.50	3.50	7.94	4.88	2.12	1.25	#12	75	848-**.**
5.50	3.50	6.12	4.56	1.75	2.25	15.50	3.50	7.94	4.88	2.12	1.25	#12	100	850-**.**
	4.00	6.12	4.44	1.75	2.25	15.50	3.50	7.81	5.00	2.12	1.25	#12	124	852-**.**
	4.50	6.12	4.56	1.75	2.25	16.25	3.50	8.50	5.63	2.12	1.25	#12	144	854-**.**
	2.50	6.75	4.62	2.00	2.50	16.00	4.00	8.00	5.31	2.62	1.25	#12	48	858-**.**
6.00	3.00	6.75	4.50	2.00	2.50	15.75	4.00	7.88	5.19	2.62	1.25	#12	70	860-**.**
6.00	3.50	6.75	4.50	2.00	2.50	15.75	4.00	7.88	5.19	2.62	1.25	#12	94	862-**.**
	4.00	6.75	4.62	2.00	2.50	16.00	4.00	8.00	5.31	2.62	1.25	#12	120	864-**.**
	2.50	8.00	5.19	2.50	3.00	17.75	5.00	9.12	5.75	2.62	1.25	#16	40	868-**.**
7.00	3.00	8.00	5.19	2.50	3.00	17.75	5.00	9.12	5.75	2.62	1.25	#16	60	870-**.**
7.00	3.50	8.00	5.19	2.50	3.00	17.75	5.00	9.12	5.75	2.62	1.25	#16	82	872-**.**
	4.00	8.00	5.06	2.50	3.00	18.25	5.00	9.00	6.38	2.62	1.25	#16	107	874-**.**
	3.50	9.00	6.50	3.00	4.00	22.00	6.00	10.69	7.44	3.12	1.50	#16	71	878-**.**
8.00	4.00	9.00	6.50	3.00	4.00	22.00	6.00	10.69	7.44	3.12	1.50	#16	94	880-**.**
	4.50	9.00	6.50	3.00	4.00	22.00	6.00	10.69	7.44	3.12	1.50	#16	118	882-**.**

- \* For stroke lengths longer than 60", add 1" for every 10" of stroke to the "F" dimension.
- \* Other port sizes and locations available upon request.
- \* Consult factory for longer stroke and or higher pressure requirements.
- \* Maximum stroke based on full load at full extension.





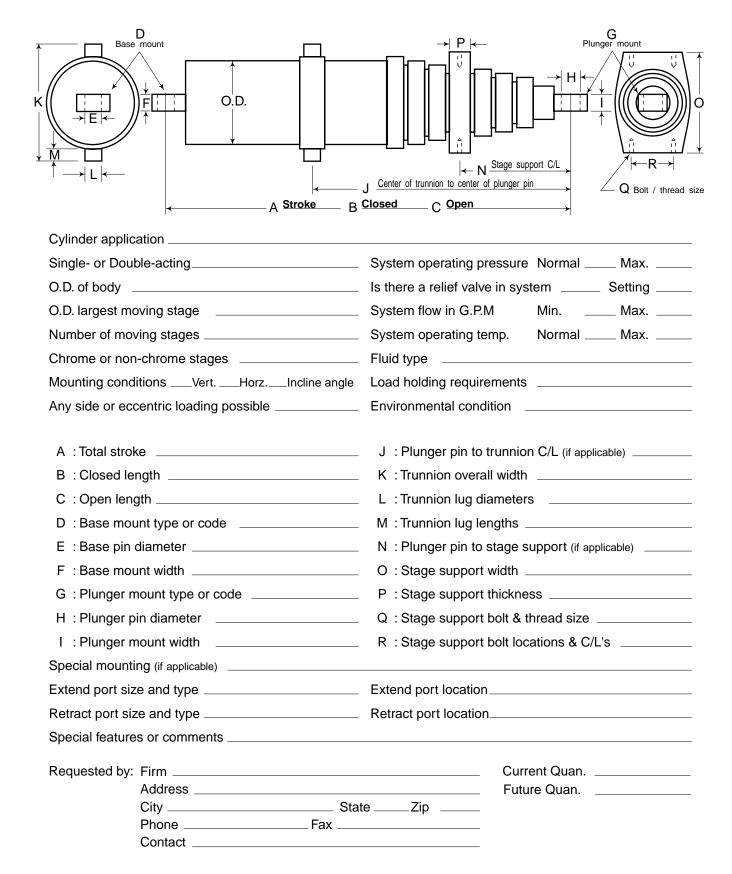
# 3000 PSI HEAVY DUTY 900 SERIES CYLINDER FEATURES

- \*PISTON AND HEAD GLAND HAVE INCREASED BEARING SURFACE AREA
- \*COLD DRAWN (HIGH IMPACT) 75,000 MIN. YIELD D.O.M. TUBING
- \*GROUND & POLISHED, HARD CHROME PLATED RODS (75,000 min. yeild)
- \*WELDED STYLE CONSTRUCTION CERTIFIED TO A.W.S. B2.1
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- \*HIGHEST QUALITY SEAL CONFIGURATIONS COMPATIBLE WITH PETROLEUM BASE FLUIDS
- \*DUCTILE IRON HEAD GLAND & PISTON
- \*PISTON UTILIZES WEAR BEARINGS
- \*NYLON INSERTED LOCK NUT
- \*STANDARD PAINT; GREY PRIMER

			_											Maximum	
Bore	Rod	Α	В	CD	E	F	G	Н	ı	J	K	L	М	Stroke	Part#
3.00	1.75	3.50	3.81	1.25	2.00	12.50	1.50	6.19	3.94	1.093	.94	#6	2.00	47	904-**.**
3.00	2.00	3.50	3.81	1.25	2.00	12.50	1.50	6.19	3.94	1.093	.94	#6	2.00	53	906-**.**
3.50	1.75	4.00	3.94	1.25	2.00	12.75	1.50	6.38	4.00	1.093	.94	#8	2.00	40	910-**.**
3.30	2.00	4.00	3.75	1.25	2.00	12.75	1.50	6.19	4.19	1.093	.94	#8	2.00	52	912-**.**
	2.00	4.62	4.00	1.50	2.50	14.25	1.88	7.06	4.81	1.312	1.12	#8	2.25	46	916-**.**
4.00	2.50	4.62	4.00	1.50	2.50	14.25	1.88	7.06	4.81	1.312	1.12	#8	2.25	72	918-**.**
	3.00	4.62	4.12	1.50	2.50	15.00	1.88	7.94	5.06	1.312	1.12	#8	2.25	99	920-**.**
	2.00	5.12	4.00	1.50	2.50	14.50	1.88	7.12	5.00	1.312	1.12	#10	2.25	40	924-**.**
4.50	2.50	5.12	4.00	1.50	2.50	14.75	1.88	7.12	5.25	1.312	1.12	#10	2.25	64	926-**.**
4.50	3.00	5.12	4.00	1.50	2.50	14.75	1.88	7.12	5.25	1.312	1.12	#10	2.25	90	928-**.**
	3.50	5.12	4.00	1.50	2.50	15.00	1.88	7.88	5.12	1.312	1.12	#10	2.25	115	930-**.**
	2.00	5.62	4.50	1.75	2.50	14.75	2.00	7.44	4.94	1.531	1.31	#12	2.50	36	934-**.**
	2.50	5.62	4.62	1.75	2.50	15.75	2.00	8.00	5.19	1.531	1.31	#12	2.50	57	936-**.**
5.00	3.00	5.62	4.62	1.75	2.50	15.75	2.00	8.00	5.19	1.531	1.31	#12	2.50	82	938-**.**
	3.50	5.62	4.50	1.75	2.50	15.50	2.00	7.88	5.06	1.531	1.31	#12	2.50	108	940-**.**
	4.00	5.62	4.62	1.75	2.50	15.75	2.00	8.56	5.19	1.531	1.31	#12	2.50	130	942-**.**
	2.50	6.12	4.69	1.75	2.50	16.00	2.00	8.06	5.25	1.531	1.31	#12	2.50	52	946-**.**
	3.00	6.12	4.56	1.75	2.50	15.75	2.00	7.94	5.12	1.531	1.31	#12	2.50	75	948-**.**
5.50	3.50	6.12	4.56	1.75	2.50	15.75	2.00	7.94	5.12	1.531	1.31	#12	2.50	100	950-**.**
	4.00	6.12	4.69	1.75	2.50	16.00	2.00	8.06	5.25	1.531	1.31	#12	2.50	124	952-**.**
	4.50	6.12	4.56	1.75	2.50	16.50	2.00	8.50	5.88	1.531	1.31	#12	2.50	144	954-**.**
	2.50	6.75	4.88	2.00	2.75	16.50	2.38	8.25	5.56	1.750	1.50	#12	2.75	48	958-**.**
	3.00	6.75	4.75	2.00	2.75	16.25	2.38	8.12	5.44	1.750	1.50	#12	2.75	70	960-**.**
6.00	3.50	6.75	4.75	2.00	2.75	16.25	2.38	8.12	5.44	1.750	1.50	#12	2.75	94	962-**.**
	4.00	6.75	4.88	2.00	2.75	16.50	2.38	8.25	5.56	1.750	1.50	#12	2.75	120	964-**.**
	2.50	8.00	5.69	2.50	3.25	18.50	3.00	9.62	6.00	2.188	1.88	#16	3.25	40	968-**.**
7.00	3.00	8.00	5.69	2.50	3.25	18.50	3.00	9.62	6.00	2.188	1.88	#16	3.25	60	970-**.**
7.00	3.50	8.00	5.69	2.50	3.25	18.50	3.00	9.62	6.00	2.188	1.88	#16	3.25	82	972-**.**
	4.00	8.00	5.56	2.50	3.25	19.00	3.00	9.50	6.62	2.188	1.88	#16	3.25	107	974-**.**
	3.50	9.00	6.50	3.00	4.25	22.25	3.75	10.69	7.69	2.625	2.25	#16	4.25	70	978-**.**
8.00	4.00	9.00	6.50	3.00	4.25	22.25	3.75	10.69	7.69	2.625	2.25	#16	4.25	94	980-**.**
	4.50	9.00	6.50	3.00	4.25	22.25	3.75	10.69	7.69	2.625	2.25	#16	4.25	118	982-**.**
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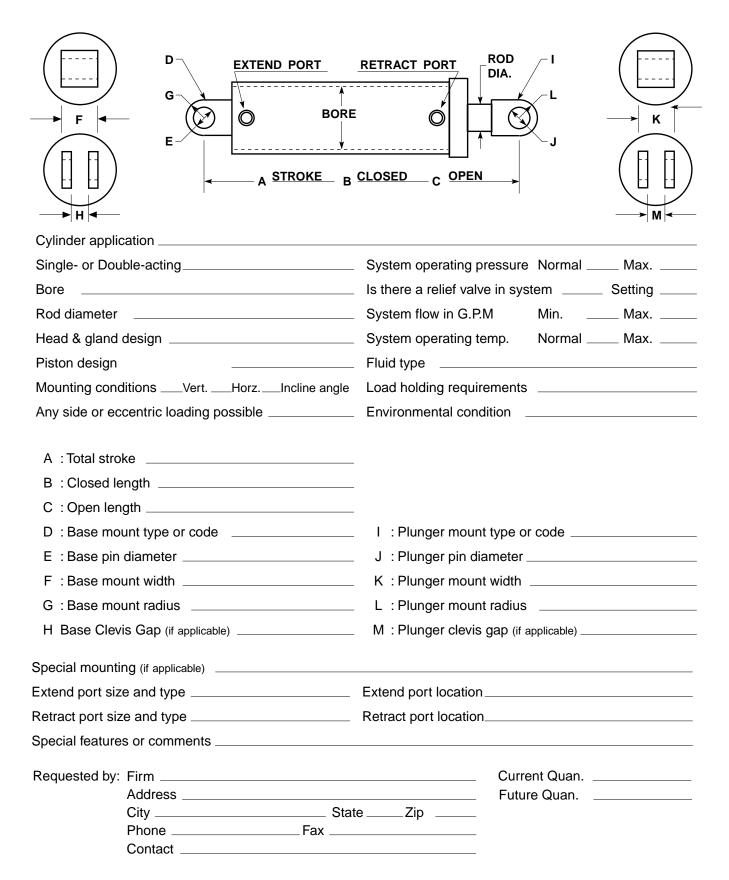
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Phone: (800) 848-5575 \* 330-480-8431 \* Fax (800) 694-3392 \* 330-480-8432





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- 2. Payment: Payment shall be made by Buyer net 30 days from the date of delivery of the items purchased hereunder. Any claims by Buyer for omissions or shortages in a shipment shall be waived unless Seller receives notice thereof within 30 days after Buyer's receipt of the shipment.
- 3. Delivery: Unless otherwise provided on the face hereof, delivery shall be made F.O.B. Seller's plant. Regardless of the method of delivery, however, risk of loss shall pass to Buyer upon Seller's delivery to a carrier. Any delivery dates shown are approximate only and Seller shall have no liability for any delays in delivery.
- 4. Warranty: Seller warrants that the items sold hereunder shall be free from defects in material or workmanship for a period of 18 months from the date of shipment from Parker Hannifin Corporation. THIS WARRANTY COMPRISES THE SOLE AND ENTIRE WARRANTY PERTAINING TO ITEMS PROVIDED HEREUNDER. SELLER MAKES NO OTHER WARRANTY, GUARANTEE, OR REPRESENTATION OF ANY KIND WHATSOEVER. ALL OTHER WARRANTIES, INCLUDING BUT NOT LIMITED TO, MERCHANTIBILITY AND FITNESS FOR PURPOSE, WHETHER EXPRESS, IMPLIED, OR ARISING BY OPERATION OF LAW, TRADE USAGE, OR COURSE OF DEALING ARE HEREBY DISCLAIMED.

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- 5. Limitation Of Remedy: SELLER'S LIABILITY ARISING FROM OR IN ANY WAY CONNECTED WITH THE ITEMS SOLD OR THIS CONTRACT SHALL BE LIMITED EXCLUSIVELY TO REPAIR OR REPLACEMENT OF THE ITEMS SOLD OR REFUND OF THE PURCHASE PRICE PAID BY BUYER, AT SELLER'S SOLE OPTION. IN NO EVENT SHALL SELLER BE LIABLE FOR ANY INCIDENTAL, CONSEQUENTIAL OR SPECIAL DAMAGES OF ANY KIND OR NATURE WHATSOEVER, INCLUDING BUT NOT LIMITED TO LOST PROFITS ARISING FROM OR IN ANY WAY CONNECTED WITH THIS AGREEMENT OR ITEMS SOLD HEREUNDER, WHETHER ALLEGED TO ARISE FROM BREACH OF CONTRACT, EXPRESS OR IMPLIED WARRANTY, OR IN TORT, INCLUDING WITHOUT LIMITATION, NEGLIGENCE. FAILURE TO WARN OR STRICT LIABILITY.
- 6. Changes, Reschedules and Cancellations: Buyer may request to modify the designs or specifications for the items sold hereunder as well as the quantities and delivery dates thereof, or may request to cancel all or part of this order, however, no such requested modification or cancellation shall become part of the contract between Buyer and Seller unless accepted by Seller in a written amendment to this Agreement. Acceptance of any such requested modification or cancellation shall be at Seller's discretion, and shall be upon such terms and conditions as Seller may require.
- 7. Special Tooling: A tooling charge may be imposed for any special tooling, including without limitation, dies, fixtures, molds and patterns, acquired to manufacture items sold pursuant to this contract. Such special tooling shall be and remain Seller's property notwithstanding payment of any charges by Buyer. In no event will Buyer acquire any interest in apparatus belonging to Seller which is utilized in the manufacture of the items sold hereunder, even if such apparatus has been specially converted or adapted for such manufacture and notwithstanding any charges paid by Buyer. Unless otherwise agreed, Seller shall have the right to alter, discard or otherwise dispose of any special tooling or other property in its sole discretion at any time.

- 8. Buyer's Property: Any designs, tools, patterns, materials, drawings, confidential information or equipment furnished by Buyer or any other items which become Buyer's property, may be considered obsolete and may be destroyed by Seller after two (2) consecutive years have elapsed without Buyer placing an order for the items which are manufactured using such property. Seller shall not be responsible for any loss or damage to such property while it is in Seller's possession or control.
- 9. Taxes: Unless otherwise indicated on the face hereof, all prices and charges are exclusive of excise, sales, use, property, occupational or like taxes which may be imposed by any taxing authority upon the manufacture, sale or delivery of the items sold hereunder. If any such taxes must be paid by Seller or if Seller is liable for the collection of such tax, the amount thereof shall be in addition to the amounts for the items sold. Buyer agrees to pay all such taxes or to reimburse Seller therefore upon receipt of its invoice. If Buyer claims exemption from any sales, use or other tax imposed by any taxing authority, Buyer shall save Seller harmless from and against any such tax, together with any interest or penalties thereon which may be assessed if the items are held to be taxable
- 10. Indemnity For Infringement of Intellectual Property Rights: Seller shall have no liability for infringement of any patents, trademarks, copyrights, trade dress, trade secrets or similar rights except as provided in this Part 10. Seller will defend and indemnify Buyer against allegations of infringement of U.S. patents, U.S. trademarks, copyrights, trade dress and trade secrets (hereinafter 'Intellectual Property Rights'). Seller will defend at its expense and will pay the cost of any settlement or damages awarded in an action brought against Buyer based on an allegation that an item sold pursuant to this contract infringes the Intellectual Property Rights of a third party. Seller's obligation to defend and indemnify Buyer is contingent on Buyer notifying Seller within ten (10) days after Buyer becomes aware of such allegations of infringement, and Seller having sole control over the defense of any allegations or actions including all negotiations for settlement or compromise. If an item sold hereunder is subject to a claim that it infringes the Intellectual Property Rights of a third party, Seller may, at its sole expense and option, procure for Buyer the right to continue using said item, replace or modify said item so as to make it noninfringing, or offer to accept return of said item and return the purchase price less a reasonable allowance for depreciation. Notwithstanding the foregoing, Seller shall have no liability for claims of infringement based on information provided by Buyer, or directed to items delivered hereunder for which the designs are specified in whole or part by Buyer, or infringements resulting from the modification, combination or use in a system of any item sold hereunder. The foregoing provisions of this Part 10 shall constitute Seller's sole and exclusive liability and Buyer's sole and exclusive remedy for infringement of Intellectual Property Rights.

If a claim is based on information provided by Buyer or if the design for an item delivered hereunder is specified in whole or in part by Buyer, Buyer shall defend and indemnify Seller for all costs, expenses or judgments resulting from any claim that such item infringes any patent, trademark, copyright, trade dress, trade secret or any similar right.

- 11. Force Majeure: Seller does not assume the risk of and shall not be liable for delay or failure to perform any of Seller's obligations by reason of circumstances beyond the reasonable control of Seller (hereinafter 'events of Force Majeure]. Events of Force Majeure shall include without limitation, accidents, acts of God, strikes or labor disputes, acts, laws, rules or regulations of any government or government agency, fires, floods, delays or failures in delivery of carriers or suppliers, shortages of materials and any other cause beyond Seller's control.
- 12. Entire Agreement/Governing Law: The terms and conditions set forth herein, together with any amendments, modifications and any different terms or conditions expressly accepted by Seller in writing, shall constitute the entire Agreement concerning the items sold, and there are no oral or other representations or agreements which pertain thereto. This Agreement shall be governed in all respects by the law of the State of Ohio. No actions arising out of the sale of the items sold hereunder or this Agreement may be brought by either party more than two (2) years after the cause of action accrues.





### **Parker Hannifin Corporation** 6035 Parkland Blvd. Cleveland, Ohio 44124-4141 Telephone: (216) 896-3000 Fax: (216) 896-4000 www.parker.com

# **Parker Hannifin Corporation**

### **About Parker Hannifin Corporation**

Parker Hannifin is a leading global motion-control company dedicated to delivering premier customer service. A Fortune 500 corporation listed on the New York Stock Exchange (PH), our components and systems comprise over 1,400 product lines that control motion in some 1,000 industrial and aerospace markets. Parker is the only manufacturer to offer its customers a choice of hydraulic, pneumatic, and electromechanical motion-control solutions. Our Company has the largest distribution network in its field, with over 7,500 distributors serving nearly 400,000 customers worldwide.

## Parker's Charter

To be a leading worldwide manufacturer of components and systems for the builders and users of durable goods. More specifically, we will design, market and manufacture products controlling motion, flow and pressure. We will achieve profitable growth through premier customer service.

### **Product Information**

North American customers seeking product information, the location of a nearby distributor, or repair services will receive prompt attention by calling the Parker Product Information Center at our toll-free number: 1-800-C-PARKER (1-800-272-7537). In Europe, call 00800-C-PARKER-H (00800-2727-5374).

### The Aerospace Group

is a leader in the development, design, manufacture and servicing of control systems and components for aerospace and related high-technology markets, while achieving growth through premier customer service.



### The Climate & Industrial **Controls Group**

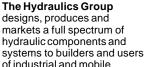
designs, manufactures and markets system-control and fluid-handling components and systems to refrigeration, airconditioning and industrial customers worldwide.



and fluid systems.



The Seal Group designs, manufactures and distributes industrial and commercial sealing devices and related products by providing superior quality and total customer satisfaction.



of industrial and mobile machinery and equipment.



### The Filtration Group

designs, manufactures and markets quality filtration and clarification products, providing customers with the best value. quality, technical support, and global availability.







### The Instrumentation Group is a global leader in the design, manufacture and distribution of high-quality critical flow components for worldwide process instrumentation, ultrahigh-purity, medical and analytical applications.



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